



KONGSBERG

Tensile test report of polylactic acid (PLA)



Kromium



University of
South-Eastern Norway

Faculty of Technology, Natural Sciences and
Maritime Sciences
Campus Kongsberg

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Testers:

Adrian Elias Haugjord
Henrik Bertelsen

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Abbreviations

PLA Polylactic acid. 8, 10

UTS Ultimate tensile strength. 10, 13, 17, 19

Description of Terms

3D printer A machine where a thread of plastic is moved through and heated up by a nozzle that melts and places it layer by layer to create a part. 10

ultimate tensile strength The end of plastic deformation region and the maximum stress a material can take. 8

yield strength The point where a material goes from elastic to plastic deformation. 8

Young's modulus Also known as E-modulus or E-module, is the measurement of the stiffness of a material when a tensile or compressive force is applied lengthwise. 8

Introduction

This test is performed as part of the 2024 bachelor project Kromium[1]. This test aims to extract necessary material data for use in custom material in SolidWorks for the project. The material in question is the Polylactic acid (PLA) supplied by the company Clas Ohlson. This material has a good shelf life even if stored exposed to open air, and becomes good quality prints with little effort. The material has no datasheet containing information regarding yield strength, Young's modulus and ultimate tensile strength which is what we need to gather from this test.

1 Standards

These are the standards we have followed when performing the tests. In the second test there is a deviation from the test specimen specified in ISO-527-2-2012

ISO-5425-2023: Specifications for use of poly(lactic acid) based filament in additive manufacturing applications[2]

ISO-527-1-2012: Determination of tensile properties Part 1:General principles[3]

ISO-527-2-2012: Determination of tensile properties Part 2:Test conditions for moulding and extrusion plastics[4]

2 Test equipment

The equipment used in this test:

- 3D-printer: Ender-3 v2
- Tensile test machine: LLOYD LR 10K
- Computer with test-software

2.1 Ender-3 v2

HB | AEH

The Ender-3 v2 is a 3D printer that has an exposed printing surface. This means that the environment of the print is the same as in the room the printer is placed in. This can affect the print, if the printer is placed in a poor environment. Fortunately, normal room temperature (around 20 degrees Celsius) and normal indoor air moisture are good enough for simple prints, with a simple filament type such as PLA.

2.2 LLOYD LR 10K

HB | AEH

The LLOYD LR 10K is a tensile testing machine with a maximum load capacity of 10.0 kN and a speed range of 0.1 to 500 mm per minute. [5] The machine is fitted with wedge grippers to fasten the test piece, increasing clamping force as more force is applied in the stretching direction. The disadvantage with the wedge grippers is at the start of the tensile test, the test piece can slip if it is not fitted correctly and/or the test piece consists of a material that has a low friction coefficient. To increase the friction between the test piece and the wedge grip, the surface of the jaw insert of the wedge grips has a serrated surface.

2.3 Computer with test-software

HB | AEH

The LLOYD LR 10K is operated using software from an external computer. This program is by the time of this test newly developed by one of the students at the university. Although faults may lie in the program, it has been approved by the professors and the results gathered seem logical.

Before a test can be carried out, some steps must be completed. The first step is to link the test file to a folder on the computer. Then some information needs to be put into the software. These values are the length between the crossheads, the thickness, and the width of the test specimen.

While the test is carried out the software plots the stress-strain curve in real time. This is great for monitoring for faults during the test. After a test has been carried out, some more steps need to be completed. These steps are to double-check that the automatically generated points, such as Ultimate tensile strength (UTS), fracture, and the proportional limit are correct. In addition, you have to choose if some of the test data needs to be adjusted. These adjustments are in cases where the start point is wrong and the machine moves a distance before putting a load on the specimen. In these cases, the zero point can be moved so that the elongation is correct.

After the adjustments are made the calculate button can be pushed and the results will be displayed. If you want to keep the results, press save. Now the data will be transferred

to an Excel document, and the raw file itself will also be saved inside the folder to which you assigned the test file.

3 Tensile test T1

3.1 Preparations

AEH | HB

Following ISO standard 527-1 section 7.1, the mean value should be calculated from at least five test specimens and the results should not differ from the mean value by more than 20%. We produced seven test pieces to have extra in case we encountered problems during the test. The dimensions of the test pieces can be found in appendix A.1. The Test pieces were 3D-printed with the following settings:

Settings	Used
Orientation	Flat
Nozzle	0.4 mm brass
Layer height	0.2 mm
Walls	3
Infill	100 %
Nozzle temperature	220°C
Bed temperature	60°C
Printing speed	60 mm/s

Table 3.1: Test T1 specimen manufacturing settings for Ender-3 V2.



Figure 3.1: Test specimen

3.2 Test

HB | AEH

The tests were performed in a room at 24 degrees Celsius.

We started by turning on the tensile testing machine and the computer connected to the machine. Then we had to find out how to use the program, due to new software on the computer with a completely different interface, than the program that we learned to use this machine on. Luckily the person who made the software had made a couple of video tutorials, but it still took some time to understand everything. So we ended up watching the tutorial and doing the first test together, step by step.

Before we started the tests, we made a new folder on the computer. This was to keep track of our test results, and easier to transfer the data to our storage device, for later review of the test results. Then we assigned the first test file to this folder and filled in the different required data for executing the test. Then we inserted the first test piece into the tensile testing machine and started the test. Because this was the first test piece, we struggled a bit to make the test piece attach itself properly. The machine moved almost 4mm before the clamps started to get a grip on the test piece. Then we selected the data we wanted and saved the results into the folder. The same happened to many of the other pieces as well, the worst one was test piece T1-4 where we failed to get a grip, and the test had to be restarted.

The test consisted of seven seemingly equal test pieces. This is also visible on the stress-strain curve, where the point of UTS and the fracture were very similar in every test. The biggest difference in the tests was the start. On several of the tests, it is visible on the stress-strain curve that the tensile testing machine was struggling to get a proper grip on the test pieces. There were only two exceptions, test pieces T1-2 and T1-7. These test pieces got a grip almost instantly.

We tested them one by one and the test program then logged the results automatically into the program. Then we adjusted some of the automatically generated points and discarded some of the start on the stress-strain curve. The amount discarded was very varying, due to some human errors during the testing. The different errors and other things that may have affected the result of the tests are written about in the Deviations chapter.

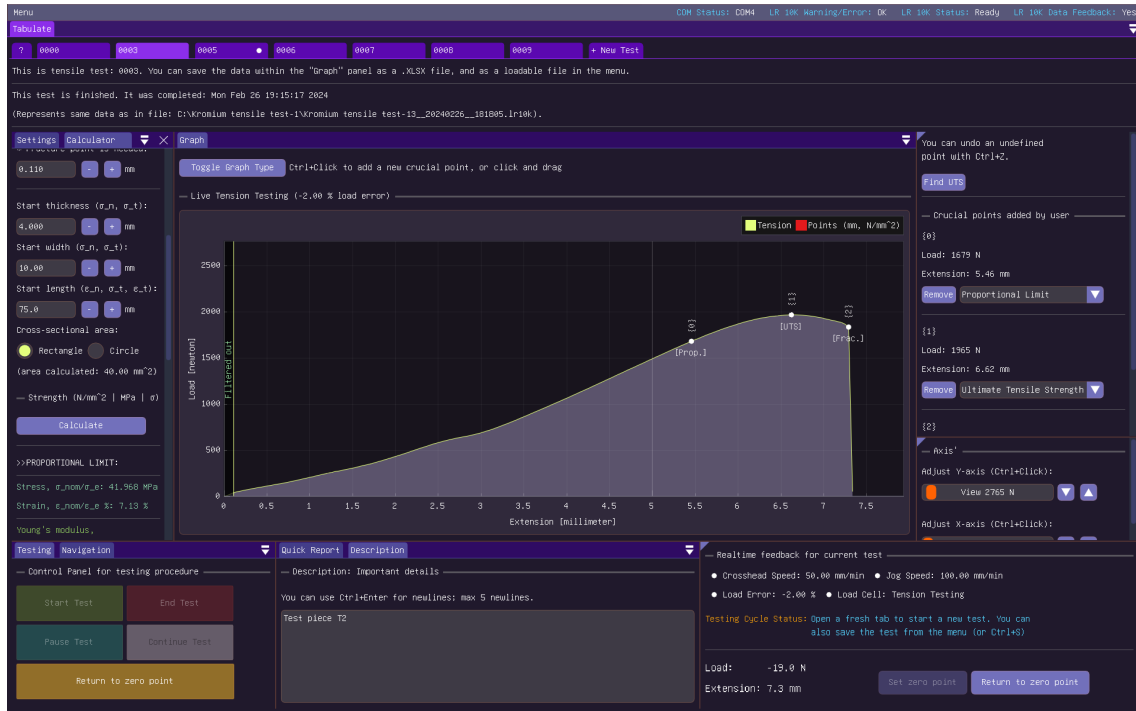


Figure 3.2: Screenshot of the software interface after a test

3.3 Results

HB | AEH

In the picture below is the fractured test specimen. Below the test specimen is a table containing the main relevant values from each test. The rest of the test data can be found in appendix B. If you look closely at the fracture points of the test specimen and compare them to the test values below, there is a pattern. Test specimens T1-2 and T1-7 are the only ones that had a straight fracture point, these were the only two test pieces that did not slip during the testing. It is also visible on Young's modulus which specimen slipped, these values are higher than the ones that did not slip. The more the pieces slipped, the higher the Young's modulus became. It is important to note that Young's modulus for the test specimen is only an approximation because polymers do not have one consistent Young's modulus like metals. Instead, the module changes depending on where you look at it. This is because the thermoplastic polymer chains don't behave the same way as atoms in metal. When the thermoplastics are exposed to forces still in the stress-strain curve's elastic region, the polymer chains untangle themselves stretch parallel to the applied force and become anisotropic.

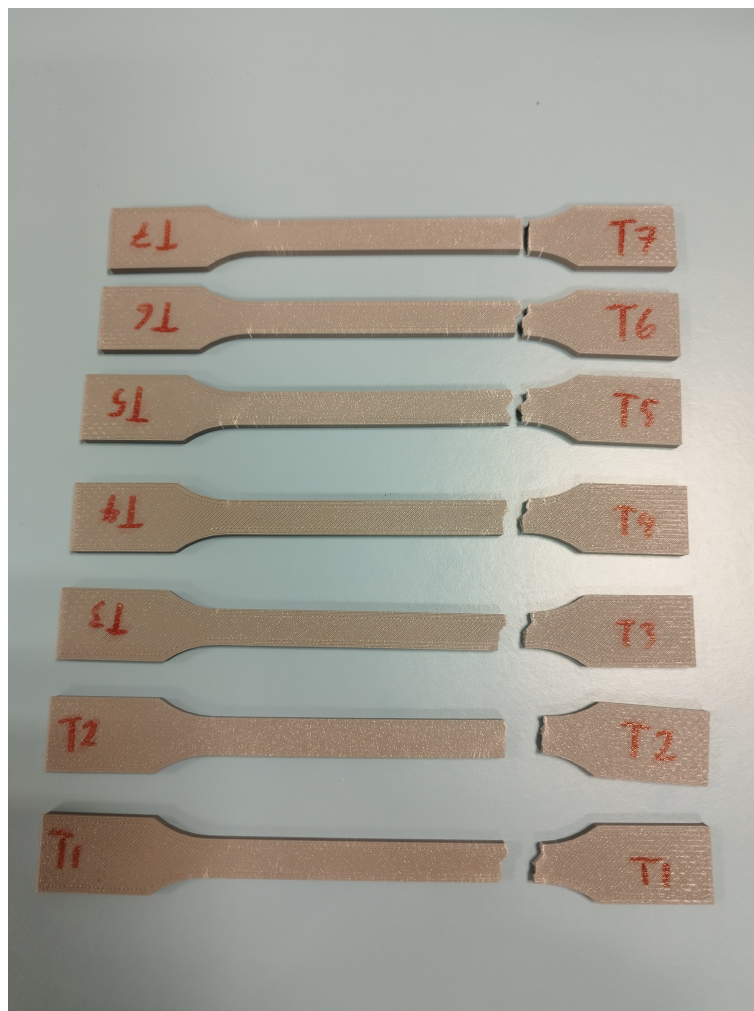


Figure 3.3: The fractured test specimens

Tensile test T1 (true stress) (MPa [N/mm ²])									
Test:	T1-1	T1-2	T1-3	T1-4	T1-5	T1-6	T1-7	Median	Mean
Ultimate tensile strength	50,401	53,390	50,214	49,704	49,860	50,373	49,960	50,214	50,557
Fracture:	47,337	50,203	48,363	47,928	46,764	47,170	47,447	47,447	47,887
Young's modulus:	696,489	588,563	684,626	737,726	879,383	923,334	572,496	696,489	726,088

Table 3.2: The tensile testing results of PLA. Colour-coded if the test is valid or if it failed.

3.4 Deviations

AEH | HB

From the results we found that five out of seven test pieces were invalid, however, some of the results are. The valid results are the UTS and fracture values, while the invalid results are Young's modulus. The reason the UTS and fracture values are valid results is that these values are calculated from the amount of applied stress (Newton) over the cross-section area (millimetres). Young's modulus results are not valid because these are calculated with the use of elongation in the direction of applied force. The invalid test pieces were slipping during the testing, which led to an inaccurate elongation reading, this can be observed on the stress-strain curve shown below.

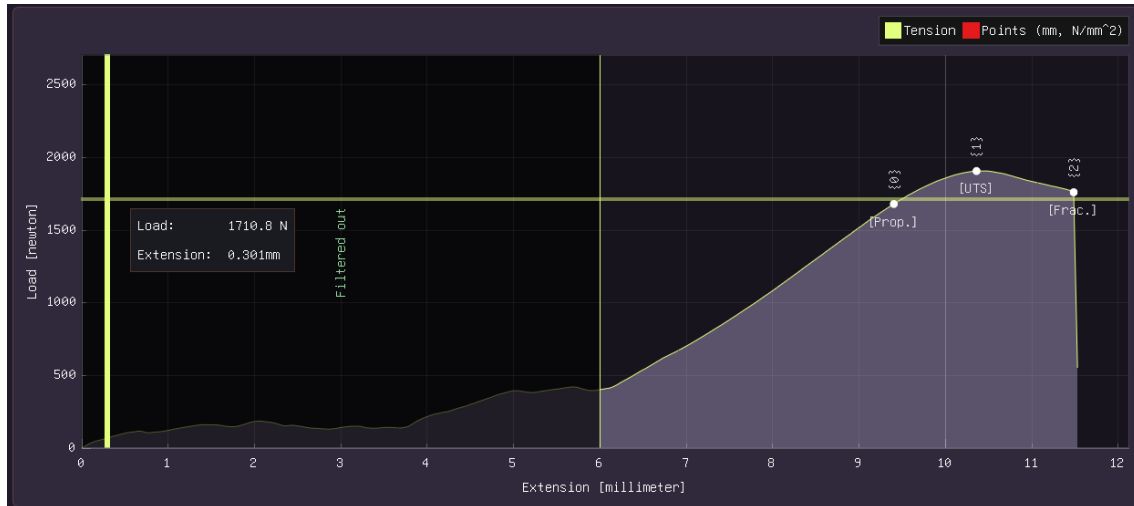


Figure 3.4: An inaccurate test result, due to improper attachment

As seen on the graph, there is a lot of movement (6mm) before the load increases with a significant value. And the Young's modulus for this test piece is becoming very inaccurate. Evidence of this is the failed first try of test T1-4 as well as the lines on five out of seven test pieces. These lines as shown below are a result of the grip slipping before getting a hold.

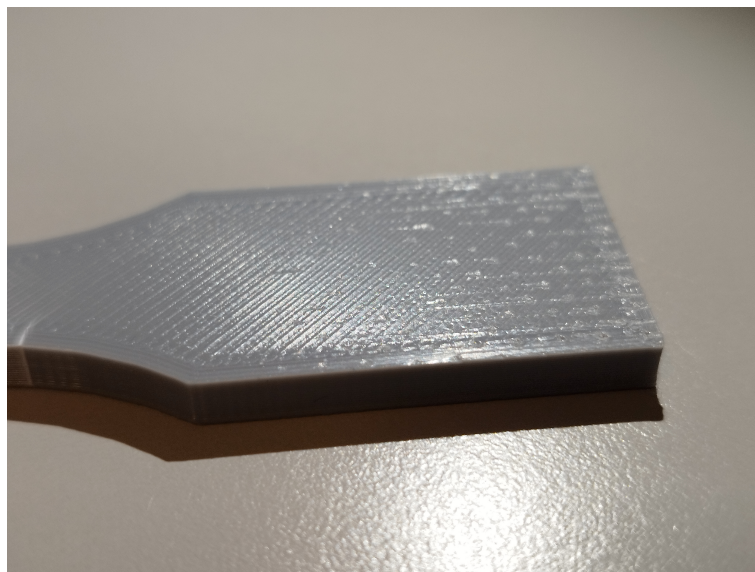


Figure 3.5: Evidence of scraping

All the test pieces broke outside the gauge length area. The reason for this is likely

because of the production method, all the pieces break in the same area which is the start of the radius where the width expands. As we can see in the picture the radius is made up of many straight lines that create edges where the tension will accumulate.

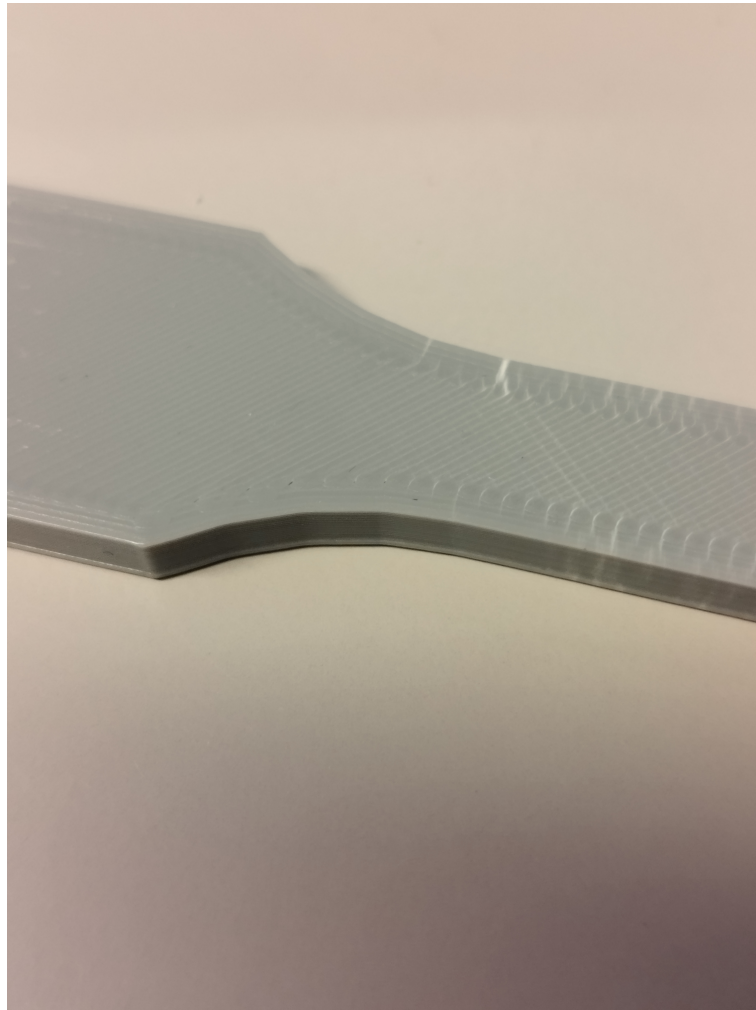


Figure 3.6: Corners on the radius

3.5 Conclusion

HB | *AEH*

The reason we choose to believe that the UTS and fracture values are somewhat correct, is because the values from test T1-2 are at the higher end of the results, while the values from test T1-7 are at the lower end of the scale. This indicates that the values for UTS and fracture for the other test pieces are within range of what they would be if they didn't slip during the testing.

Because only two of the tests can be defined as valid tests due to slipping, we will not be able to use the values we got for a material specification in Solid Works. However, the test results that are categorized as valid give us a good indication of an approximate value. But our team want to have as accurate values as possible, so a new test will be carried out later on.

4 Tensile test T2

4.1 Preparations

AEH | HB

Since the deviations from the first test showed an indication of scraping, we chose to go for the university's test specimen design which can be found in appendix A.2. These test specimens are designed so that the ends can be secured properly within a holder. For this test, we produced eight test specimens. The settings for manufacturing the test specimen can be found in the table below.

Settings	Used
Orientation	Flat
Nozzle	0.4 mm brass
Layer height	0.2 mm
Walls	3
Infill	100 %
Nozzle temperature	220°C
Bed temperature	60°C
Printing speed	60 mm/s

Table 4.1: Test T2 specimen manufacturing settings for Ender-3 V2.

4.2 Test

AEH | HB

Each specimen test was performed following the same steps as discussed in 3.2 except the way that they were fastened. For this test, the specimens were secured by putting the holders shown below, on both ends and then placing them in the test machine.

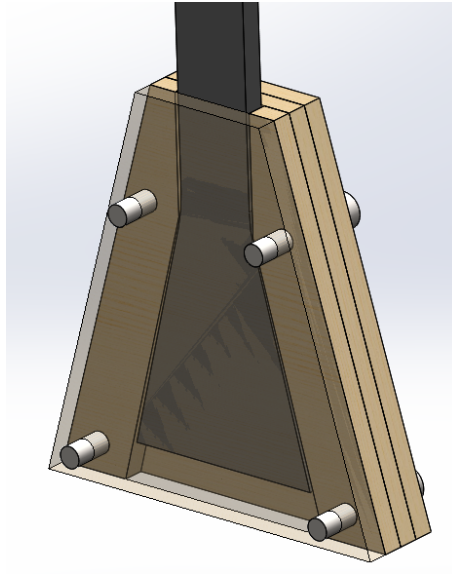


Figure 4.1: The test specimen put into a holder

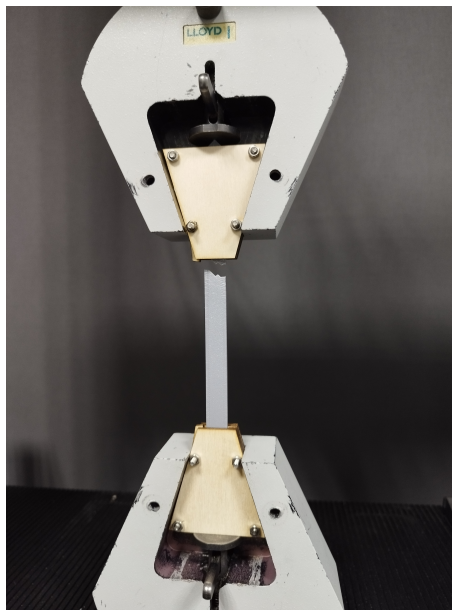


Figure 4.2: Test specimen placed in test machine

4.3 Results

AEH | HB

The tables below contain the main interest data. The rest of the collected data can be found in appendix C.



Figure 4.3: The test specimen after the test

Tensile test T2 (true stress) (MPa [N/mm ²])										
Test:	T2-1	T2-2	T2-3	T2-4	T2-5	T2-6	T2-7	T2-8	Median	Mean
Young's modulus	646,790	594,052	660,158	866,162	668,935	762,796	597,099	632,418	653,474	678,551
Yielding (lower)	35,004	32,884	40,930	35,658	30,983	31,205	35,576	42,666	35,290	35,613
Ultimate tensile strength	51,077	48,858	50,606	45,951	46,864	47,387	46,912	50,307	48,123	48,495
Fracture	47,800	45,736	47,806	43,176	43,711	42,506	45,370	45,646	45,508	45,219

Table 4.2: Table containing results of true stress from all tests as well as median and mean values. Red values deviate from the mean value by more than 20%.

Tensile test T2 (true stress) (MPa [N/mm ²])									
Test:	T2-1	T2-2	T2-3	T2-5	T2-6	T2-7	T2-8	Median	Mean
Young's modulus	646,790	594,052	660,158	668,935	762,796	597,099	632,418	646,790	651,750
Yielding (lower)	35,004	32,884	40,930	30,983	31,205	35,576	42,666	35,004	35,607
Ultimate tensile strength	51,077	48,858	50,606	46,864	47,387	46,912	50,307	48,858	48,859
Fracture	47,800	45,736	47,806	43,711	42,506	45,370	45,646	45,646	45,511

Table 4.3: Table containing results of true stress from the valid tests as well as median and mean values.

4.4 Deviations

AEH | HB

Small defects and deviations will occur during manufacturing and because of this, we can see that the test pieces break at different areas of the cross-section and different loads.

The test T2-4 has Young's modulus over 20% higher than the mean of the tests and is therefore excluded from the final calculation.

Other possible deviations may come from variations in the zero point, proportional limit and yield strength points, and wrong readings from the machine sensors.

4.5 Conclusion

AEH | HB

There were performed eight tests this time with a new design that properly secured the ends and prevented slipping. As a result, seven out of the eight tests were usable and gave us a proper estimate of the material strength properties.

5 References

- [1] Kromium. Kromium website. [Online]. Available: <https://itfag.usn.no/grupper/D03-24/>
- [2] International Organization for Standardization. ISO 5425:2023. [Online]. Available: <https://www.iso.org/standard/81238.html>
- [3] ——. ISO 527-1:2019. [Online]. Available: <https://www.iso.org/standard/75824.html>
- [4] ——. ISO 527-2:2012. [Online]. Available: <https://www.iso.org/standard/56046.html>
- [5] Lloyd LR10k tensile machine. [Online]. Available: <https://www.tts-ltd.co.uk/lloyd-lr10k-tensile-machine>

A Technical drawings

A.1 Tensile test piece ISO 527

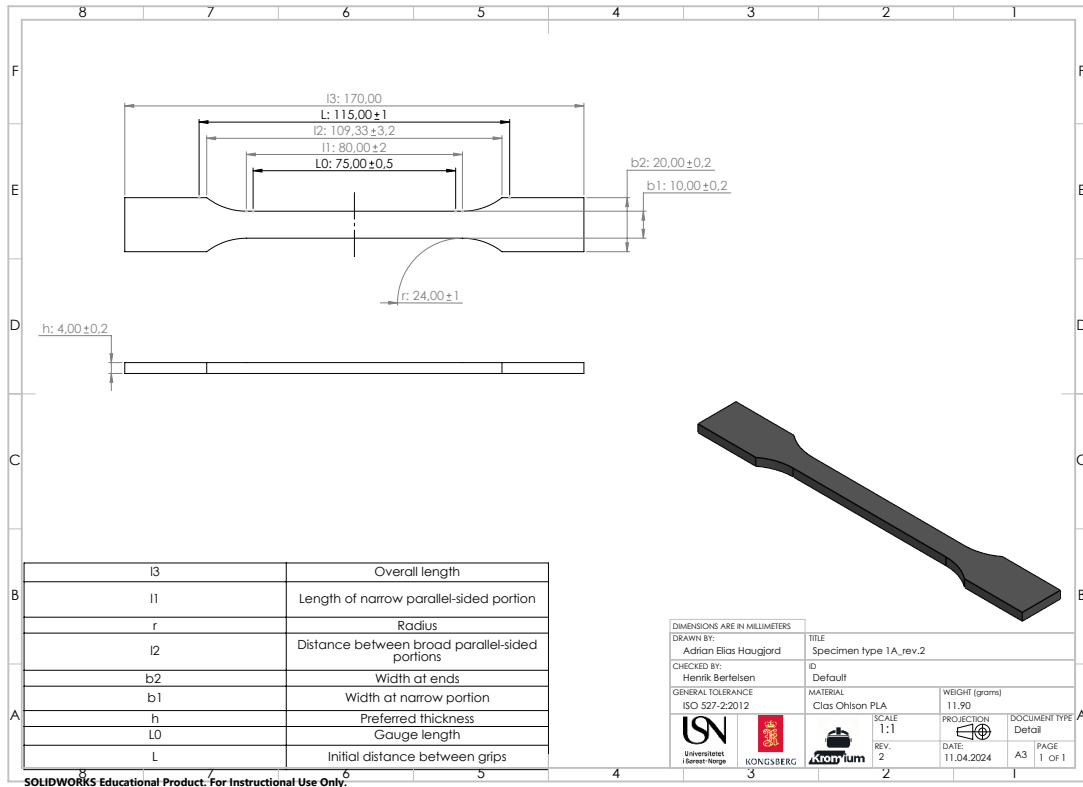


Figure A.1: Tensile test specimen type A dimensions technical drawing

A.2 Tensile test piece USN Kongsberg

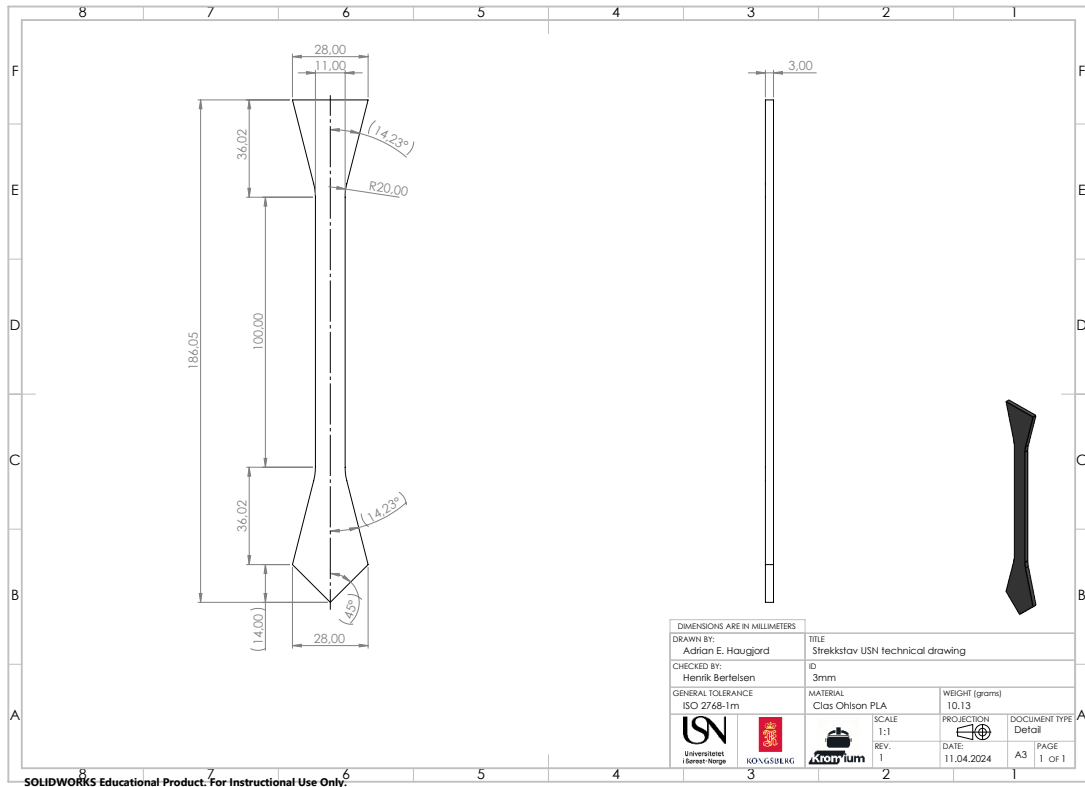


Figure A.2: Tensile test specimen used at USN Kongsberg

B Results test T1

B.1 Results T1-1

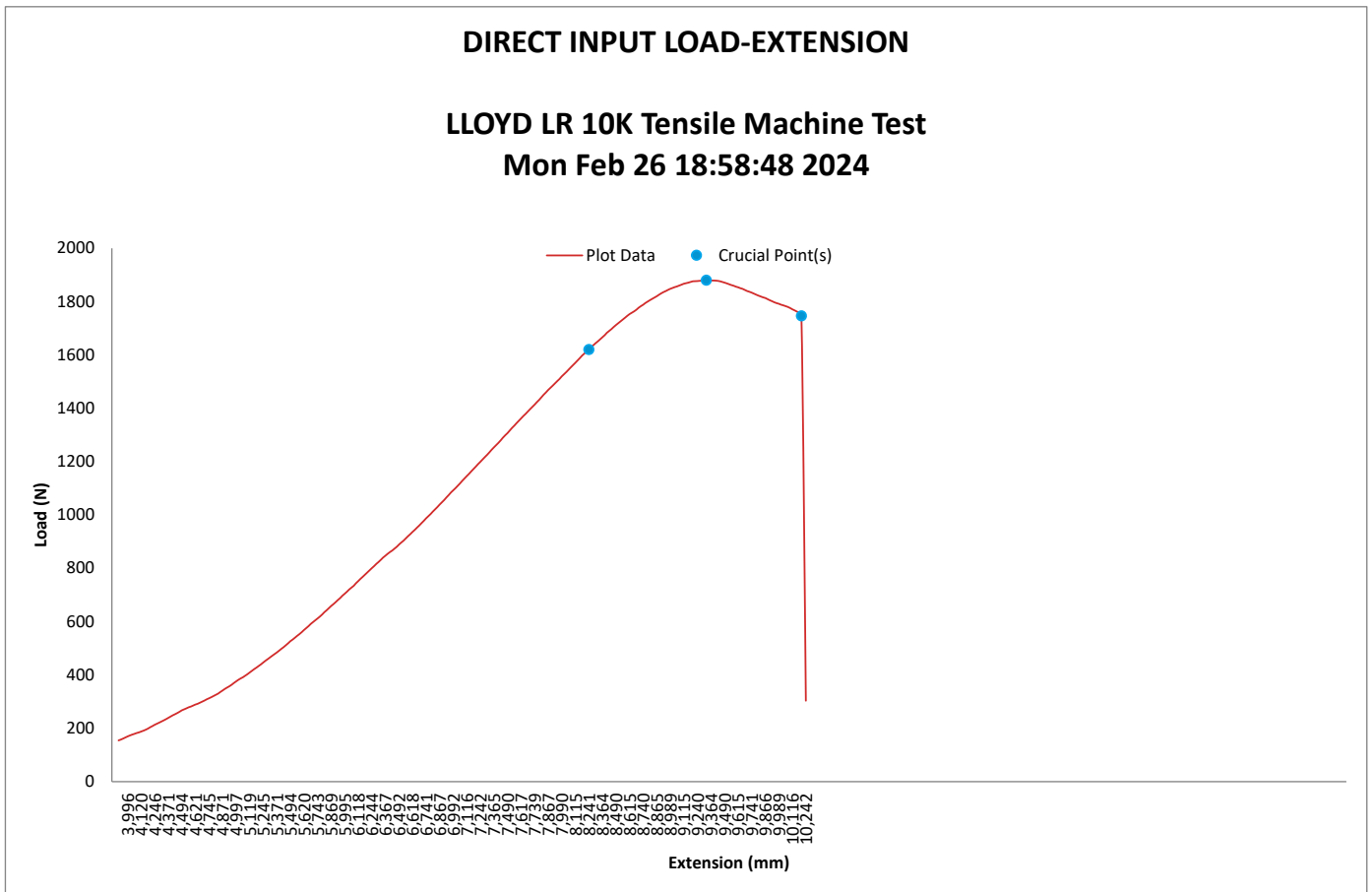
MATERIAL TESTING REPORT		
Mon Feb 26 18:58:48 2024		
Details:		
Test piece T1-1		
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,498	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,81%	
<u>Young's modulus, E</u>	696,489	MPa [N/mm ²]
True Stress, σ_t	42,853	MPa [N/mm ²]
True Strain, ϵ_t	5,65%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,991	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,26%	
True Stress, σ_t	50,401	MPa [N/mm ²]
True Strain, ϵ_t	7,00%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,659	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,43%	
True Stress, σ_t	47,337	MPa [N/mm ²]
True Strain, ϵ_t	8,09%	

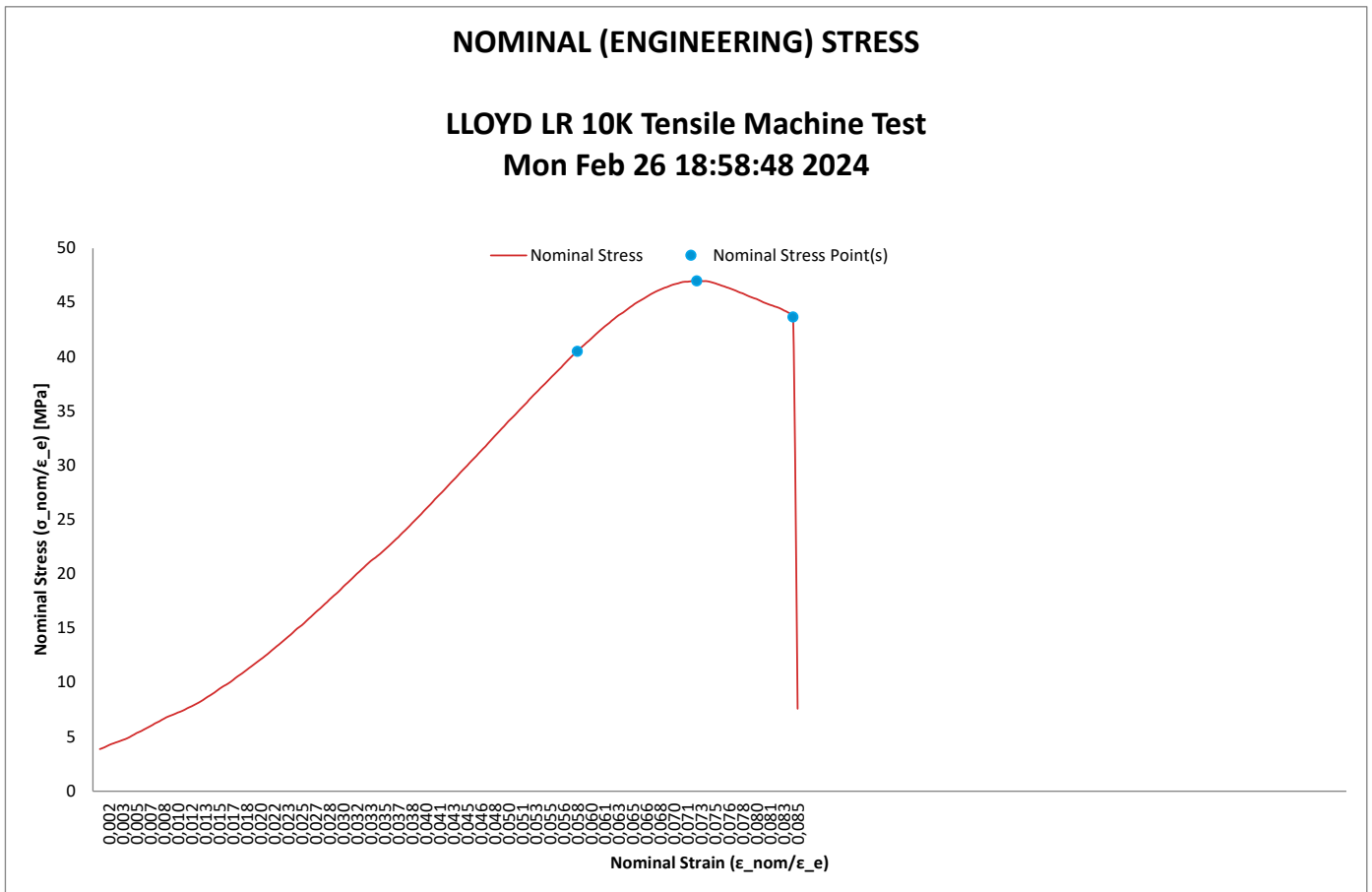
LR 10K Tensile Machine Settings

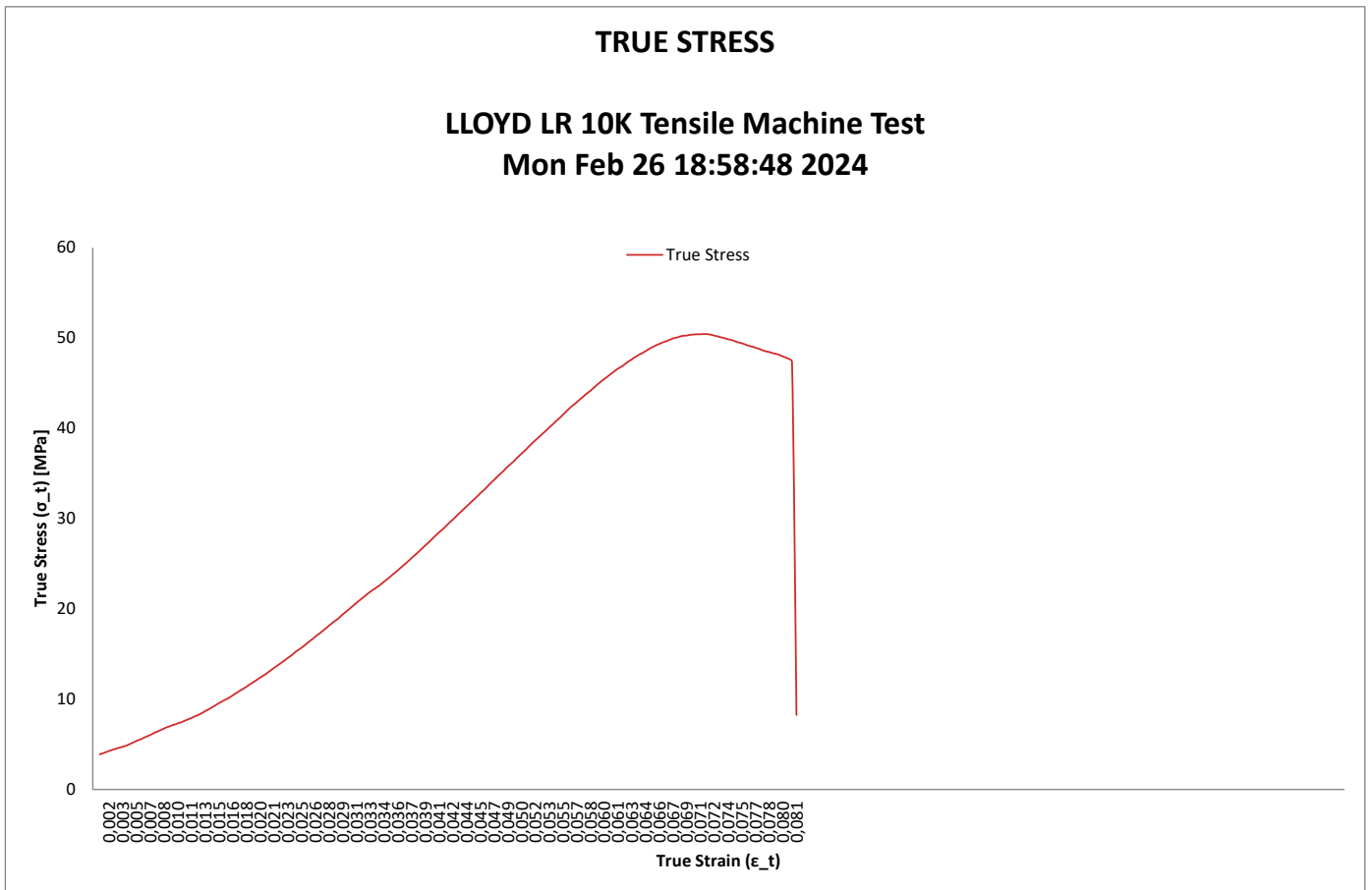
Mon Feb 26 18:58:48 2024

Details: ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





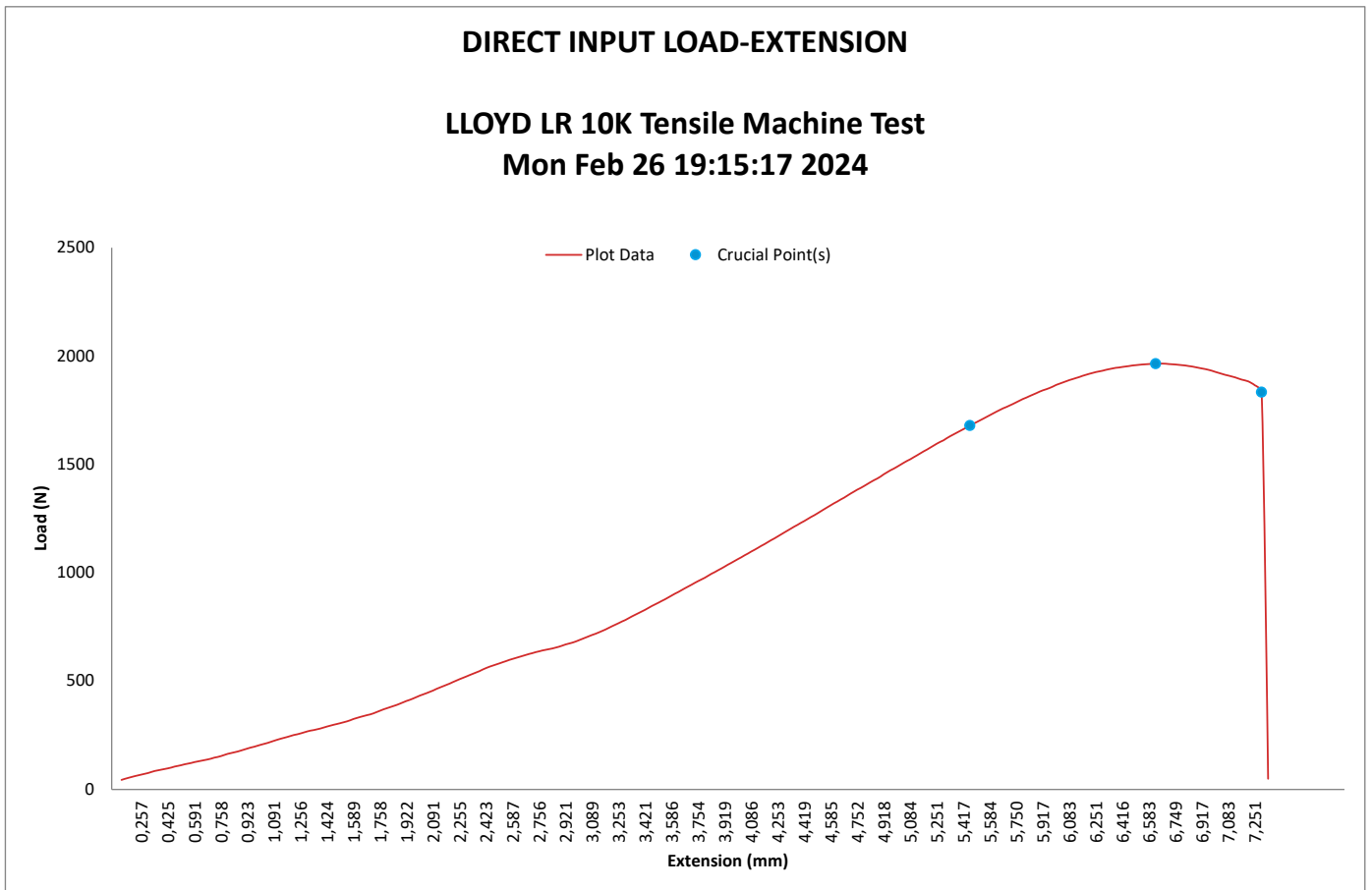


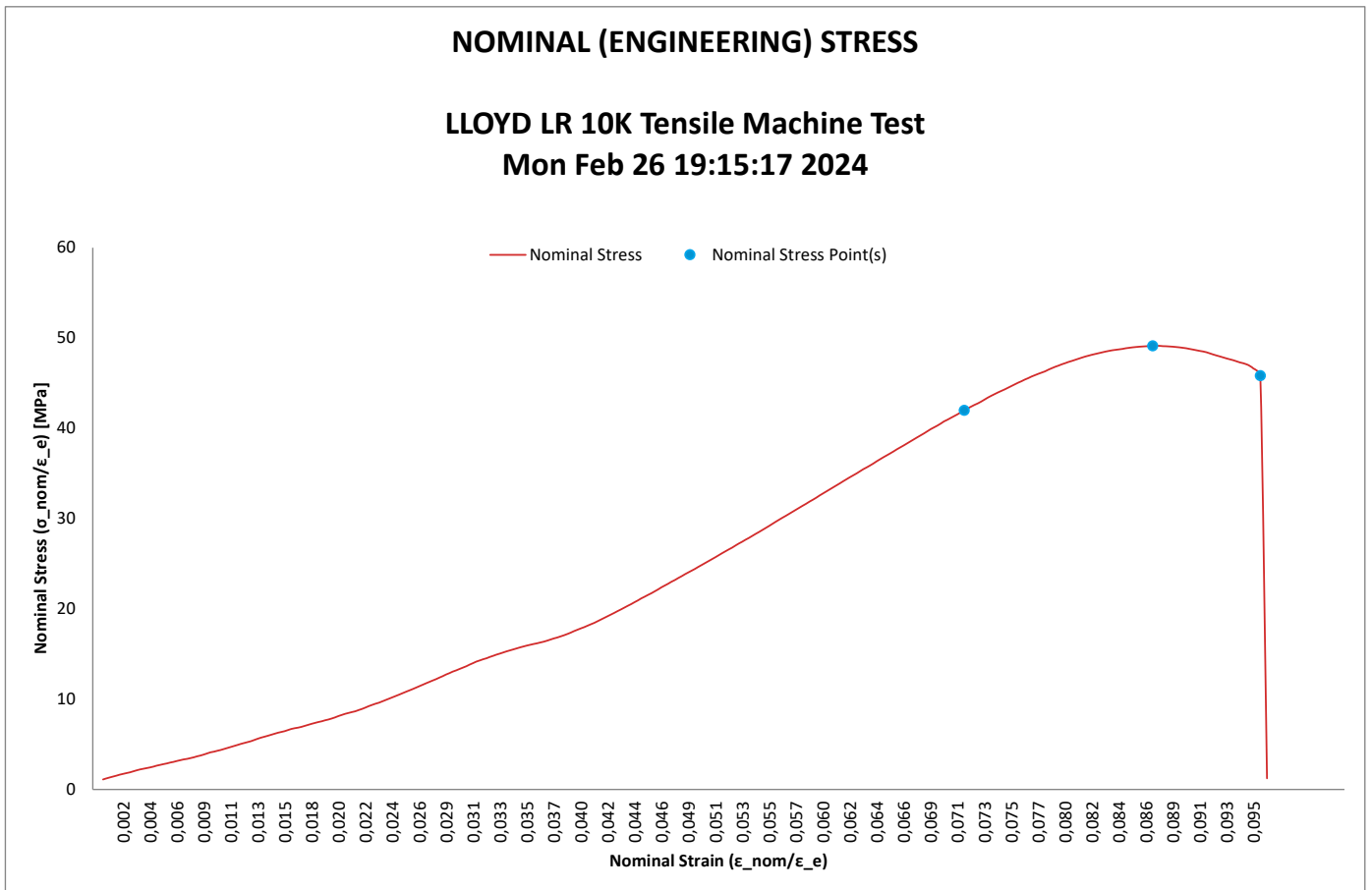
B.2 Results T1-2

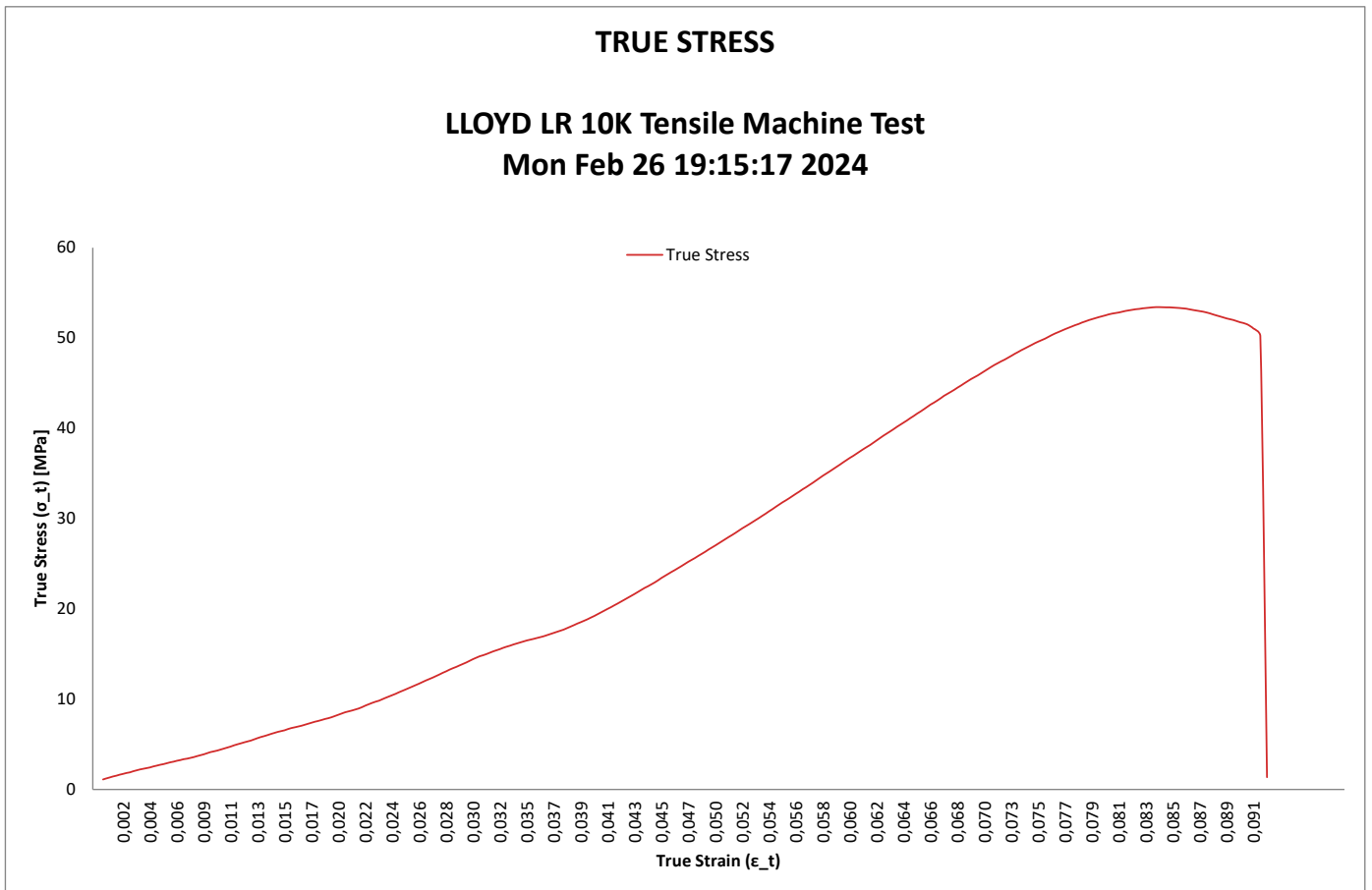
MATERIAL TESTING REPORT		
Mon Feb 26 19:15:17 2024		
Details:	Test piece T1-2	
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	41,968	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,13%	
<u>Young's modulus, E</u>	588,563	MPa [N/mm ²]
True Stress, σ_t	44,961	MPa [N/mm ²]
True Strain, ϵ_t	6,89%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	49,123	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,69%	
True Stress, σ_t	53,390	MPa [N/mm ²]
True Strain, ϵ_t	8,33%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	45,815	MPa [N/mm ²]
Nominal Strain, ϵ_n	9,58%	
True Stress, σ_t	50,203	MPa [N/mm ²]
True Strain, ϵ_t	9,15%	

LR 10K Tensile Machine Settings**Mon Feb 26 19:15:17 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%







B.3 Results T1-3

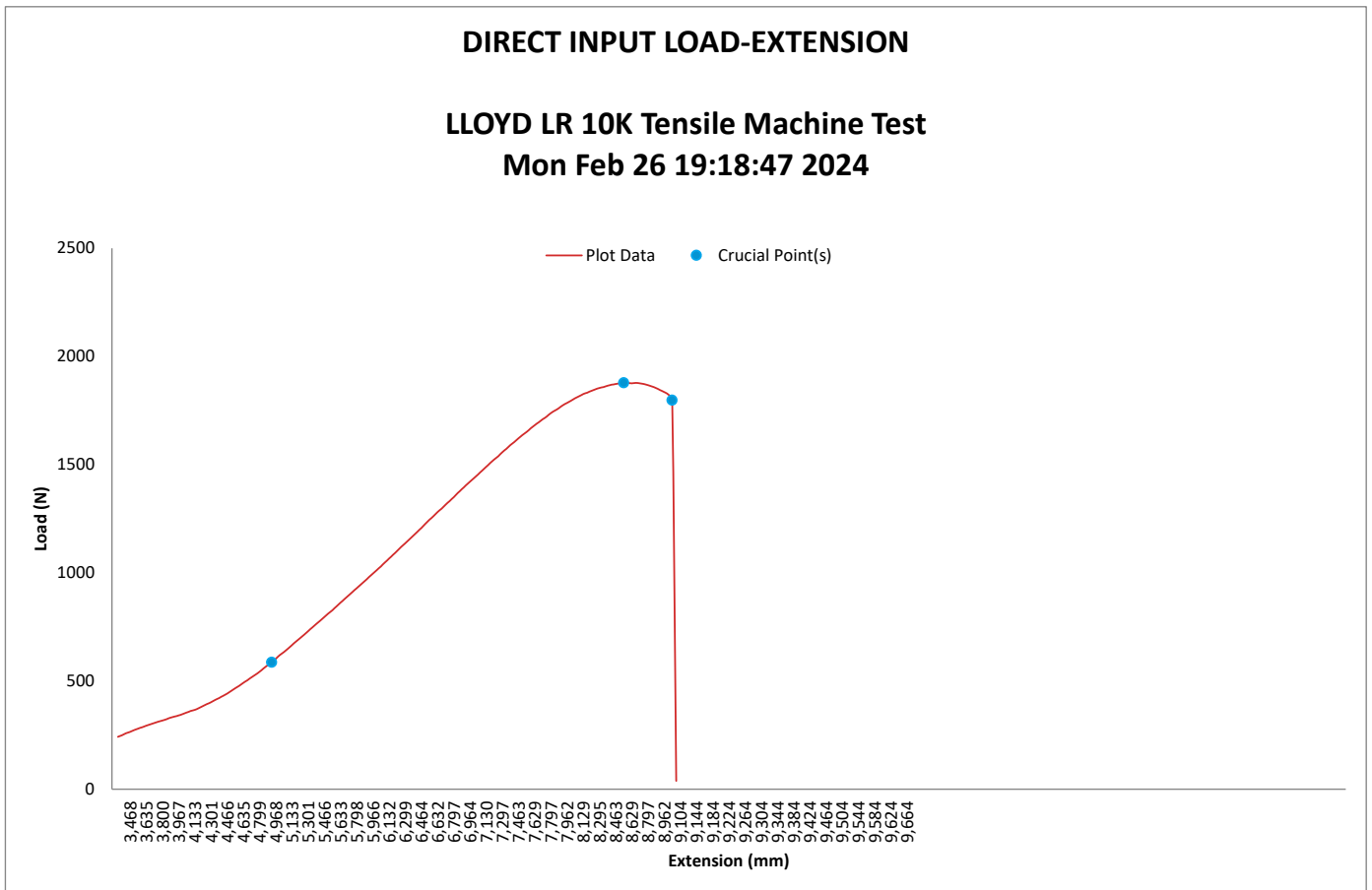
MATERIAL TESTING REPORT		
Mon Feb 26 19:18:47 2024		
Details:		
Tensile test T1-3		
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	14,651	MPa [N/mm ²]
Nominal Strain, ϵ_n	2,14%	
<u>Young's modulus, E</u>	684,626	MPa [N/mm ²]
True Stress, σ_t	14,965	MPa [N/mm ²]
True Strain, ϵ_t	2,12%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,942	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,97%	
True Stress, σ_t	50,214	MPa [N/mm ²]
True Strain, ϵ_t	6,74%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	44,933	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,63%	
True Stress, σ_t	48,363	MPa [N/mm ²]
True Strain, ϵ_t	7,36%	

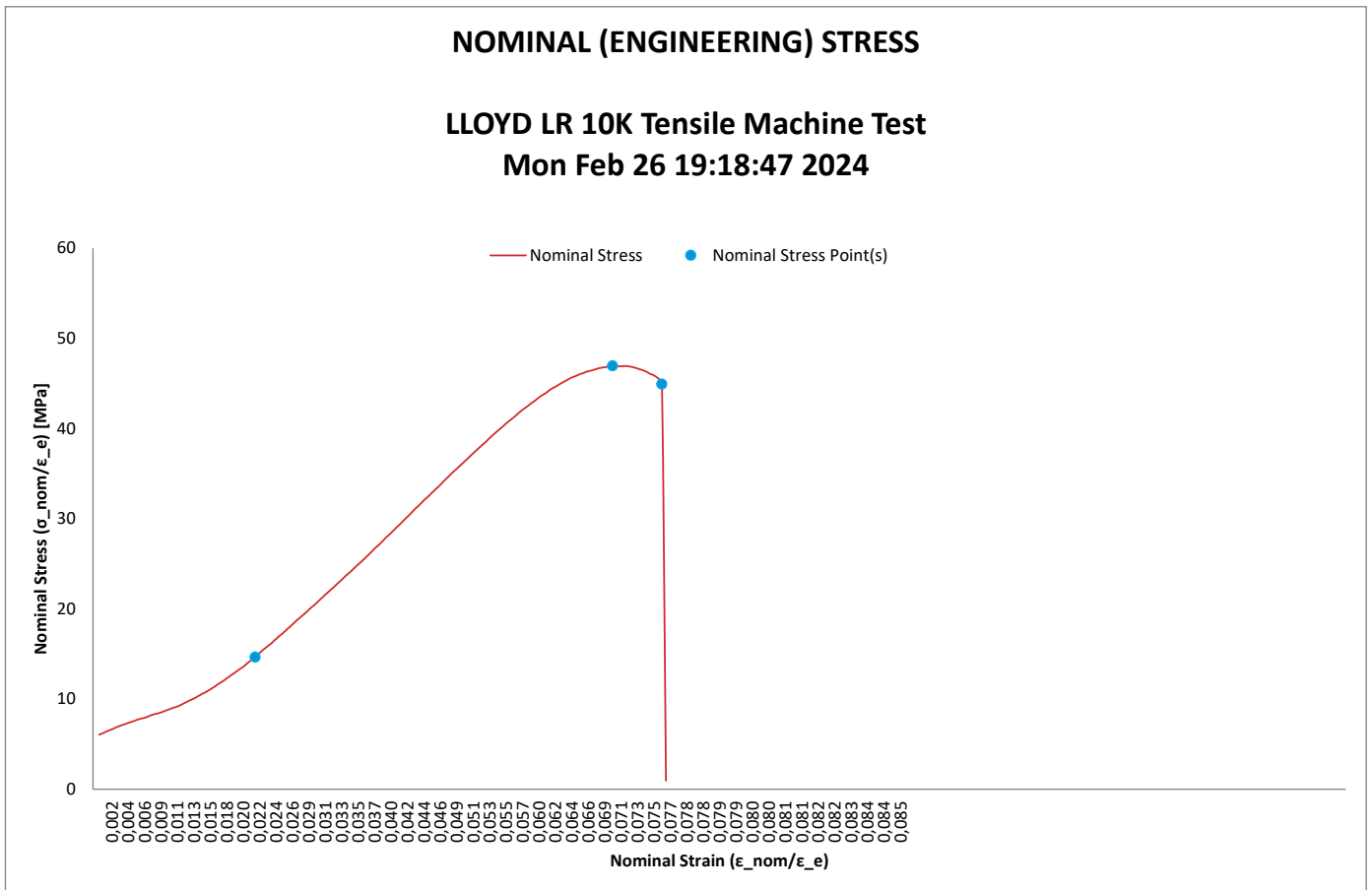
LR 10K Tensile Machine Settings

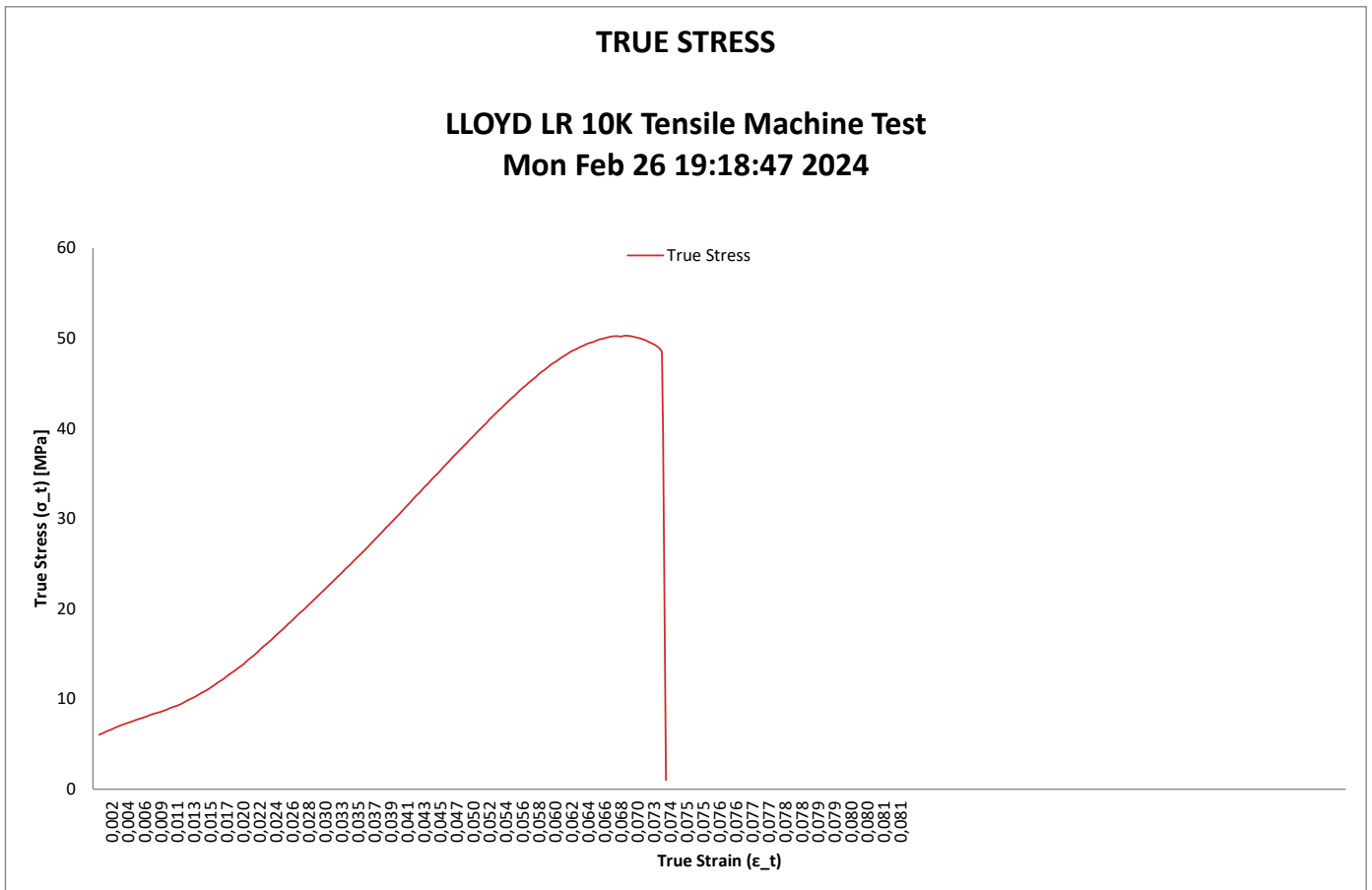
Mon Feb 26 19:18:47 2024

Details: ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





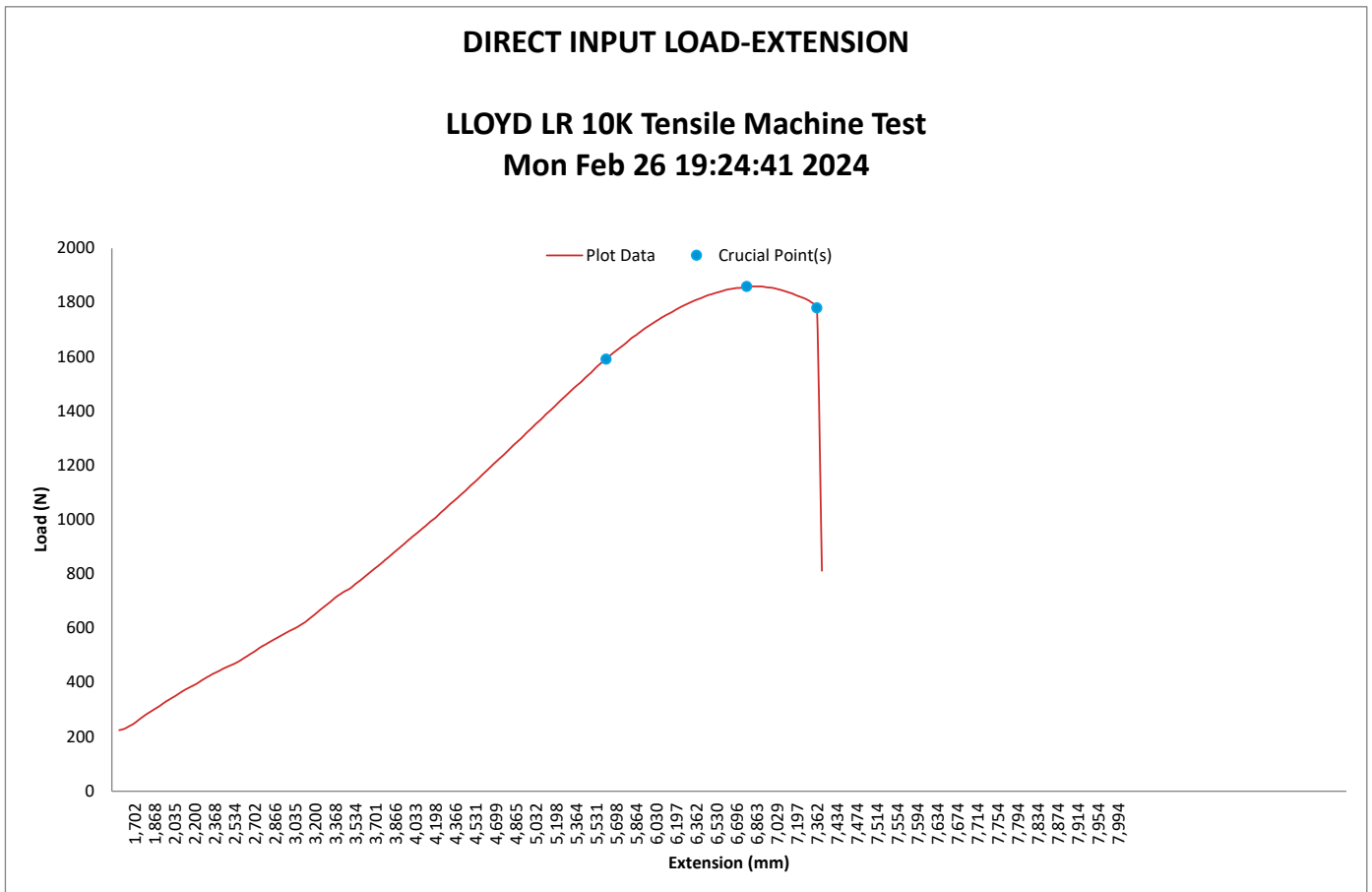


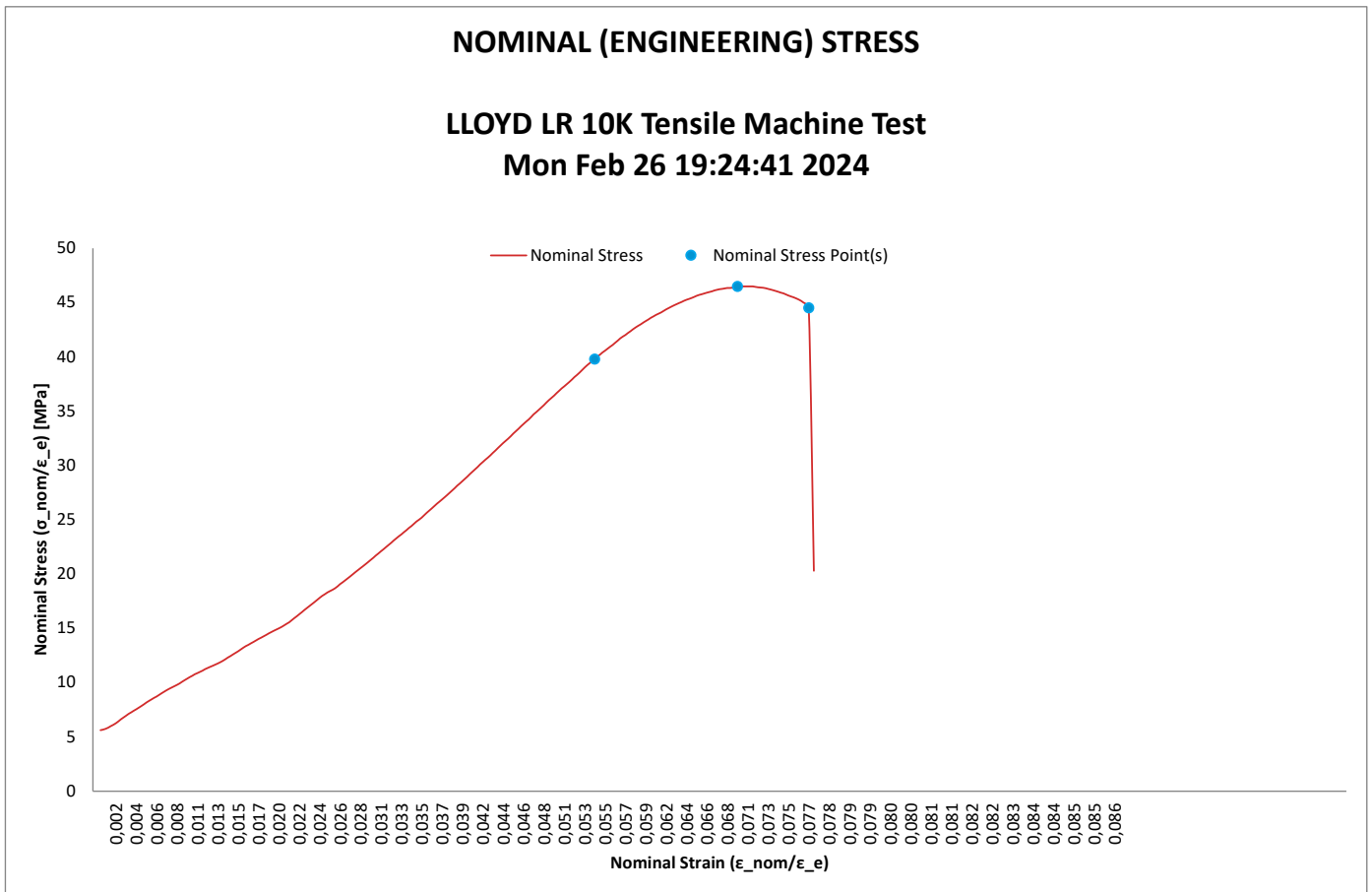
B.4 Results T1-4

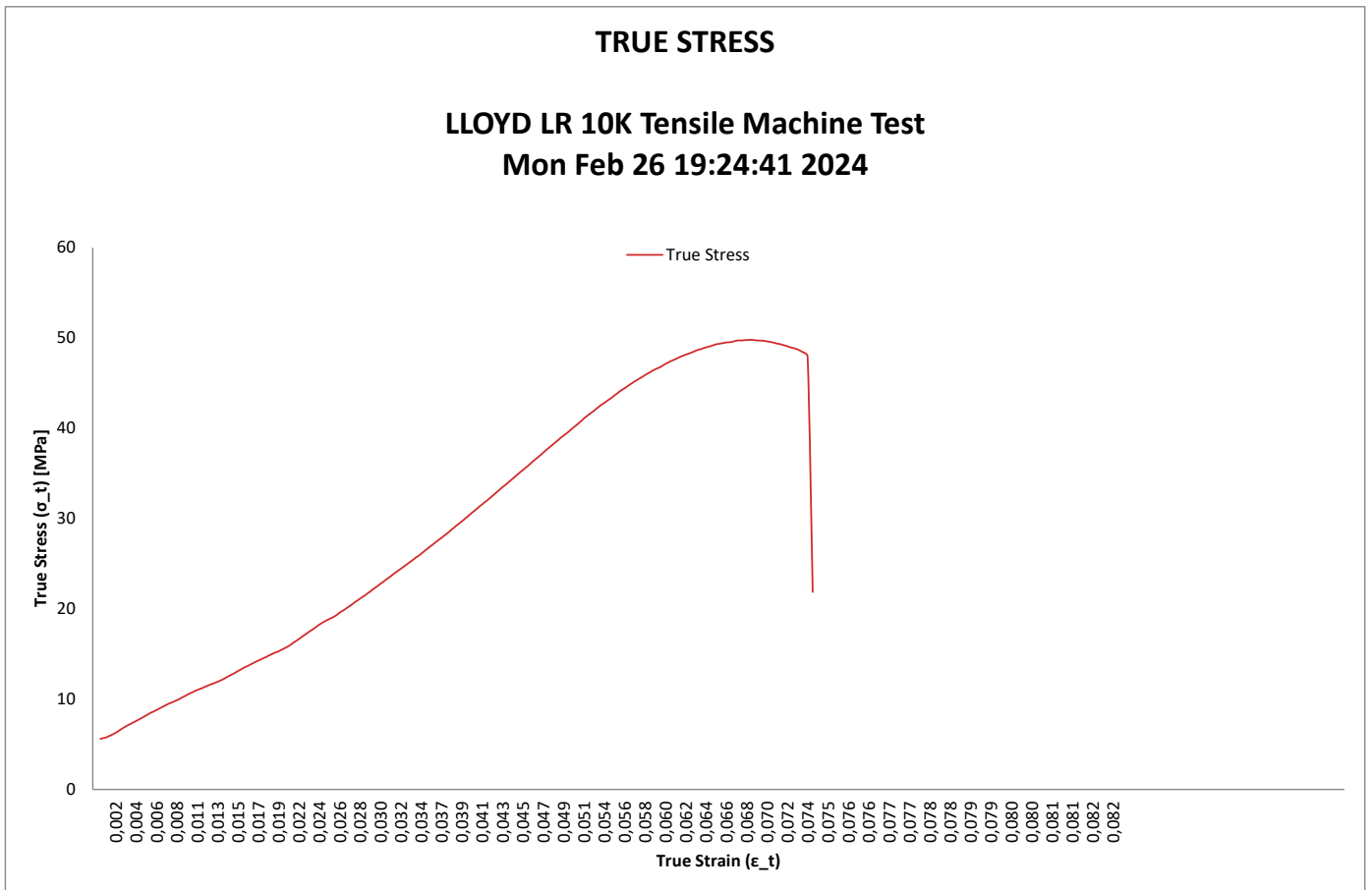
MATERIAL TESTING REPORT		
Mon Feb 26 19:24:41 2024		
Details:		
Test piece T1-4-2		
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	39,788	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,39%	
<u>Young's modulus, E</u>	737,726	MPa [N/mm ²]
True Stress, σ_t	41,934	MPa [N/mm ²]
True Strain, ϵ_t	5,25%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,477	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,95%	
True Stress, σ_t	49,704	MPa [N/mm ²]
True Strain, ϵ_t	6,71%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	44,492	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,72%	
True Stress, σ_t	47,928	MPa [N/mm ²]
True Strain, ϵ_t	7,44%	

LR 10K Tensile Machine Settings**Mon Feb 26 19:24:41 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





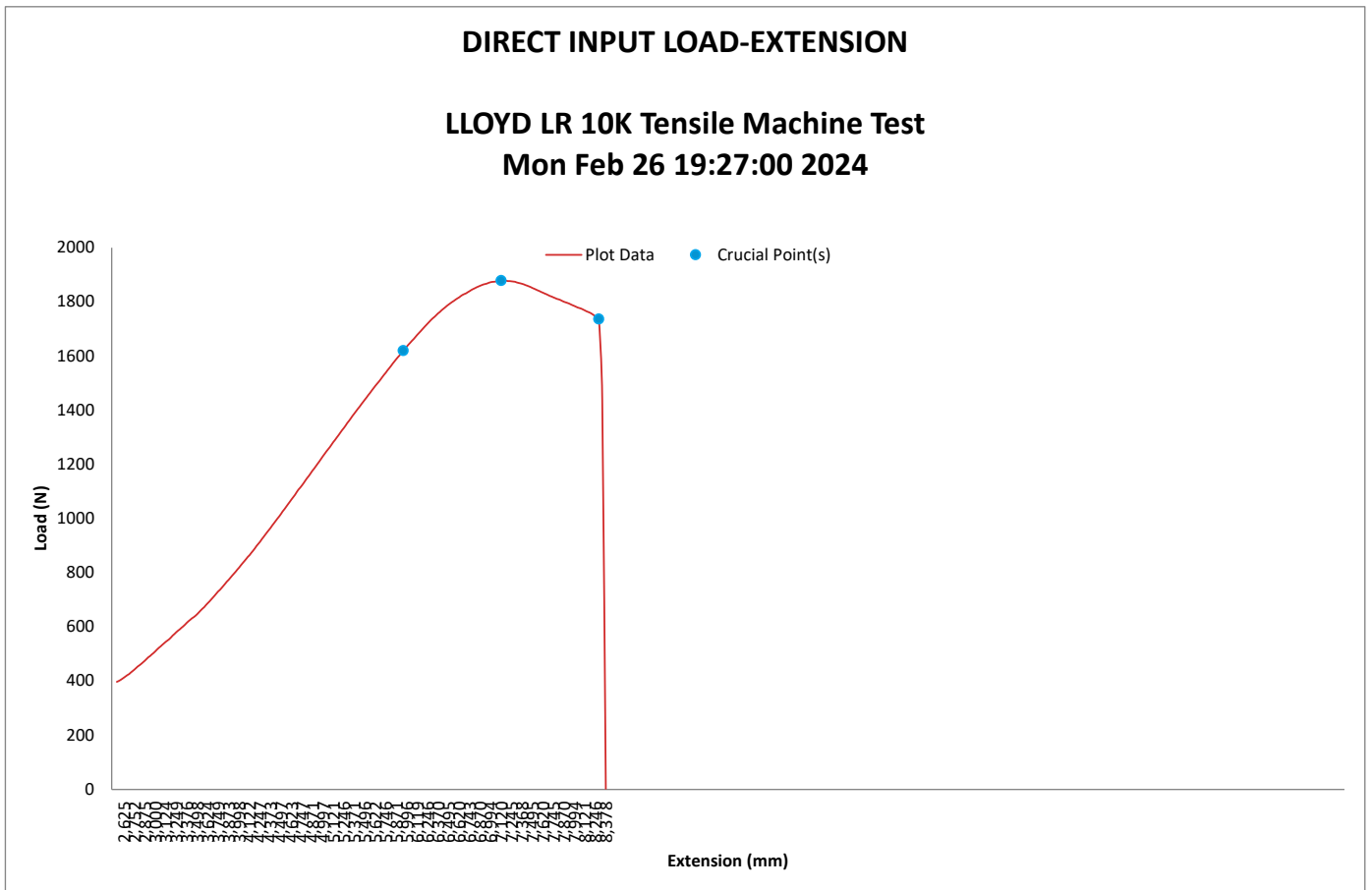


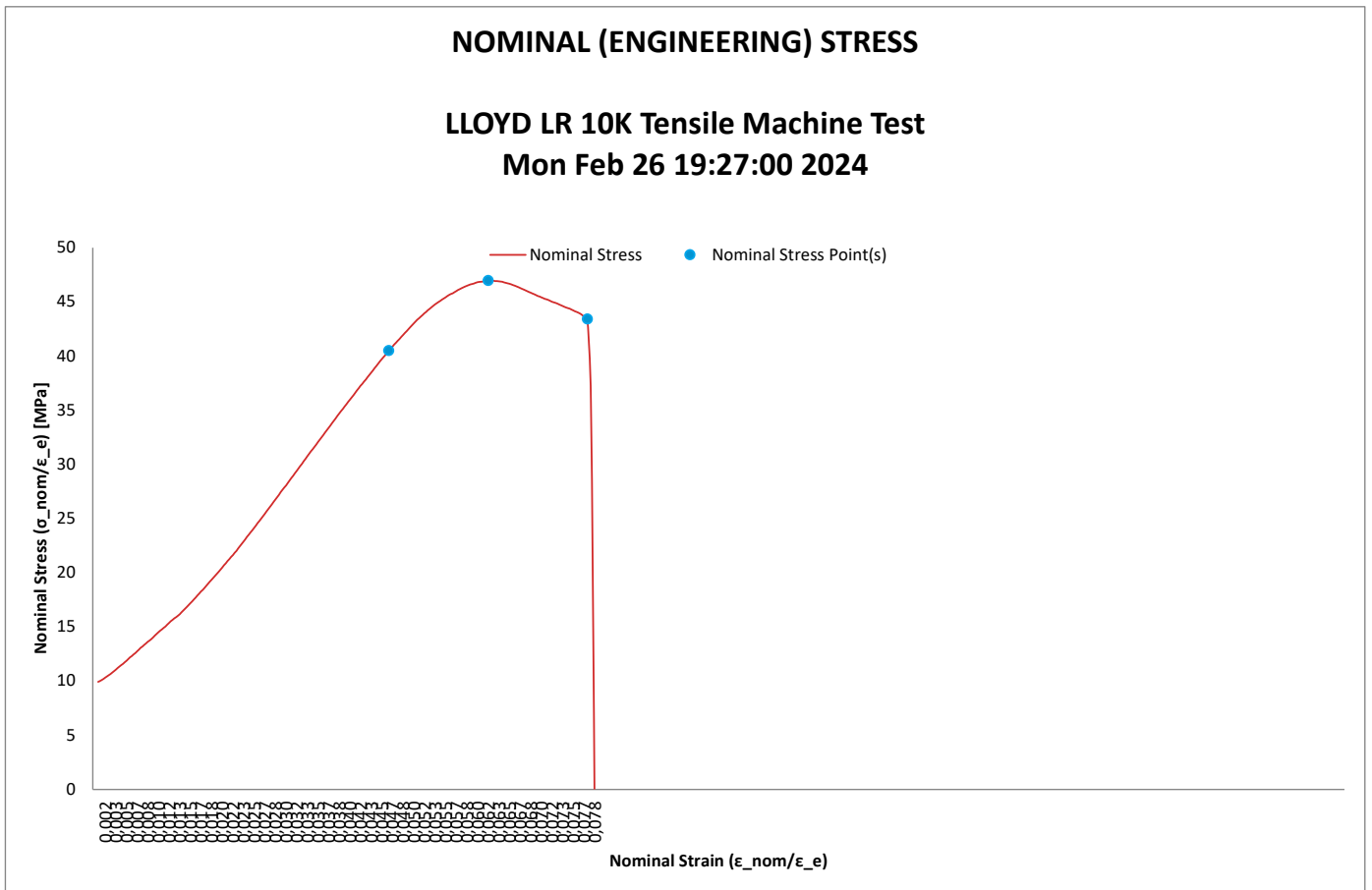
B.5 Results T1-5

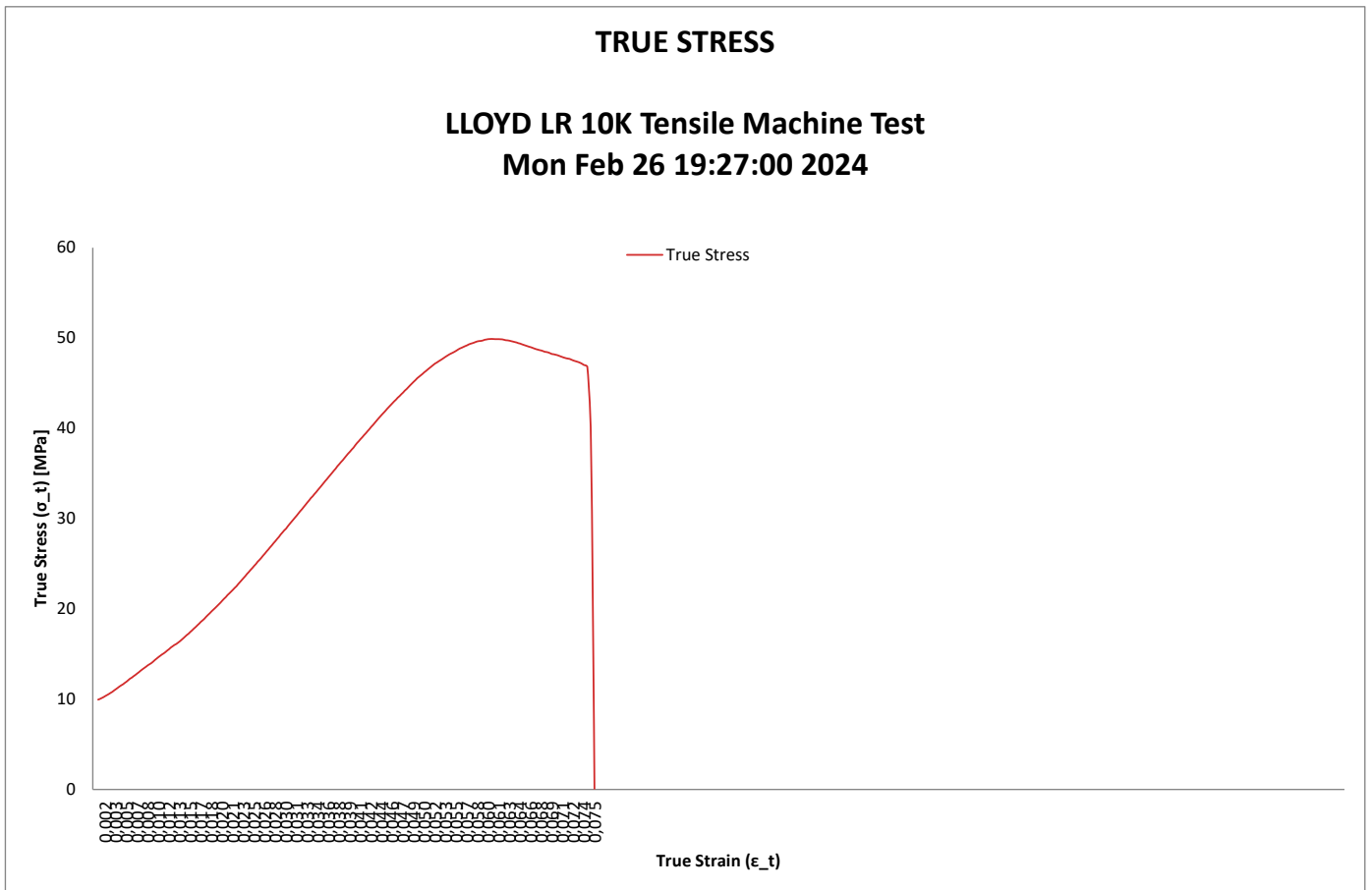
MATERIAL TESTING REPORT		
Mon Feb 26 19:27:00 2024		
Details:	Test piece T1-5	
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,498	MPa [N/mm ²]
Nominal Strain, ϵ_n	4,61%	
<u>Young's modulus, E</u>	879,383	MPa [N/mm ²]
True Stress, σ_t	42,364	MPa [N/mm ²]
True Strain, ϵ_t	4,50%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,967	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,16%	
True Stress, σ_t	49,860	MPa [N/mm ²]
True Strain, ϵ_t	5,98%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,414	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,72%	
True Stress, σ_t	46,764	MPa [N/mm ²]
True Strain, ϵ_t	7,43%	

LR 10K Tensile Machine Settings**Mon Feb 26 19:27:00 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





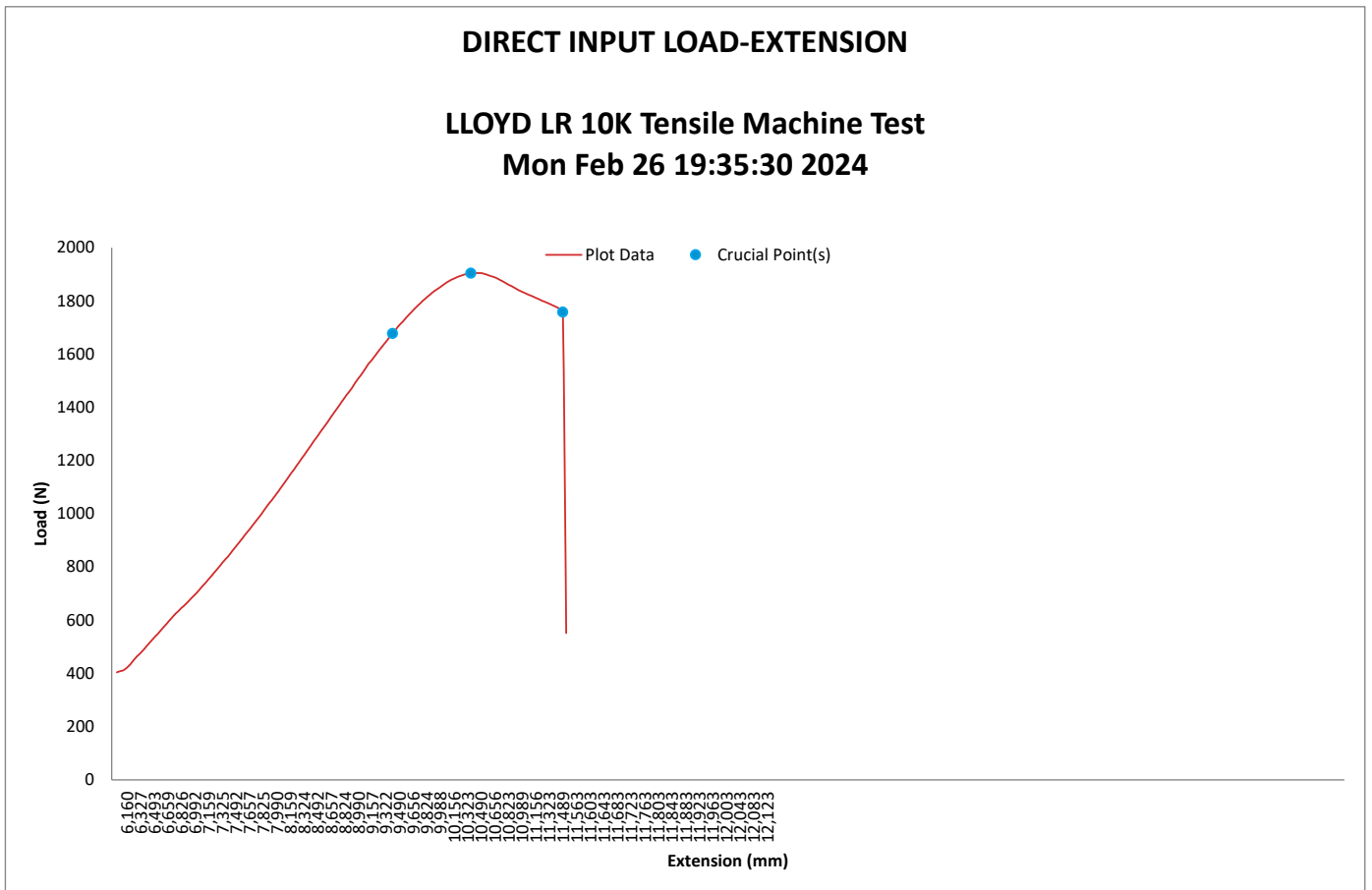


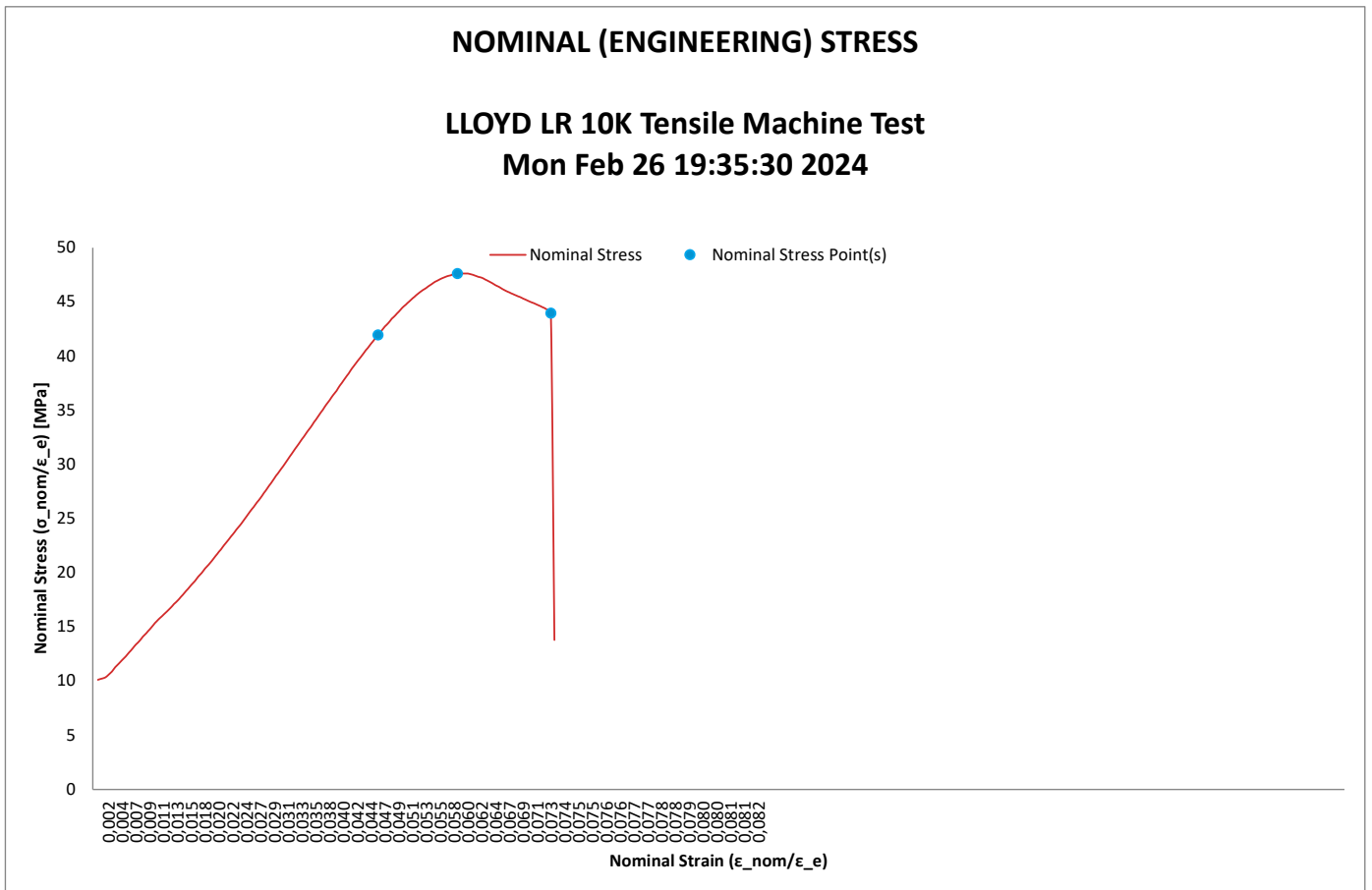
B.6 Results T1-6

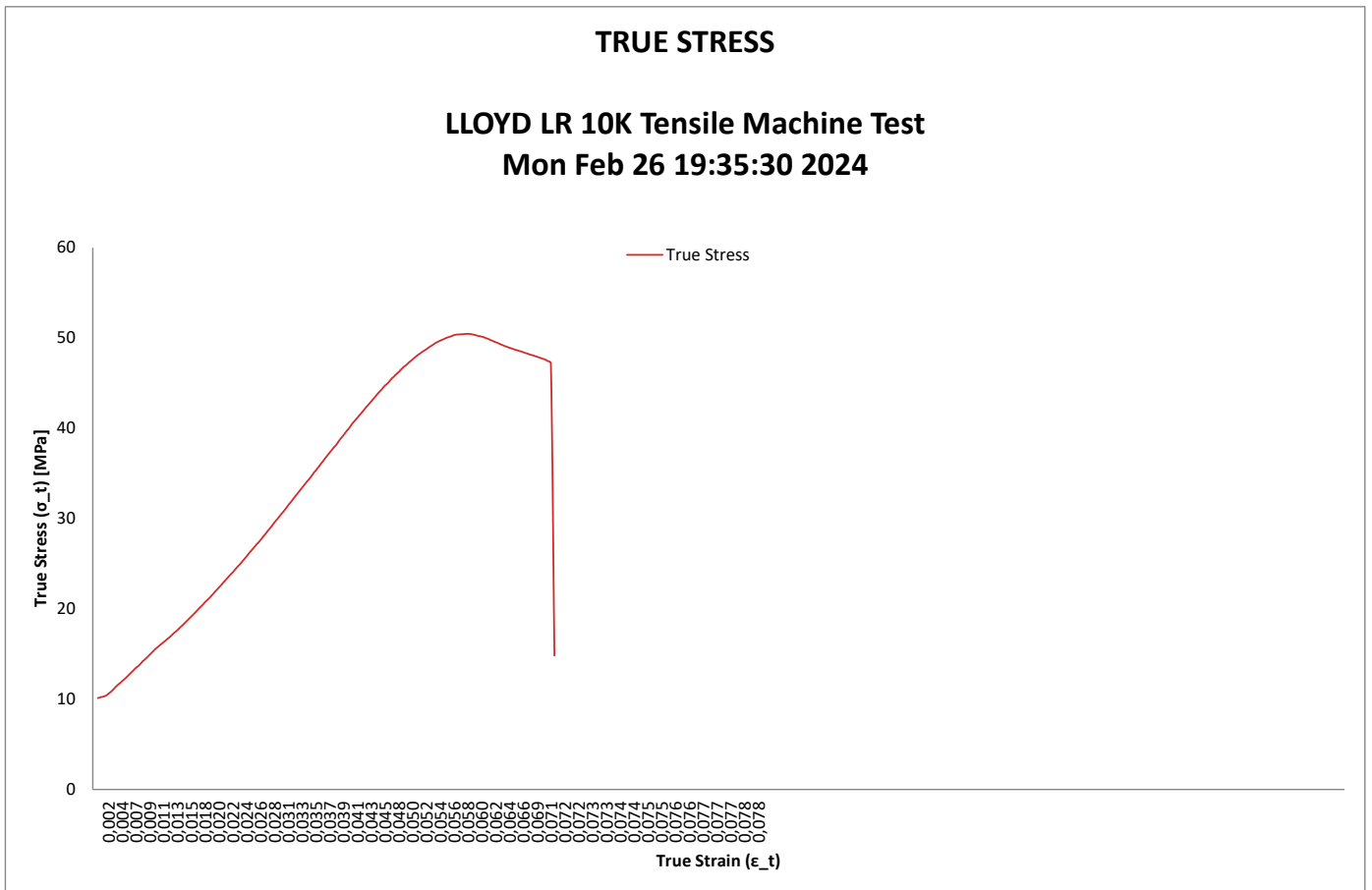
MATERIAL TESTING REPORT		
Mon Feb 26 19:35:30 2024		
<i>Details:</i>	Test piece T1-6	
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	41,944	MPa [N/mm ²]
Nominal Strain, ϵ_n	4,54%	
<u>Young's modulus, E</u>	923,334	MPa [N/mm ²]
<i>True Stress, σ_t</i>	43,849	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	4,44%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
<i>True Stress, σ_t</i>	0,000	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	47,604	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,82%	
<i>True Stress, σ_t</i>	50,373	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	5,66%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,953	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,32%	
<i>True Stress, σ_t</i>	47,170	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	7,06%	

LR 10K Tensile Machine Settings**Mon Feb 26 19:35:30 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





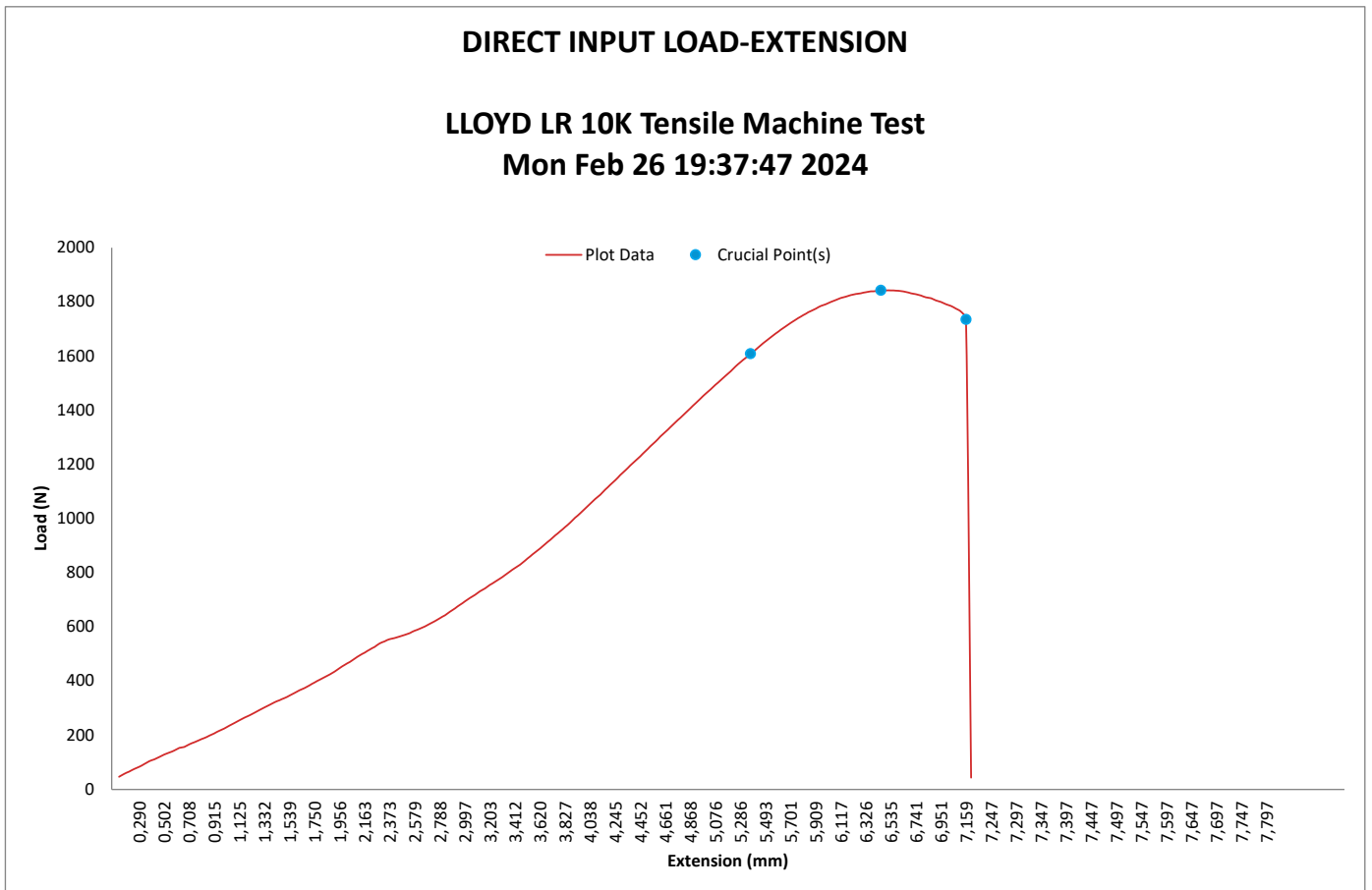


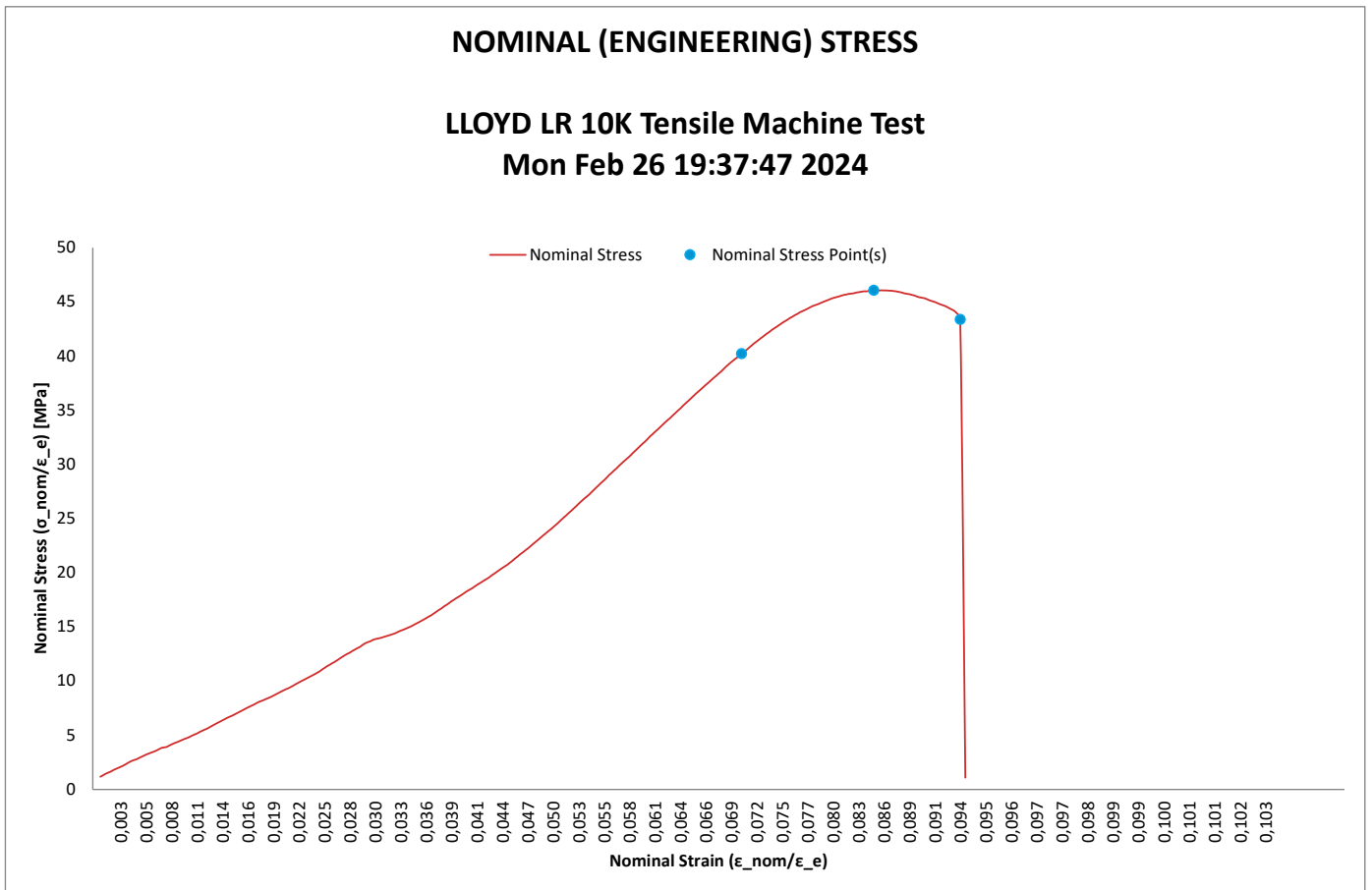
B.7 Results T1-7

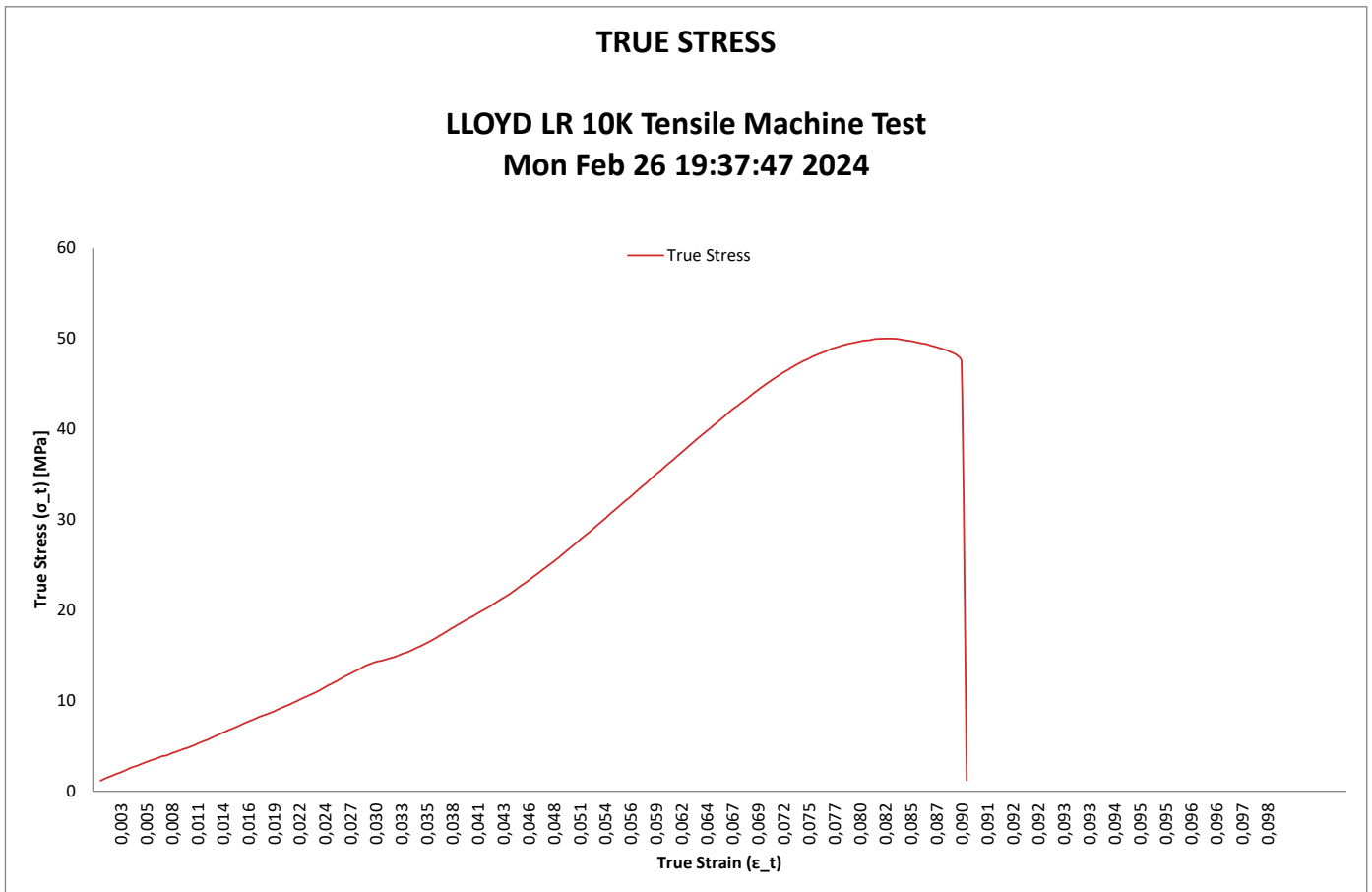
MATERIAL TESTING REPORT		
Mon Feb 26 19:37:47 2024		
Details:	Test piece T1-7	
Material Properties:		
Start Thickness	4,000	mm
Start Width	10,000	mm
Start Length	75,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,205	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,02%	
<u>Young's modulus, E</u>	572,496	MPa [N/mm ²]
True Stress, σ_t	43,028	MPa [N/mm ²]
True Strain, ϵ_t	6,79%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	0,000	MPa [N/mm ²]
Nominal Strain, ϵ_n	0,00%	
True Stress, σ_t	0,000	MPa [N/mm ²]
True Strain, ϵ_t	0,00%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,060	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,47%	
True Stress, σ_t	49,960	MPa [N/mm ²]
True Strain, ϵ_t	8,13%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,365	MPa [N/mm ²]
Nominal Strain, ϵ_n	9,41%	
True Stress, σ_t	47,447	MPa [N/mm ²]
True Strain, ϵ_t	9,00%	

LR 10K Tensile Machine Settings**Mon Feb 26 19:37:47 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%







C Results test T2

C.1 Results T2-1

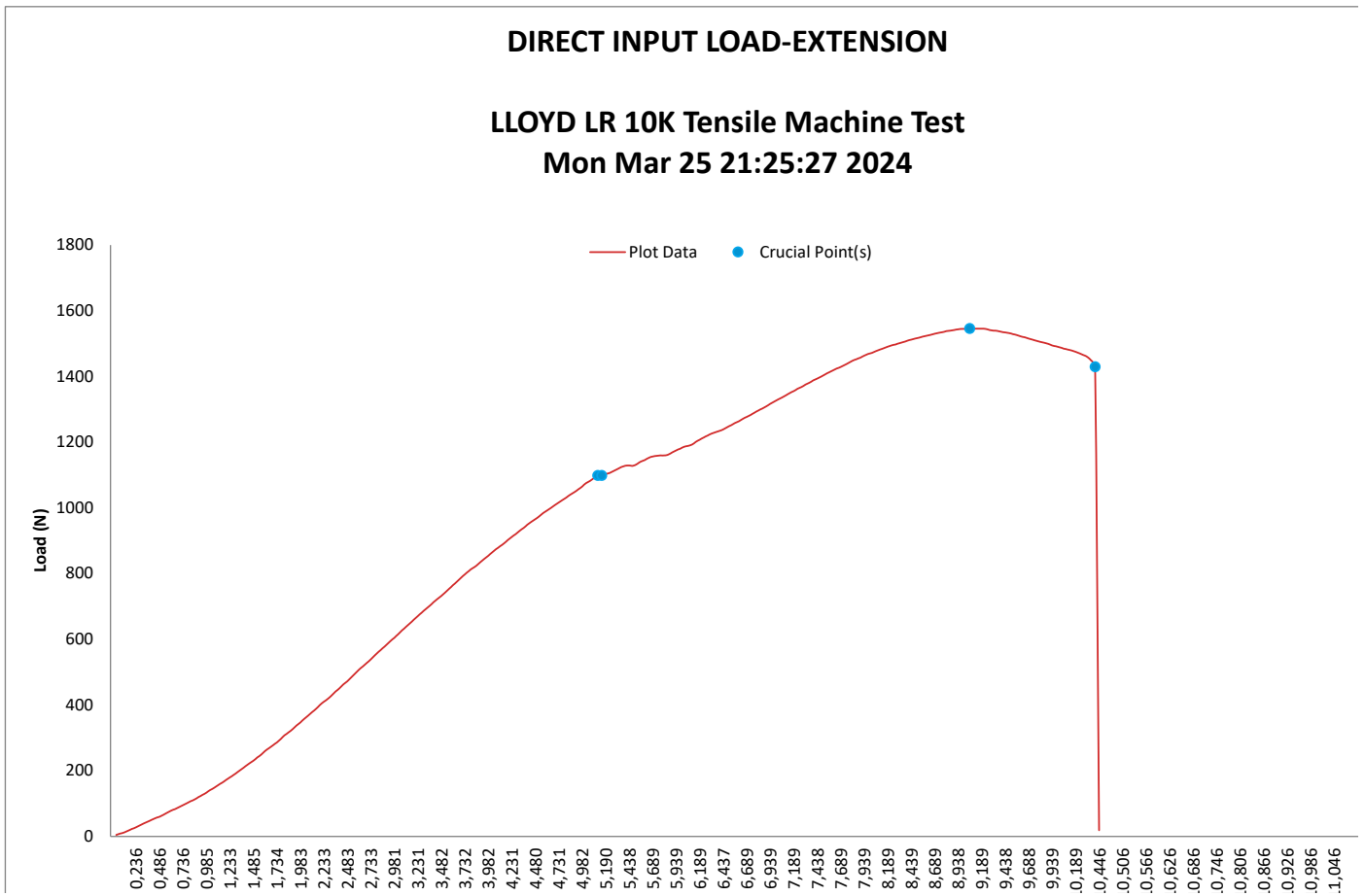
MATERIAL TESTING REPORT		
Mon Mar 25 21:25:27 2024	Test T2-1	
Details:		
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	33,290	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,15%	
<u>Young's modulus, E</u>	646,790	MPa [N/mm ²]
True Stress, σ_t	35,004	MPa [N/mm ²]
True Strain, ϵ_t	5,02%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	33,290	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,15%	
True Stress, σ_t	35,004	MPa [N/mm ²]
True Strain, ϵ_t	5,02%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,832	MPa [N/mm ²]
Nominal Strain, ϵ_n	9,06%	
True Stress, σ_t	51,077	MPa [N/mm ²]
True Strain, ϵ_t	8,68%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,298	MPa [N/mm ²]
Nominal Strain, ϵ_n	10,40%	
True Stress, σ_t	47,800	MPa [N/mm ²]
True Strain, ϵ_t	9,89%	

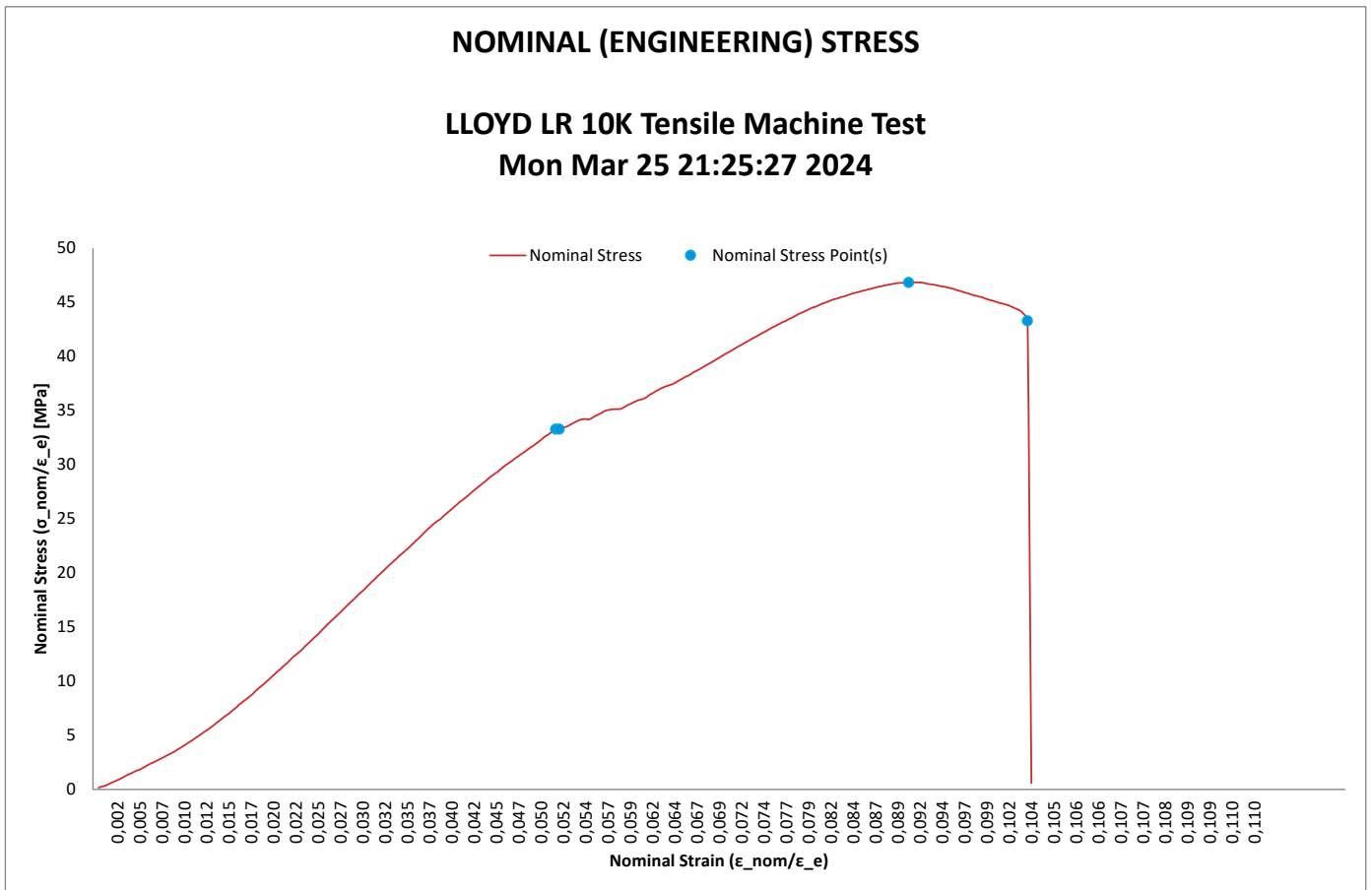
LR 10K Tensile Machine Settings

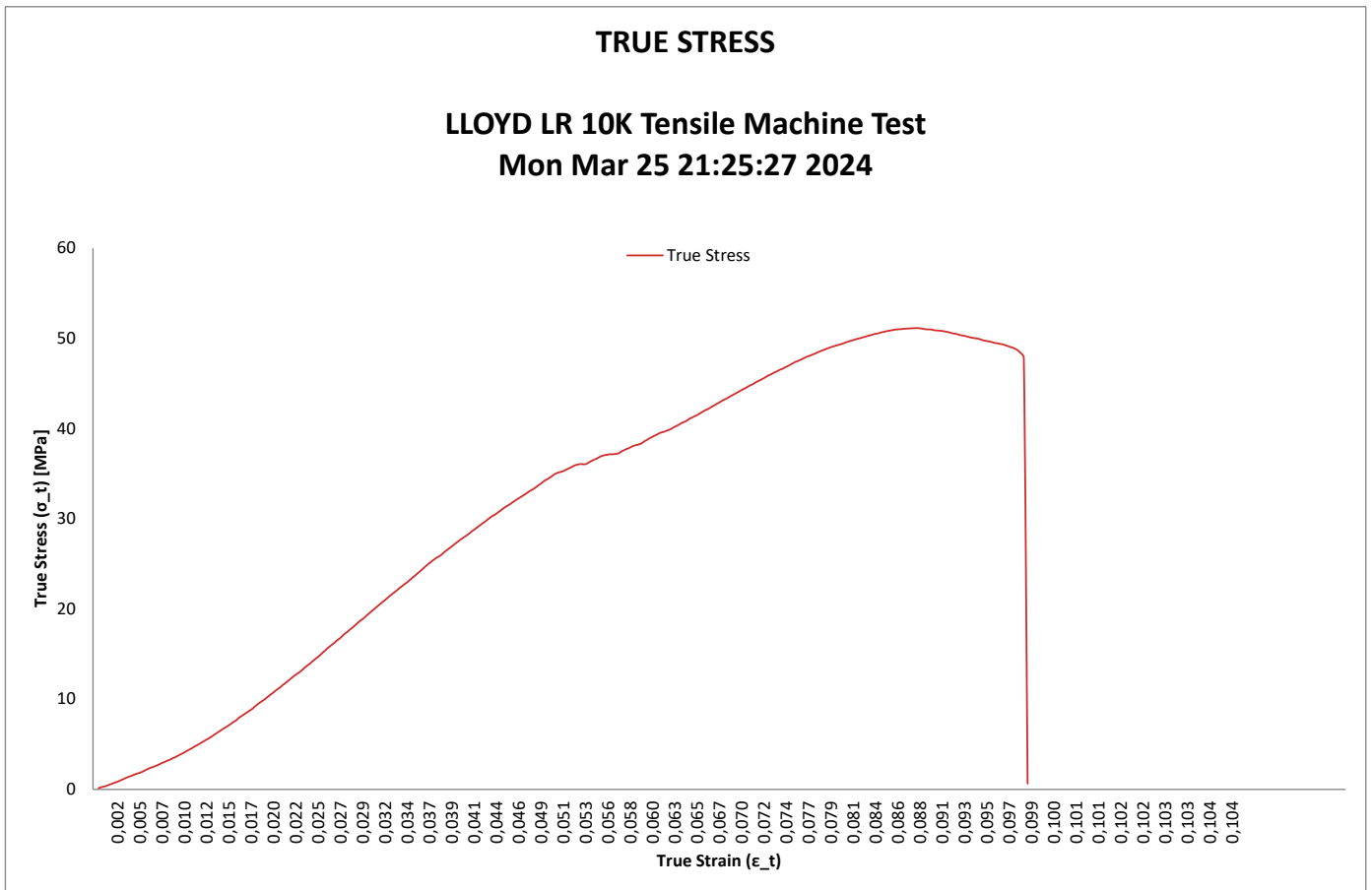
Mon Mar 25 21:25:27 2024

Details: ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%







C.2 Results T2-2

MATERIAL TESTING REPORT

Tue Mar 26 13:47:28 2024

Details:

Test T2-2

Material Properties:

Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm

Proportional limit:

Nominal Stress, σ_{nom}/σ_e (S _y)	31,241	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,26%	
<u>Young's modulus, E</u>	594,052	MPa [N/mm ²]
True Stress, σ_t	32,884	MPa [N/mm ²]
True Strain, ϵ_t	5,13%	

Yielding (lower):

Nominal Stress, σ_{nom}/σ_e (S _y)	31,241	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,26%	
True Stress, σ_t	32,884	MPa [N/mm ²]
True Strain, ϵ_t	5,13%	

Ultimate Tensile Strength:

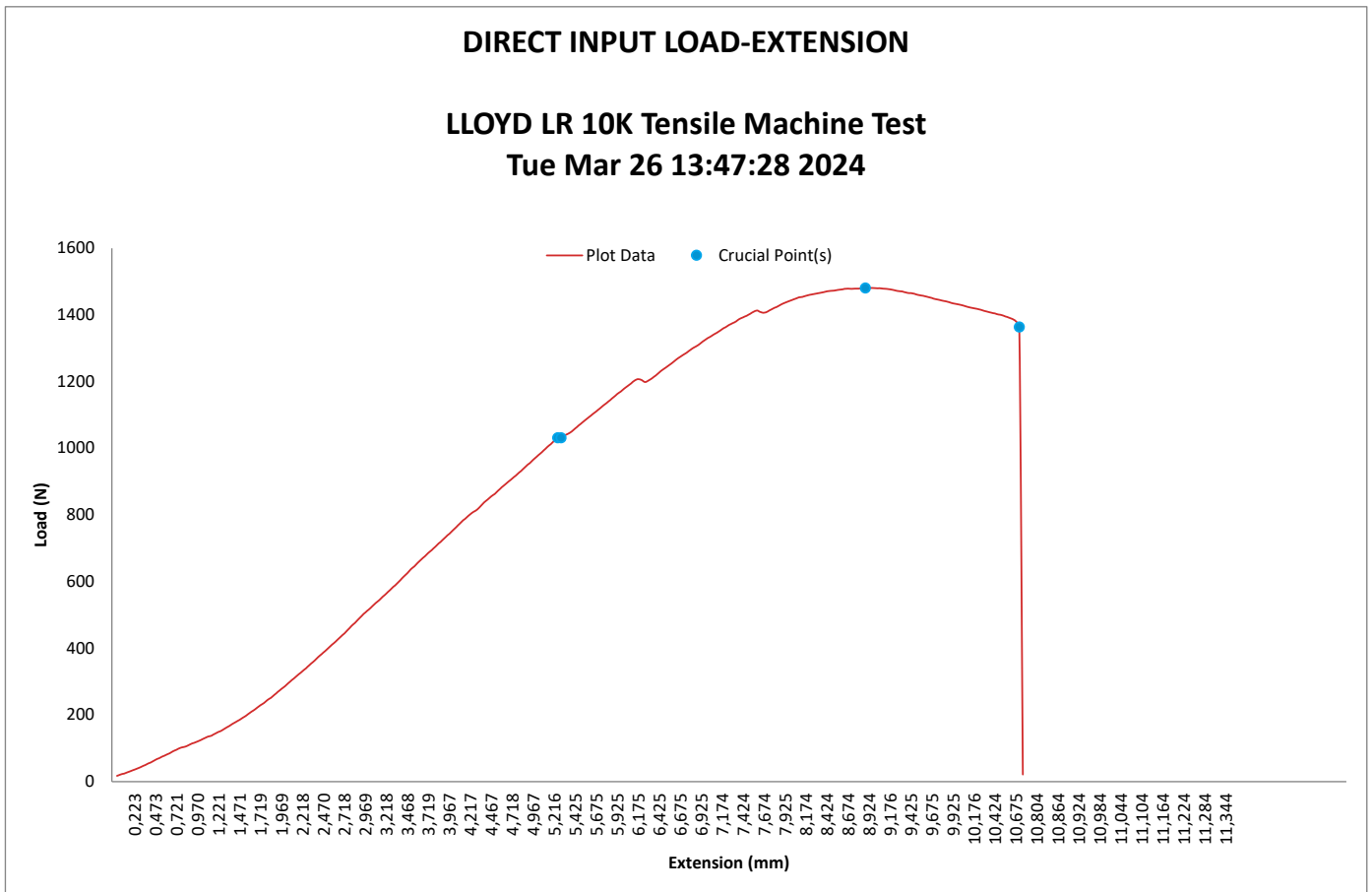
Nominal Stress, σ_{nom}/σ_e (S _{ut})	44,872	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,88%	
True Stress, σ_t	48,858	MPa [N/mm ²]
True Strain, ϵ_t	8,51%	

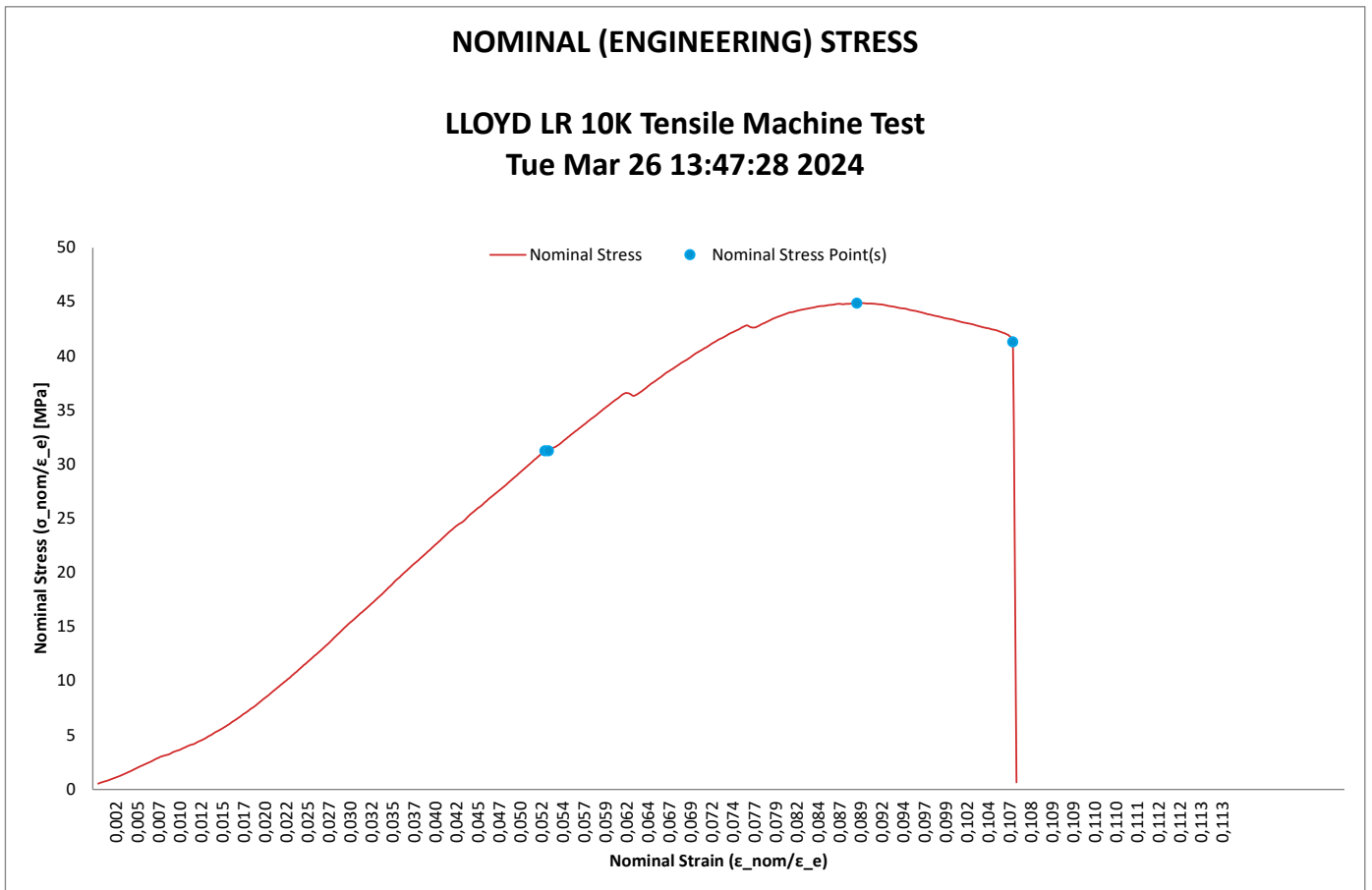
Fracture:

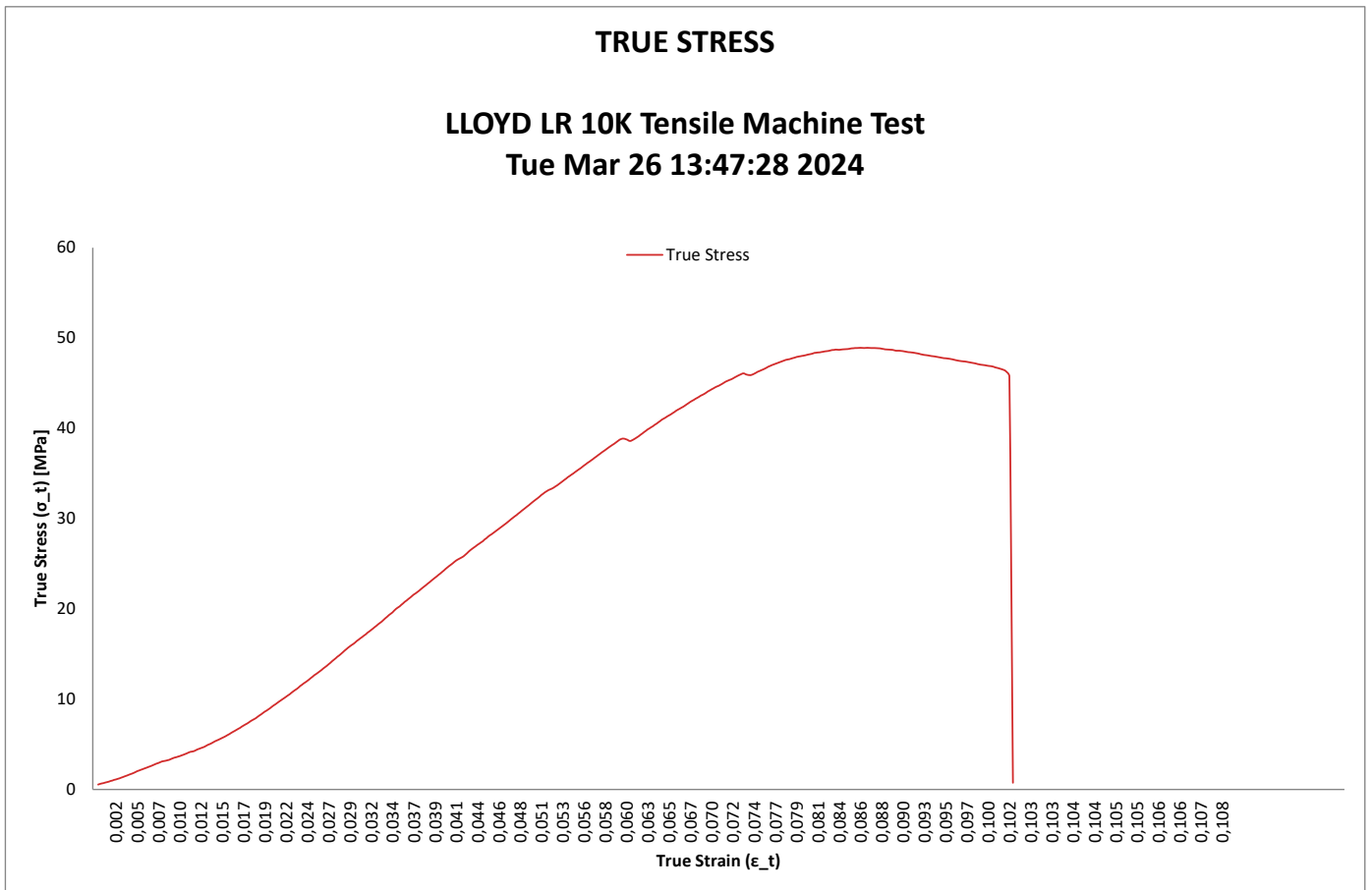
Nominal Stress, σ_{nom}/σ_e (S _y)	41,308	MPa [N/mm ²]
Nominal Strain, ϵ_n	10,72%	
True Stress, σ_t	45,736	MPa [N/mm ²]
True Strain, ϵ_t	10,18%	

LR 10K Tensile Machine Settings**Tue Mar 26 13:47:28 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





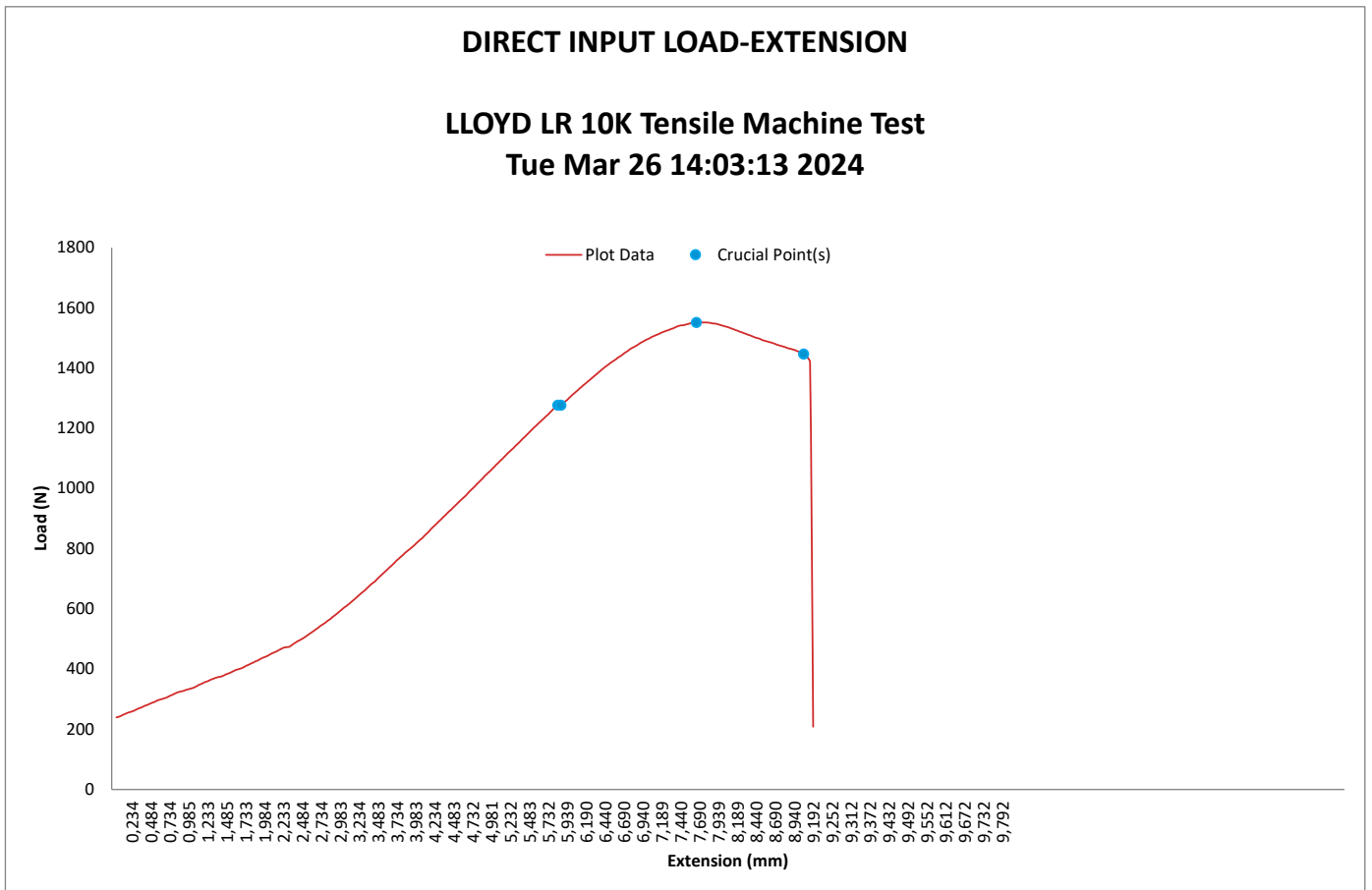


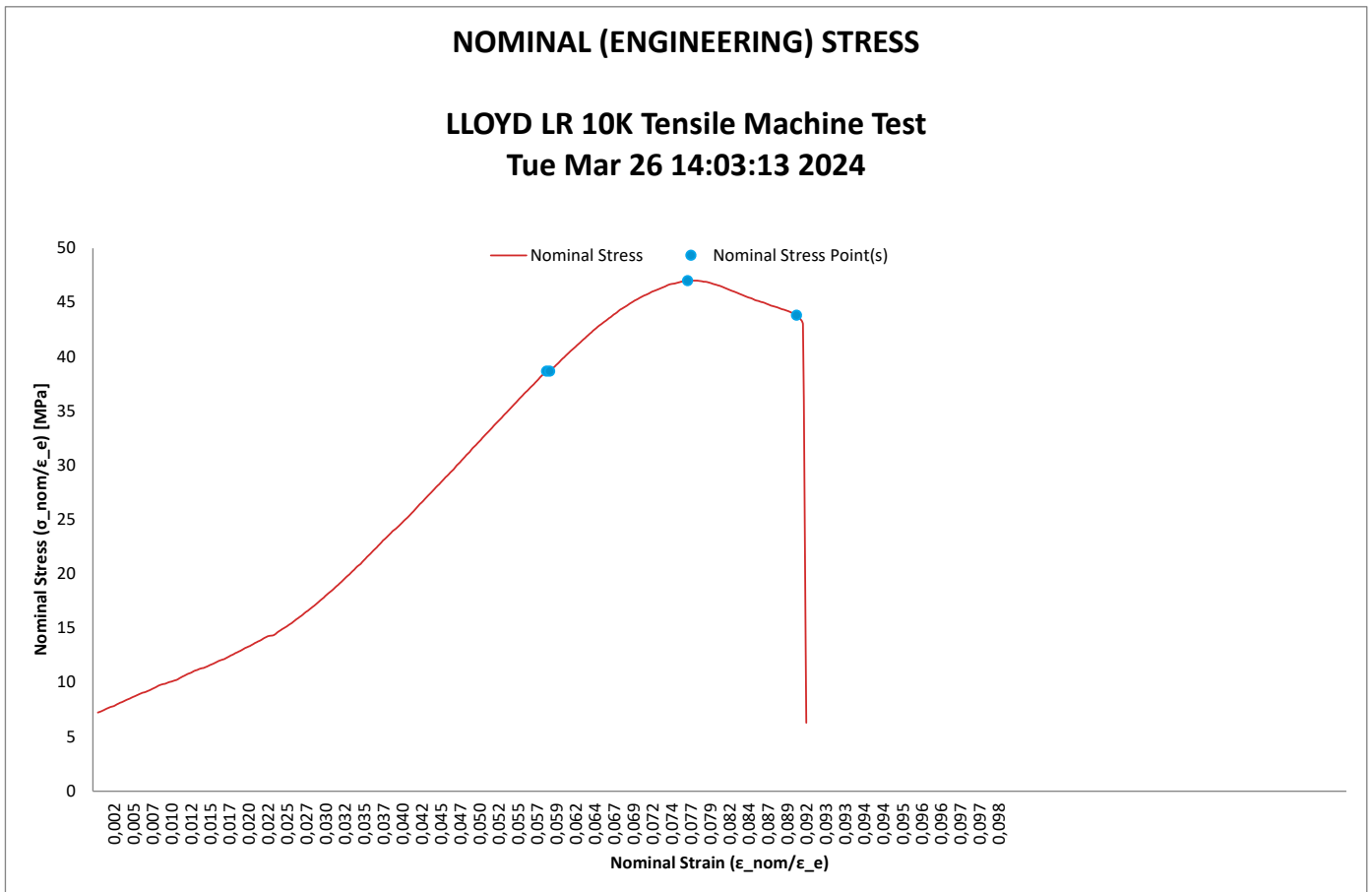
C.3 Results T2-3

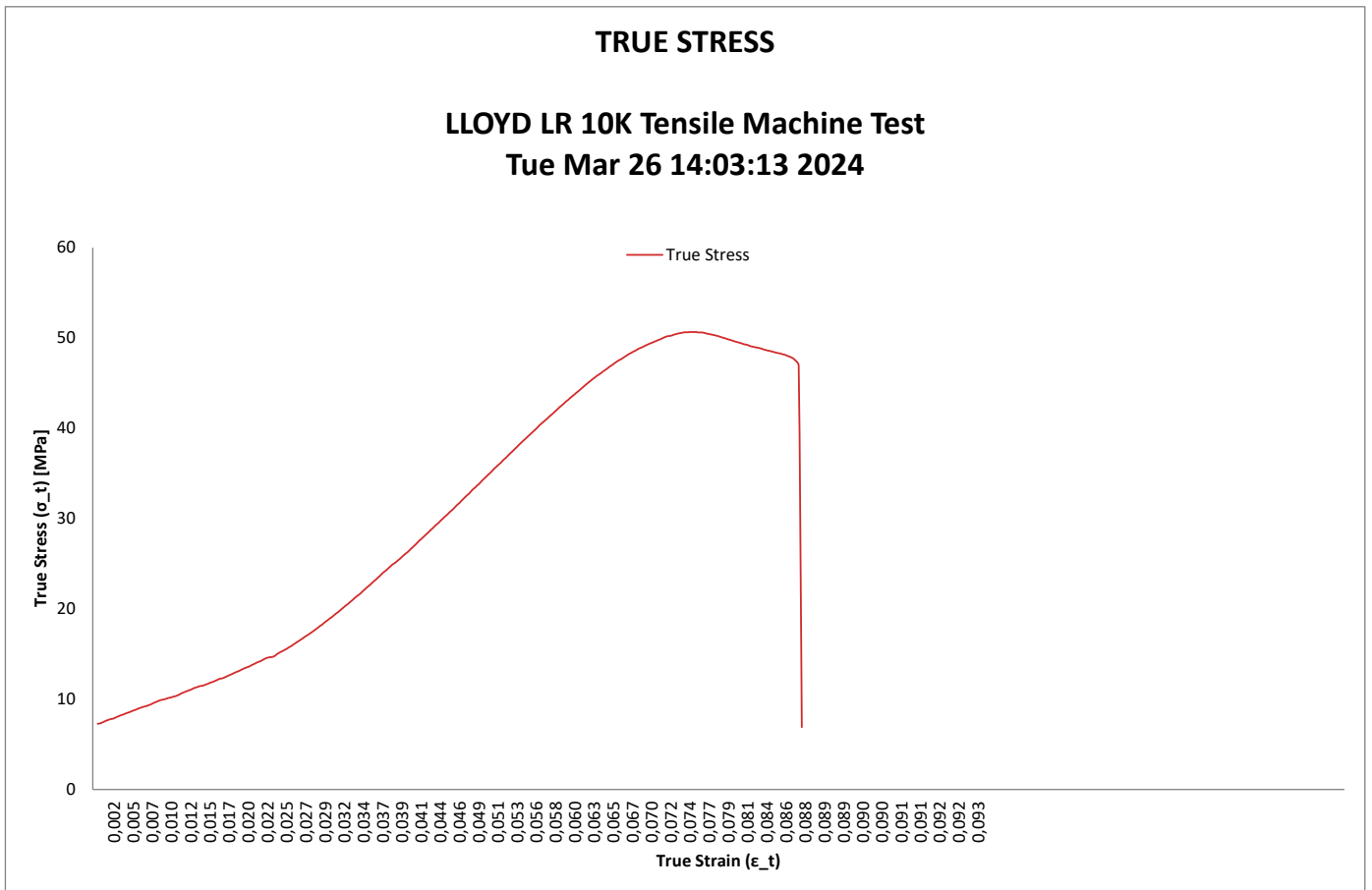
MATERIAL TESTING REPORT		
Tue Mar 26 14:03:13 2024		
Details:	Test T2-3	
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	38,665	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,86%	
<u>Young's modulus, E</u>	660,158	MPa [N/mm ²]
True Stress, σ_t	40,930	MPa [N/mm ²]
True Strain, ϵ_t	5,69%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	38,665	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,86%	
True Stress, σ_t	40,930	MPa [N/mm ²]
True Strain, ϵ_t	5,69%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	47,010	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,65%	
True Stress, σ_t	50,606	MPa [N/mm ²]
True Strain, ϵ_t	7,37%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	43,833	MPa [N/mm ²]
Nominal Strain, ϵ_n	9,06%	
True Stress, σ_t	47,806	MPa [N/mm ²]
True Strain, ϵ_t	8,68%	

LR 10K Tensile Machine Settings**Tue Mar 26 14:03:13 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





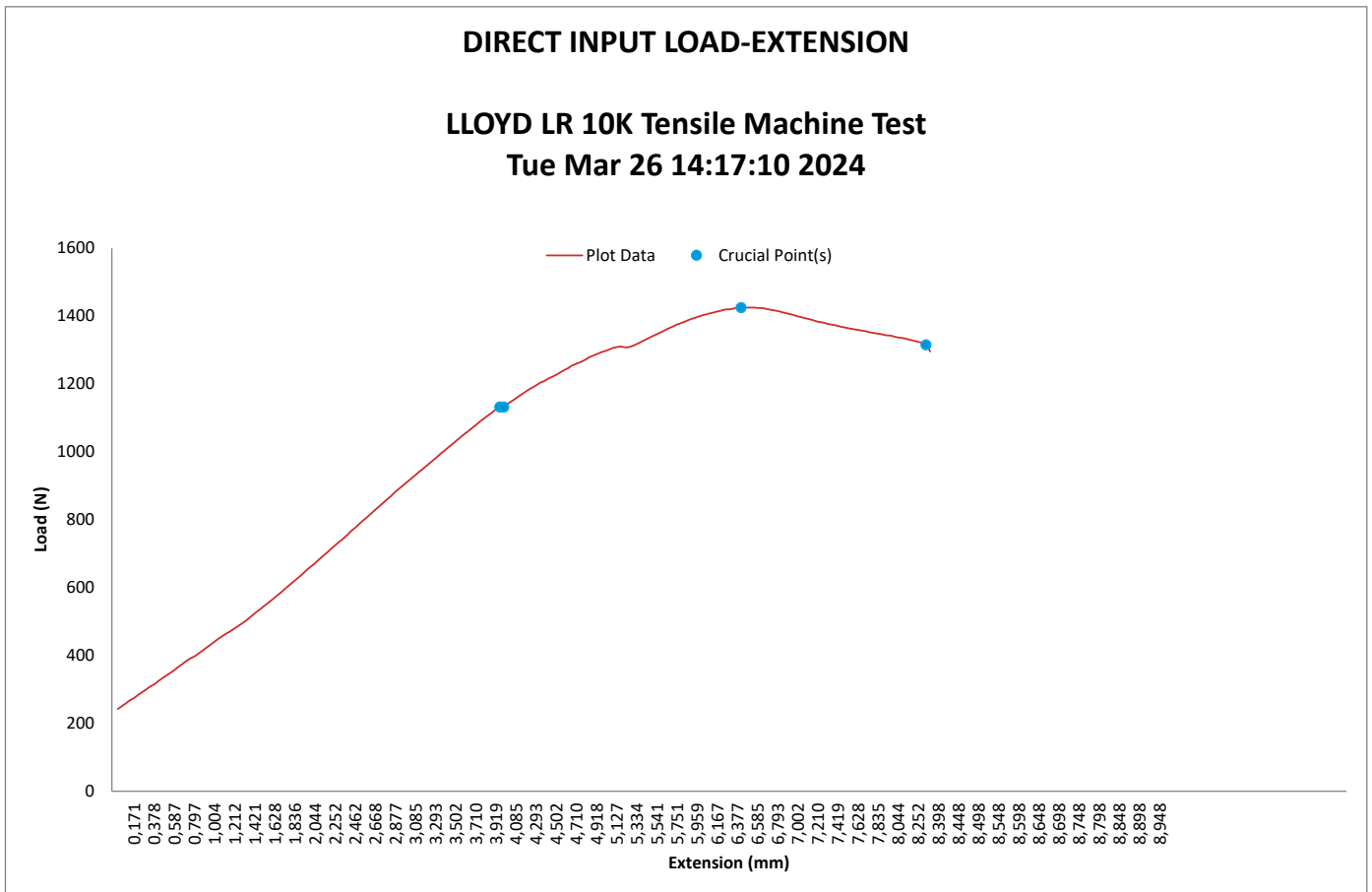


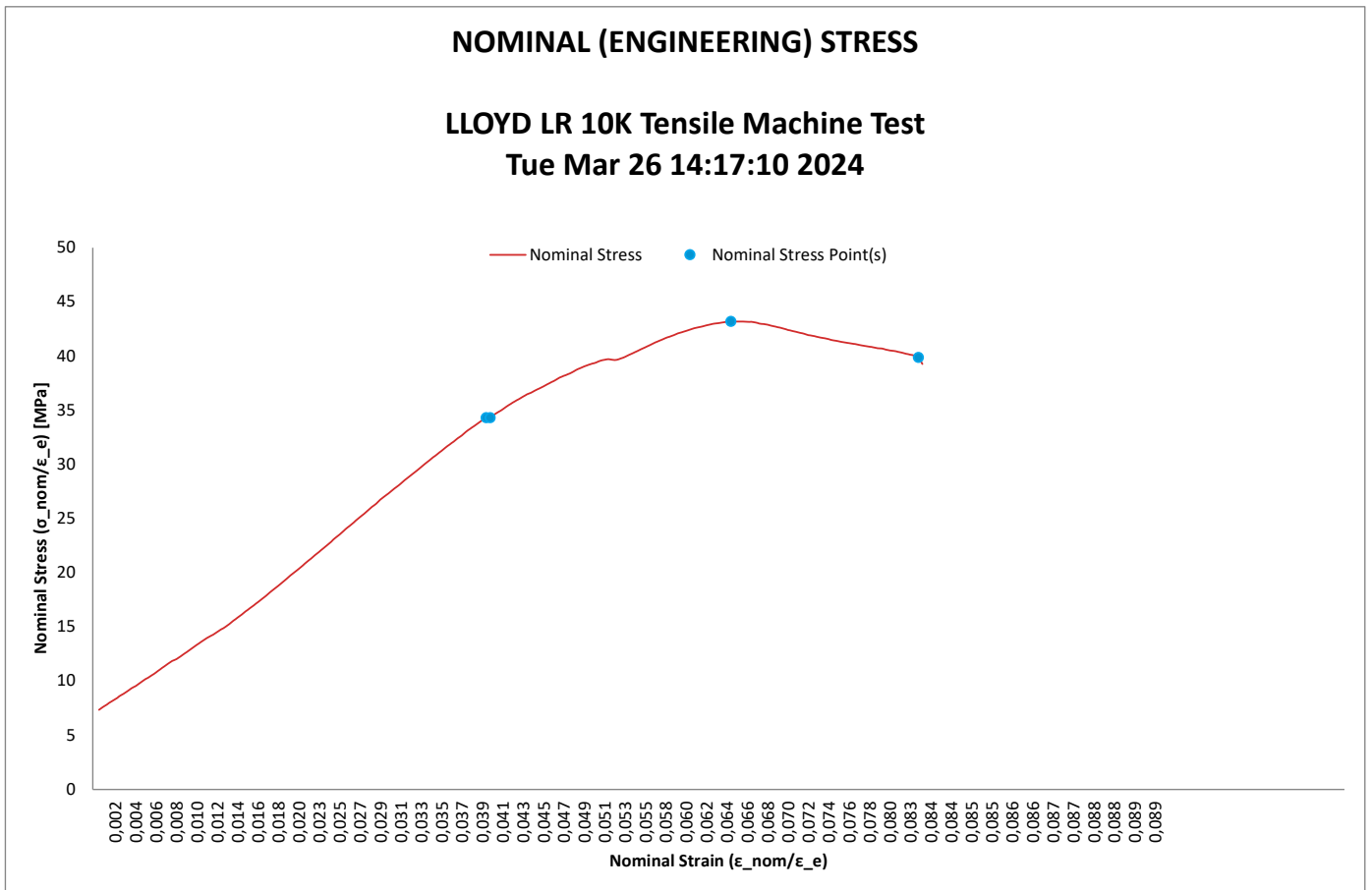
C.4 Results T2-4

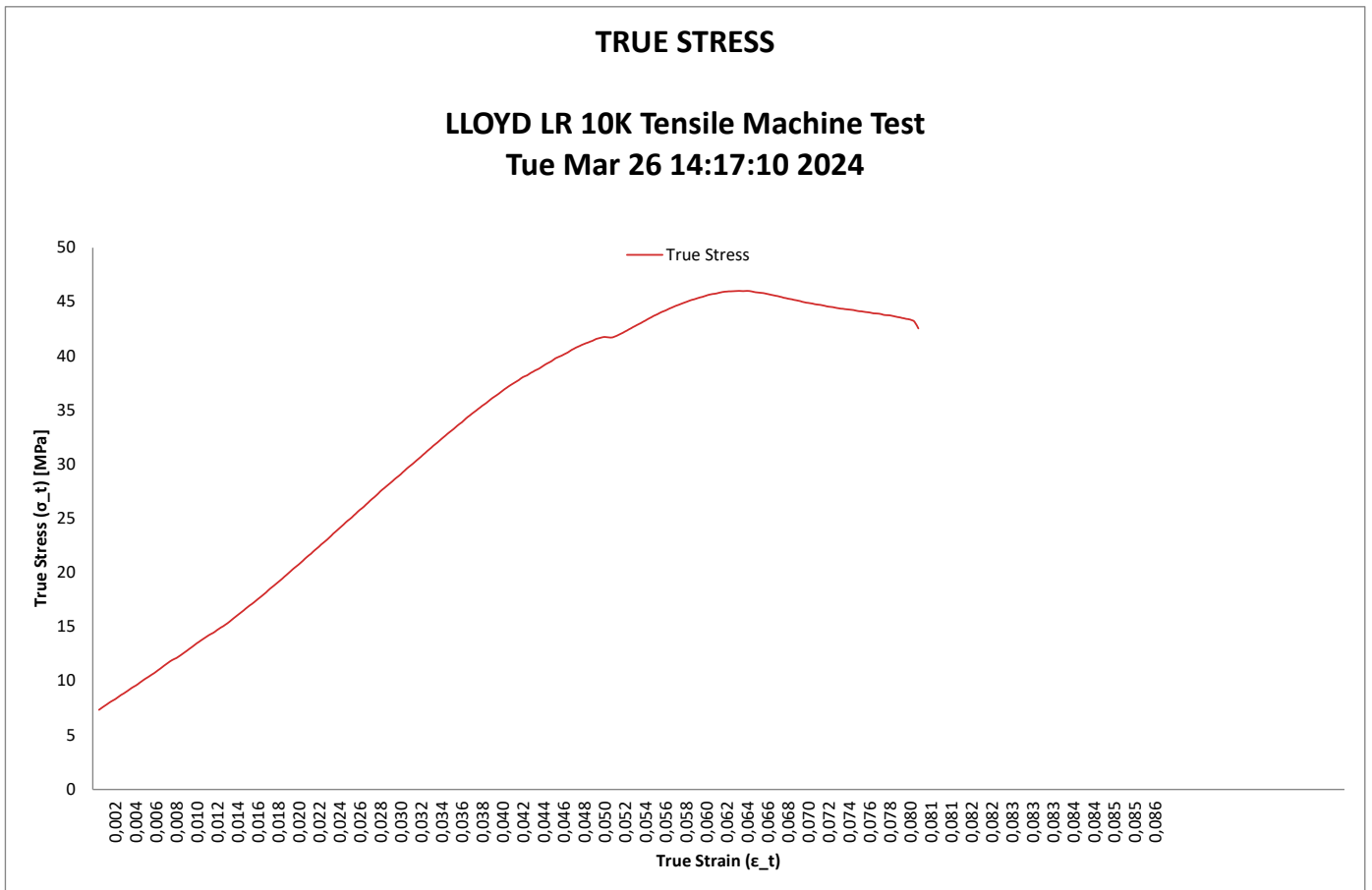
MATERIAL TESTING REPORT		
Tue Mar 26 14:17:10 2024		
Details:	Test T2-4	
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	34,300	MPa [N/mm ²]
Nominal Strain, ϵ_n	3,96%	
<u>Young's modulus, E</u>	866,162	MPa [N/mm ²]
True Stress, σ_t	35,658	MPa [N/mm ²]
True Strain, ϵ_t	3,88%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	34,300	MPa [N/mm ²]
Nominal Strain, ϵ_n	3,96%	
True Stress, σ_t	35,658	MPa [N/mm ²]
True Strain, ϵ_t	3,88%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	43,179	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,42%	
True Stress, σ_t	45,951	MPa [N/mm ²]
True Strain, ϵ_t	6,22%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	39,853	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,34%	
True Stress, σ_t	43,176	MPa [N/mm ²]
True Strain, ϵ_t	8,01%	

LR 10K Tensile Machine Settings**Tue Mar 26 14:17:10 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%







C.5 Results T2-5

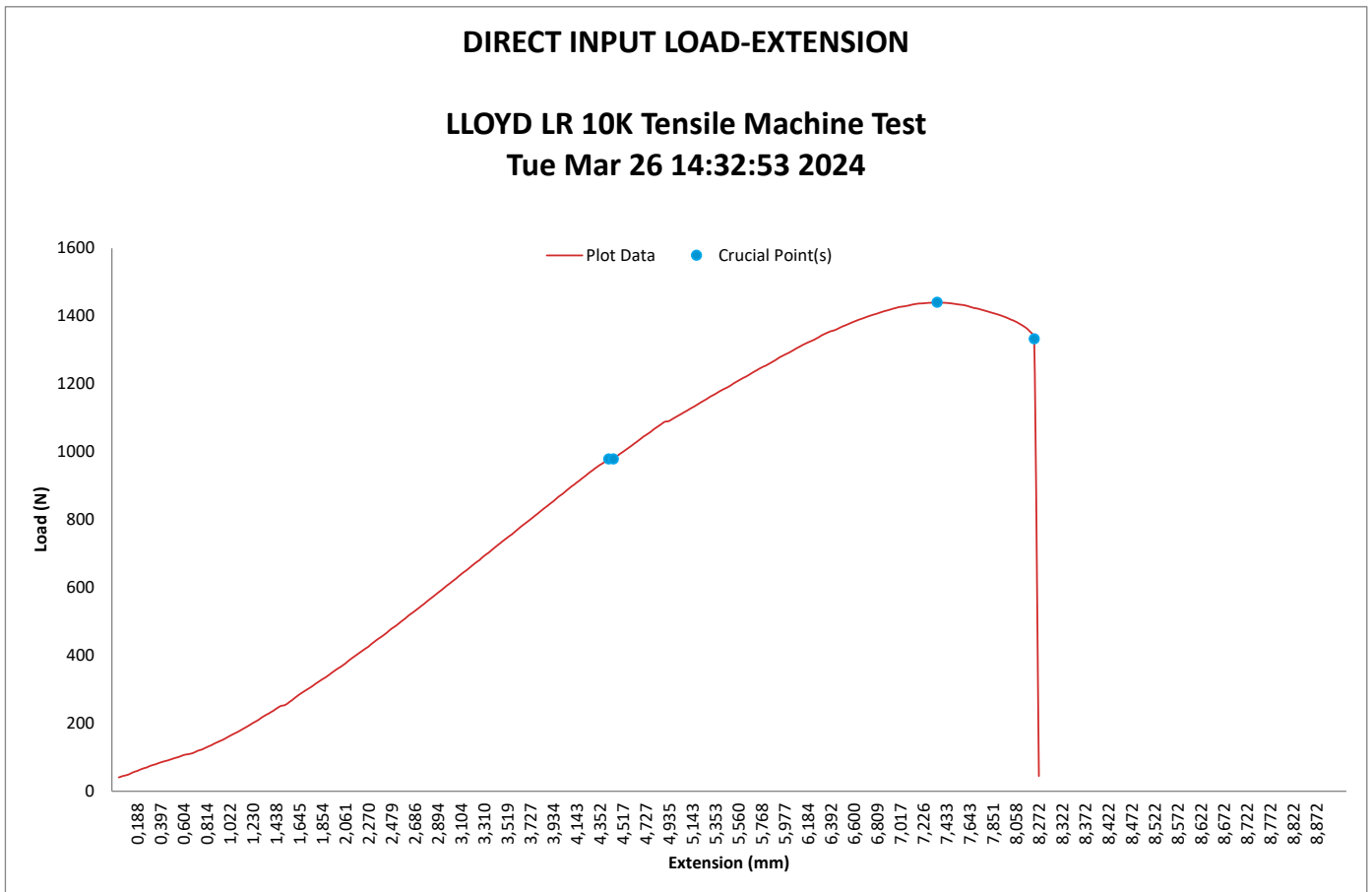
MATERIAL TESTING REPORT		
Tue Mar 26 14:32:53 2024		
<i>Details:</i>	T2-5	
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	29,667	MPa [N/mm ²]
Nominal Strain, ϵ_n	4,43%	
<u>Young's modulus, E</u>	668,935	MPa [N/mm ²]
<i>True Stress, σ_t</i>	30,983	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	4,34%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	29,667	MPa [N/mm ²]
Nominal Strain, ϵ_n	4,43%	
<i>True Stress, σ_t</i>	30,983	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	4,34%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	43,655	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,35%	
<i>True Stress, σ_t</i>	46,864	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	7,09%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,388	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,23%	
<i>True Stress, σ_t</i>	43,711	MPa [N/mm ²]
<i>True Strain, ϵ_t</i>	7,91%	

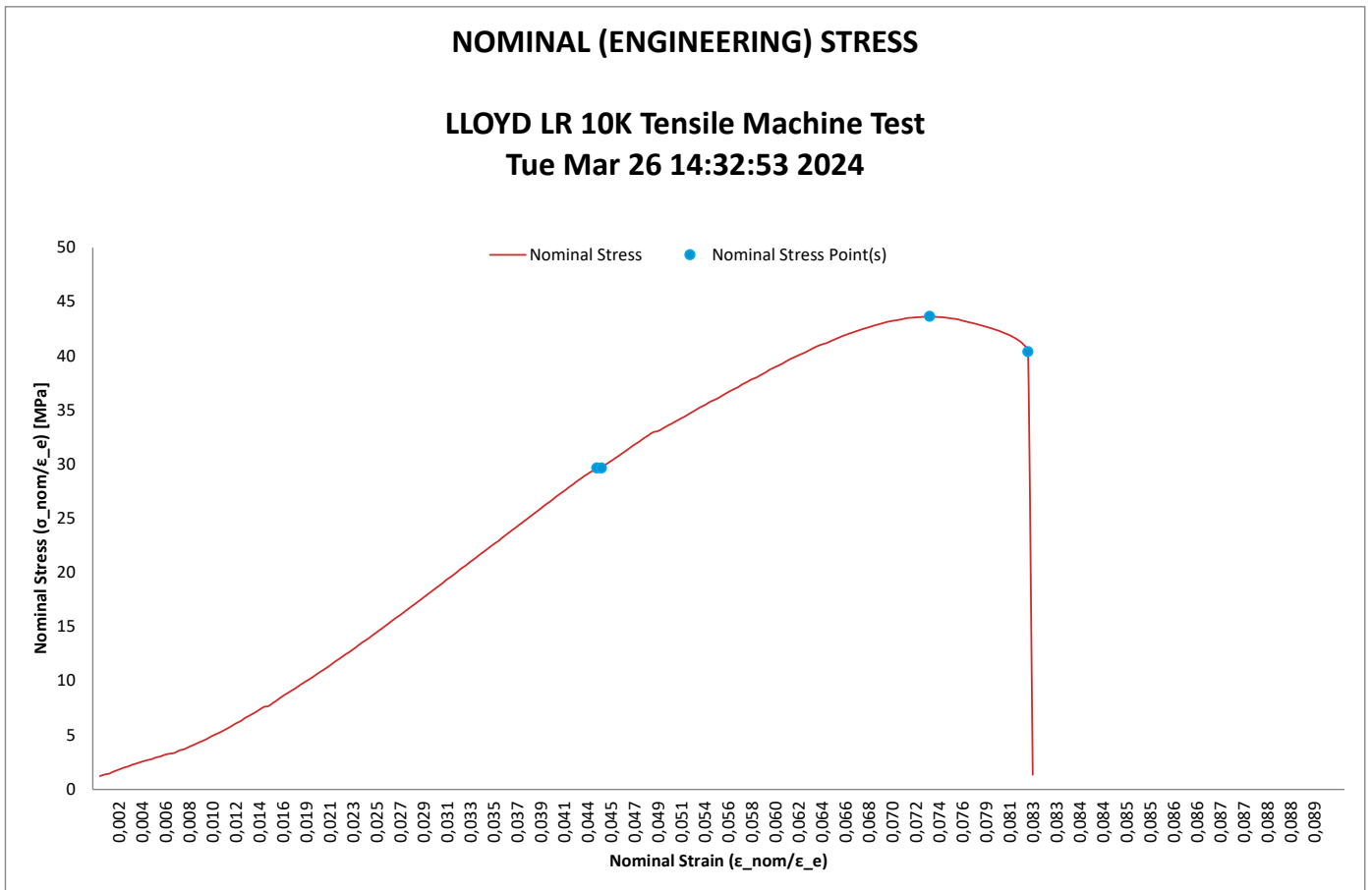
LR 10K Tensile Machine Settings

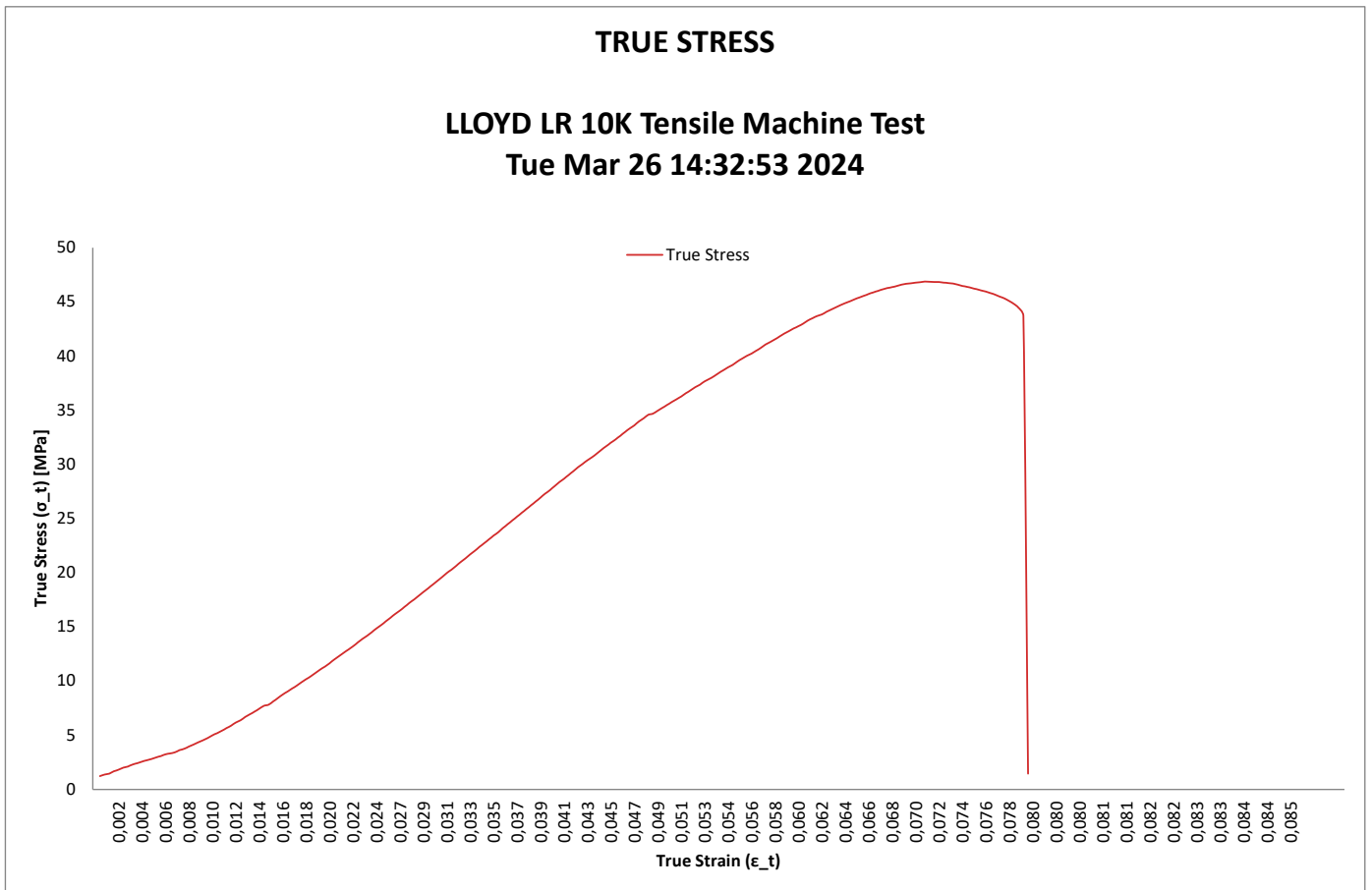
Tue Mar 26 14:32:53 2024

Details: ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





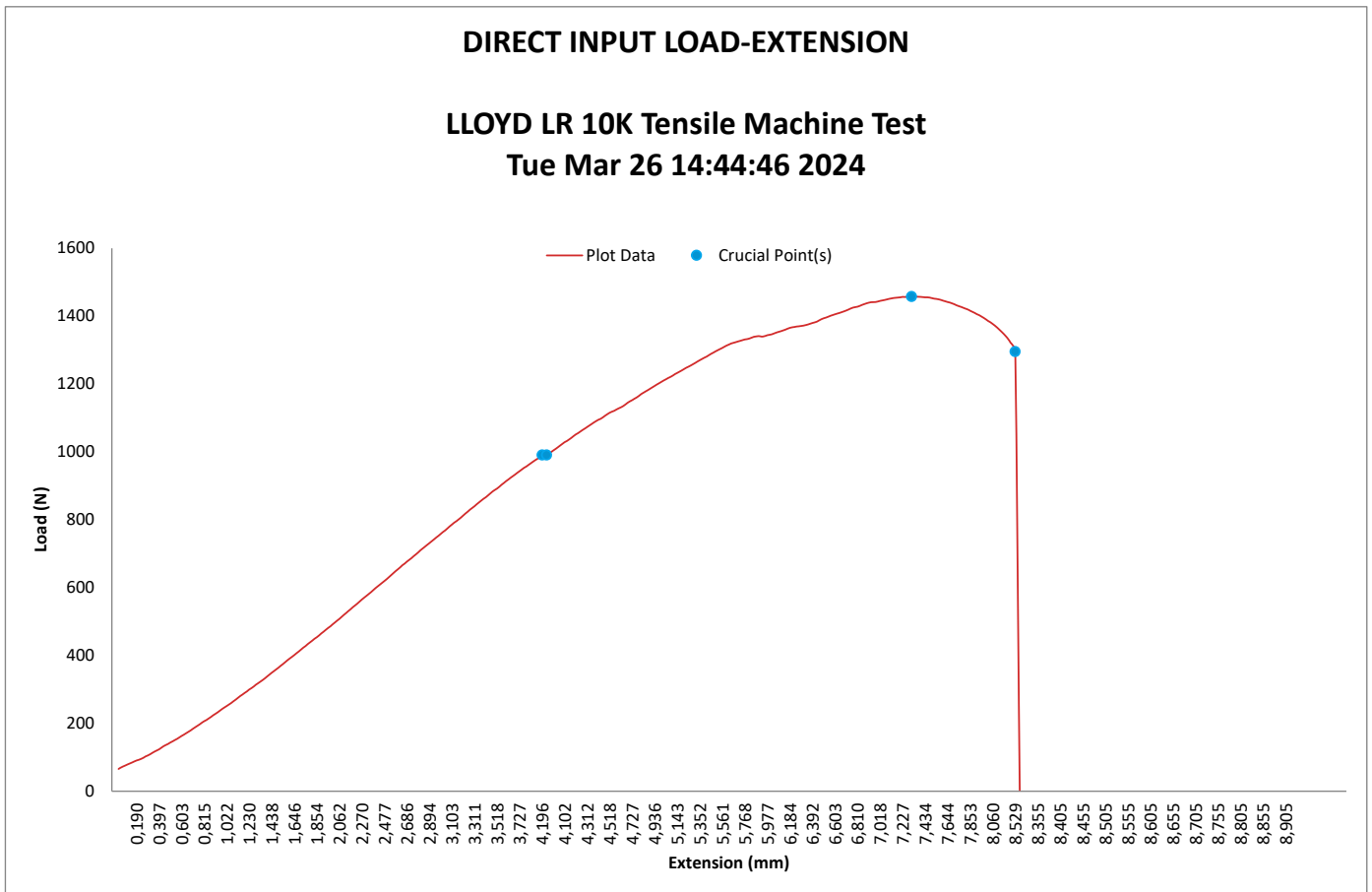


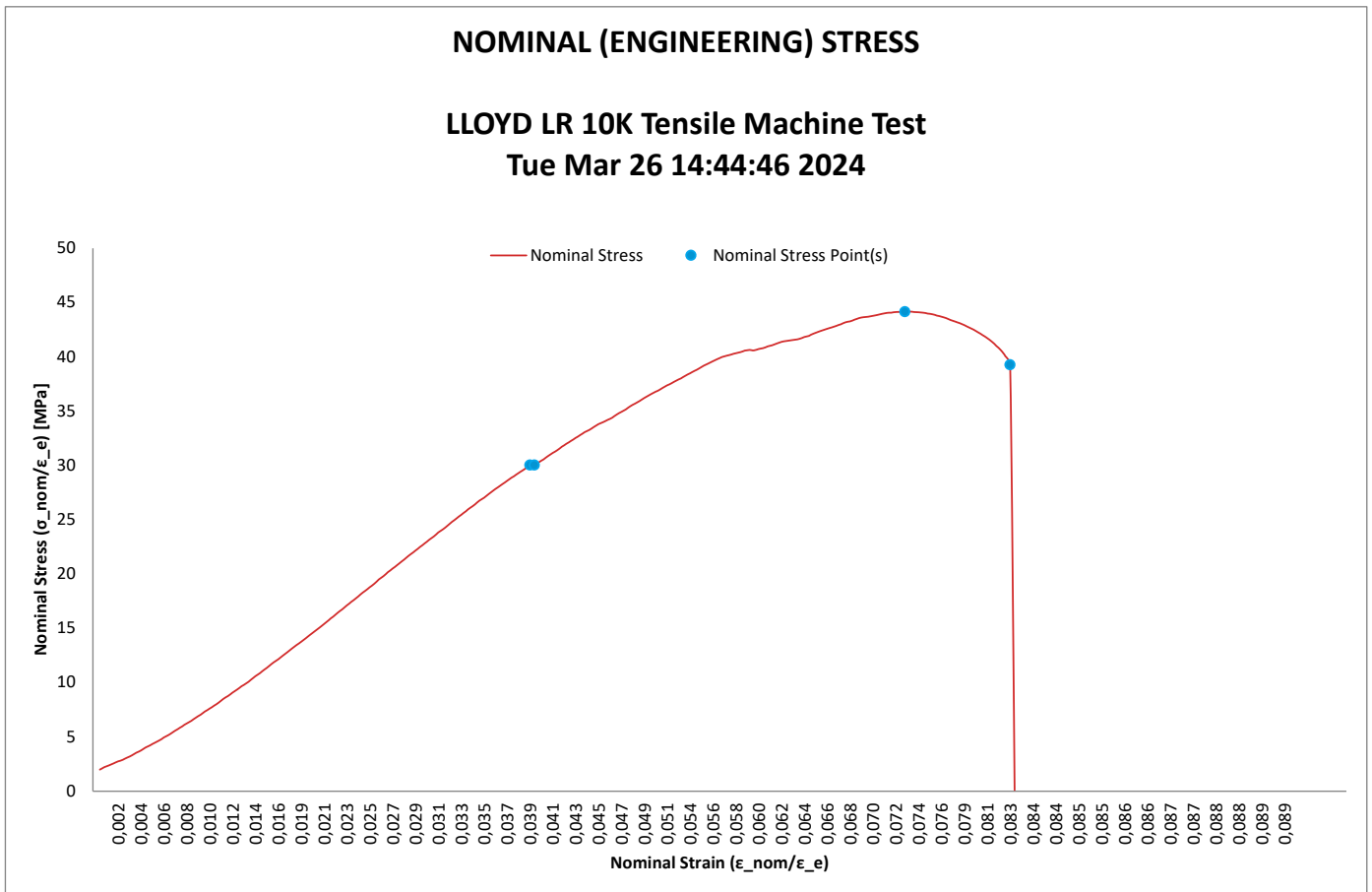
C.6 Results T2-6

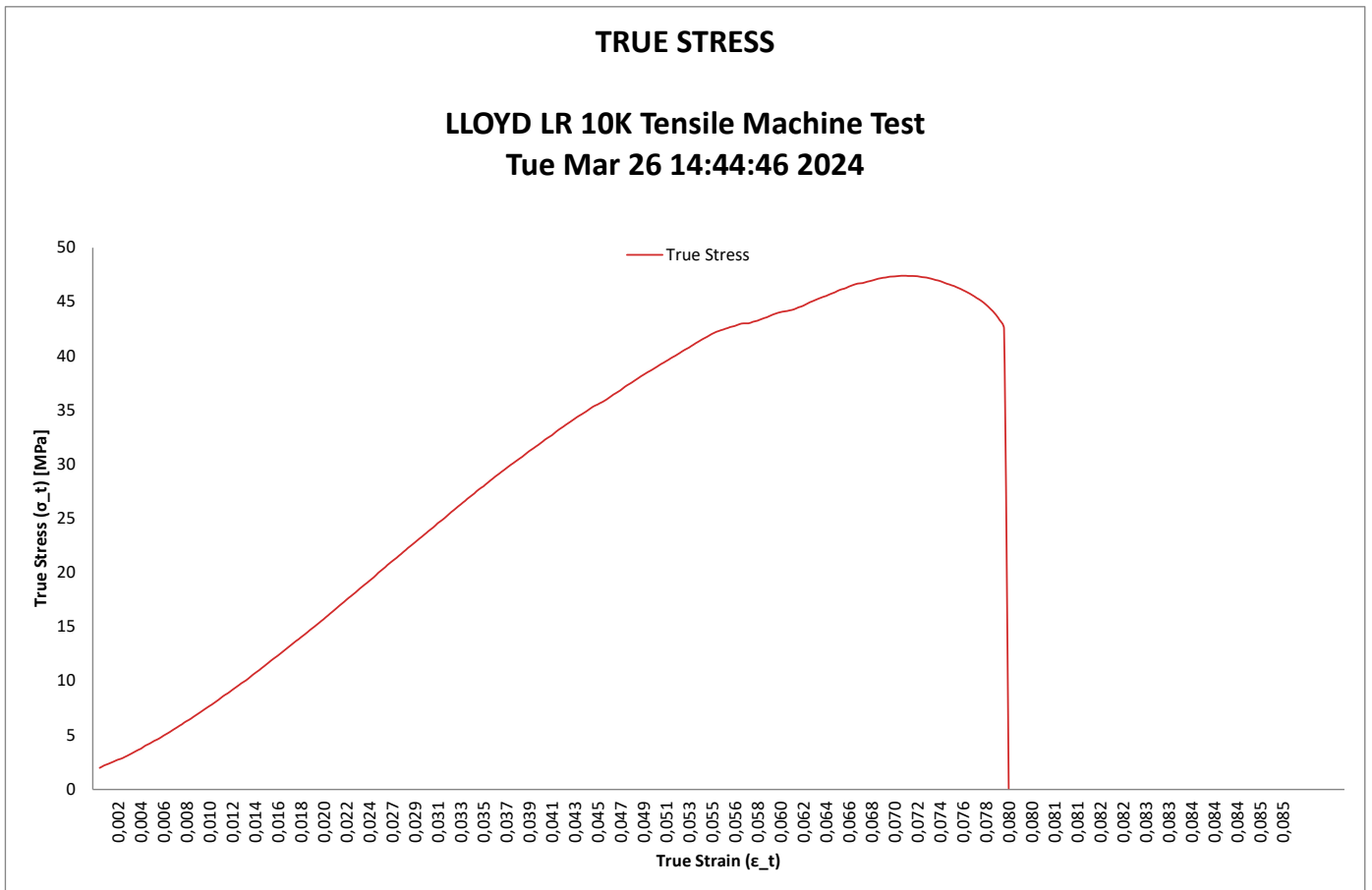
<p>Tue Mar 26 14:44:46 2024 <i>Details:</i></p> <p>Material Properties:</p> <p>Start Thickness Start Width Start Length</p> <p>Proportional limit:</p> <p>Nominal Stress, σ_{nom}/σ_e (S_y) Nominal Strain, ϵ_n <u>Young's modulus, E</u> True Stress, σ_t True Strain, ϵ_t</p> <p>Yielding (lower):</p> <p>Nominal Stress, σ_{nom}/σ_e (S_y) Nominal Strain, ϵ_n True Stress, σ_t True Strain, ϵ_t</p> <p>Ultimate Tensile Strength:</p> <p>Nominal Stress, σ_{nom}/σ_e (S_{ut}) Nominal Strain, ϵ_n True Stress, σ_t True Strain, ϵ_t</p> <p>Fracture:</p> <p>Nominal Stress, σ_{nom}/σ_e (S_y) Nominal Strain, ϵ_n True Stress, σ_t True Strain, ϵ_t</p>	<p style="text-align: center;">MATERIAL TESTING REPORT</p> <div style="border: 2px solid orange; padding: 10px; margin: 10px 0; text-align: center;"> Test T2-6 </div> <table border="0"> <tr> <td style="padding-right: 10px;">3,000</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>mm</td> </tr> <tr> <td>11,000</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>mm</td> </tr> <tr> <td>100,000</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>mm</td> </tr> </table> <table border="0"> <tr> <td style="padding-right: 10px;">30,024</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>3,94%</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td></td> </tr> <tr> <td>762,796</td> <td style="background-color: #c8e6c9; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>31,205</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>3,86%</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td></td> </tr> </table> <table border="0"> <tr> <td style="padding-right: 10px;">30,024</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>3,94%</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td></td> </tr> <tr> <td>31,205</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>3,86%</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td></td> </tr> </table> <table border="0"> <tr> <td style="padding-right: 10px;">44,159</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>7,31%</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td></td> </tr> <tr> <td>47,387</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>7,06%</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td></td> </tr> </table> <table border="0"> <tr> <td style="padding-right: 10px;">39,259</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>8,27%</td> <td style="background-color: #ffffcc; padding: 2px 10px;"></td> <td></td> </tr> <tr> <td>42,506</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td>MPa [N/mm²]</td> </tr> <tr> <td>7,94%</td> <td style="background-color: #bbdefb; padding: 2px 10px;"></td> <td></td> </tr> </table>	3,000		mm	11,000		mm	100,000		mm	30,024		MPa [N/mm ²]	3,94%			762,796		MPa [N/mm ²]	31,205		MPa [N/mm ²]	3,86%			30,024		MPa [N/mm ²]	3,94%			31,205		MPa [N/mm ²]	3,86%			44,159		MPa [N/mm ²]	7,31%			47,387		MPa [N/mm ²]	7,06%			39,259		MPa [N/mm ²]	8,27%			42,506		MPa [N/mm ²]	7,94%		
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42,506		MPa [N/mm ²]																																																											
7,94%																																																													

LR 10K Tensile Machine Settings**Tue Mar 26 14:44:46 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





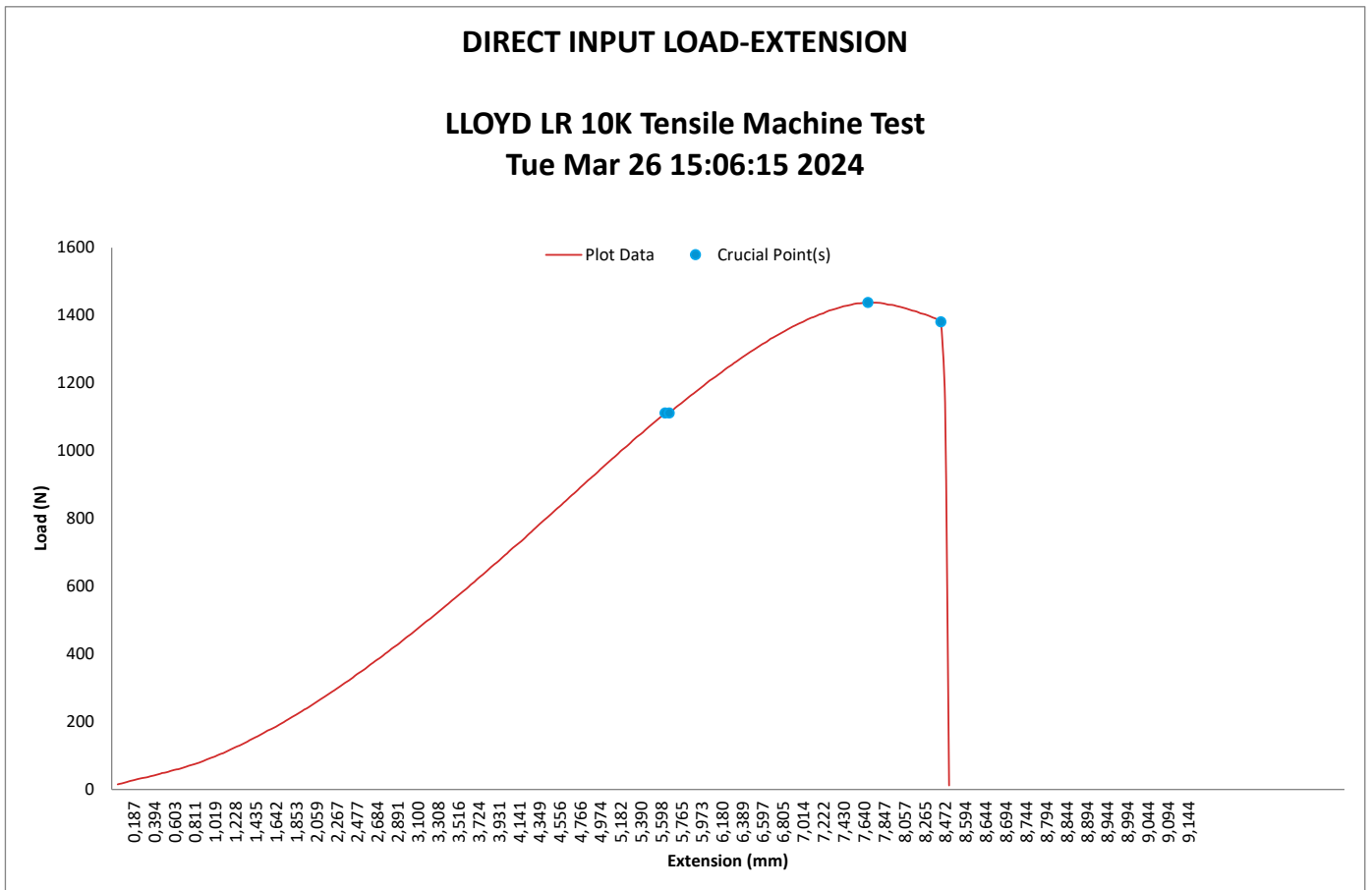


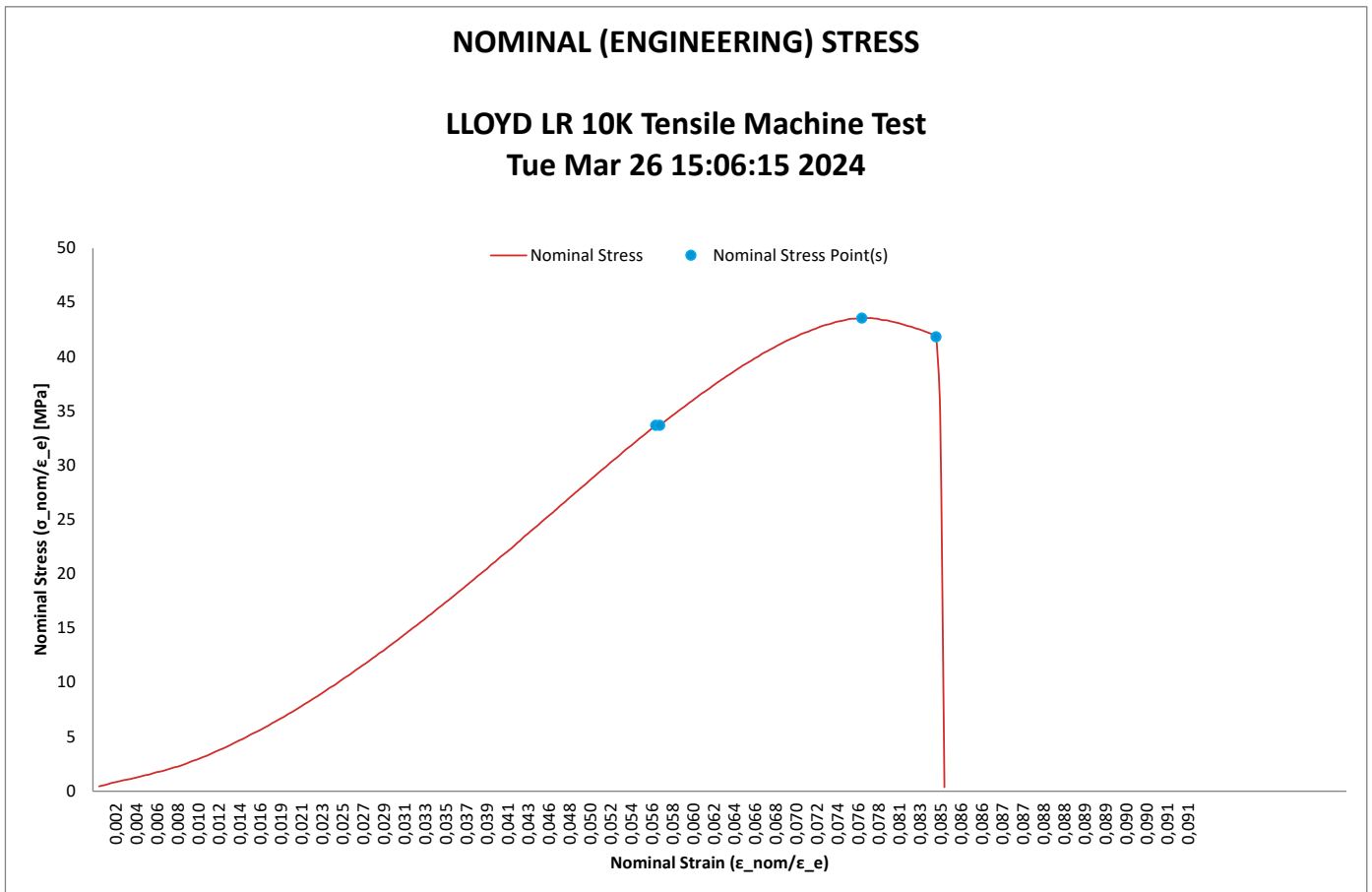
C.7 Results T2-7

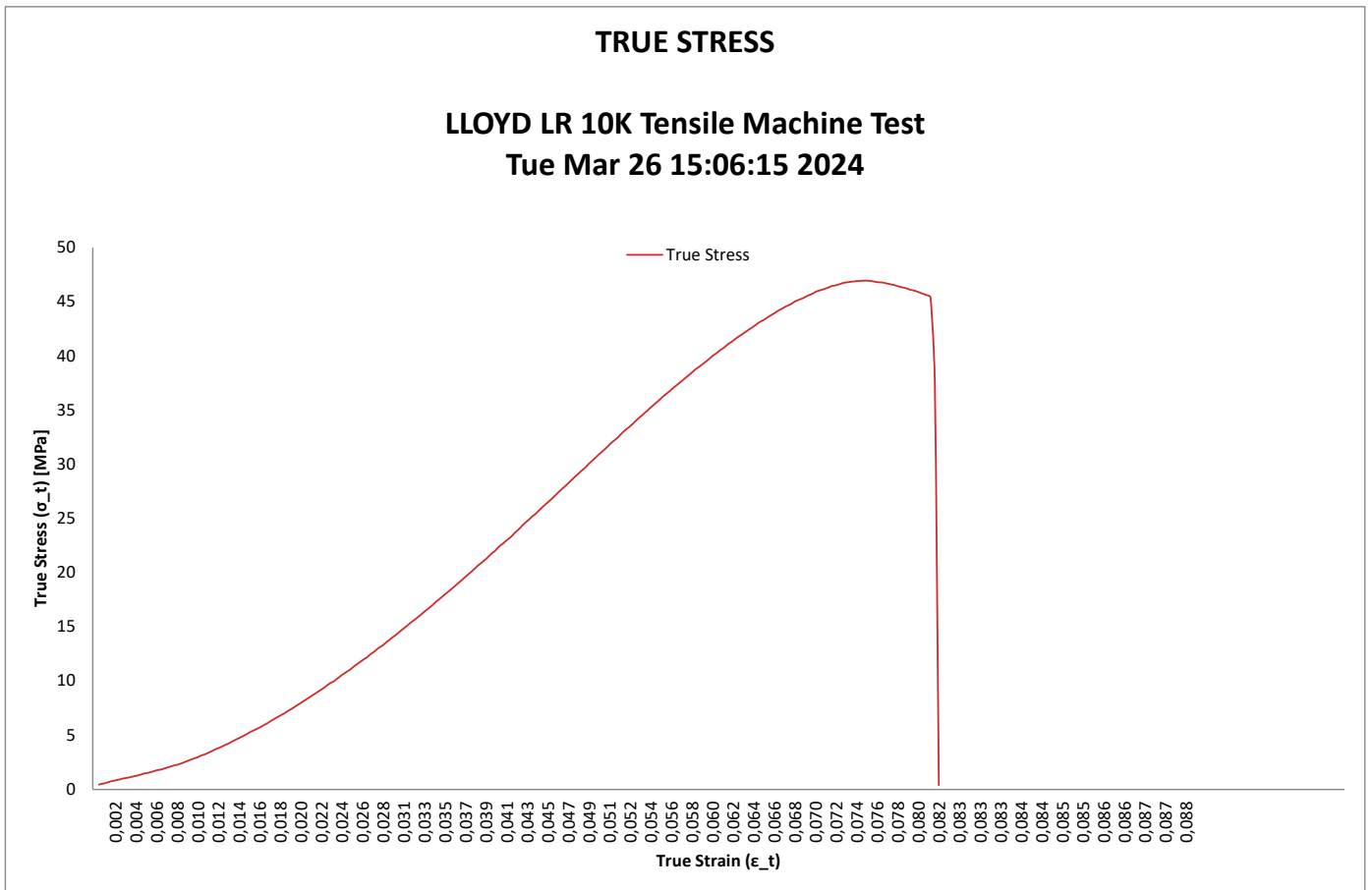
<p>Tue Mar 26 15:06:15 2024 Details:</p>	<p>MATERIAL TESTING REPORT</p> <p>T2-7</p>	
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	33,676	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,64%	
<u>Young's modulus, E</u>	597,099	MPa [N/mm ²]
True Stress, σ_t	35,576	MPa [N/mm ²]
True Strain, ϵ_t	5,49%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	33,676	MPa [N/mm ²]
Nominal Strain, ϵ_n	5,64%	
True Stress, σ_t	35,576	MPa [N/mm ²]
True Strain, ϵ_t	5,49%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	43,565	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,68%	
True Stress, σ_t	46,912	MPa [N/mm ²]
True Strain, ϵ_t	7,40%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	41,843	MPa [N/mm ²]
Nominal Strain, ϵ_n	8,43%	
True Stress, σ_t	45,370	MPa [N/mm ²]
True Strain, ϵ_t	8,09%	

LR 10K Tensile Machine Settings**Tue Mar 26 15:06:15 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%





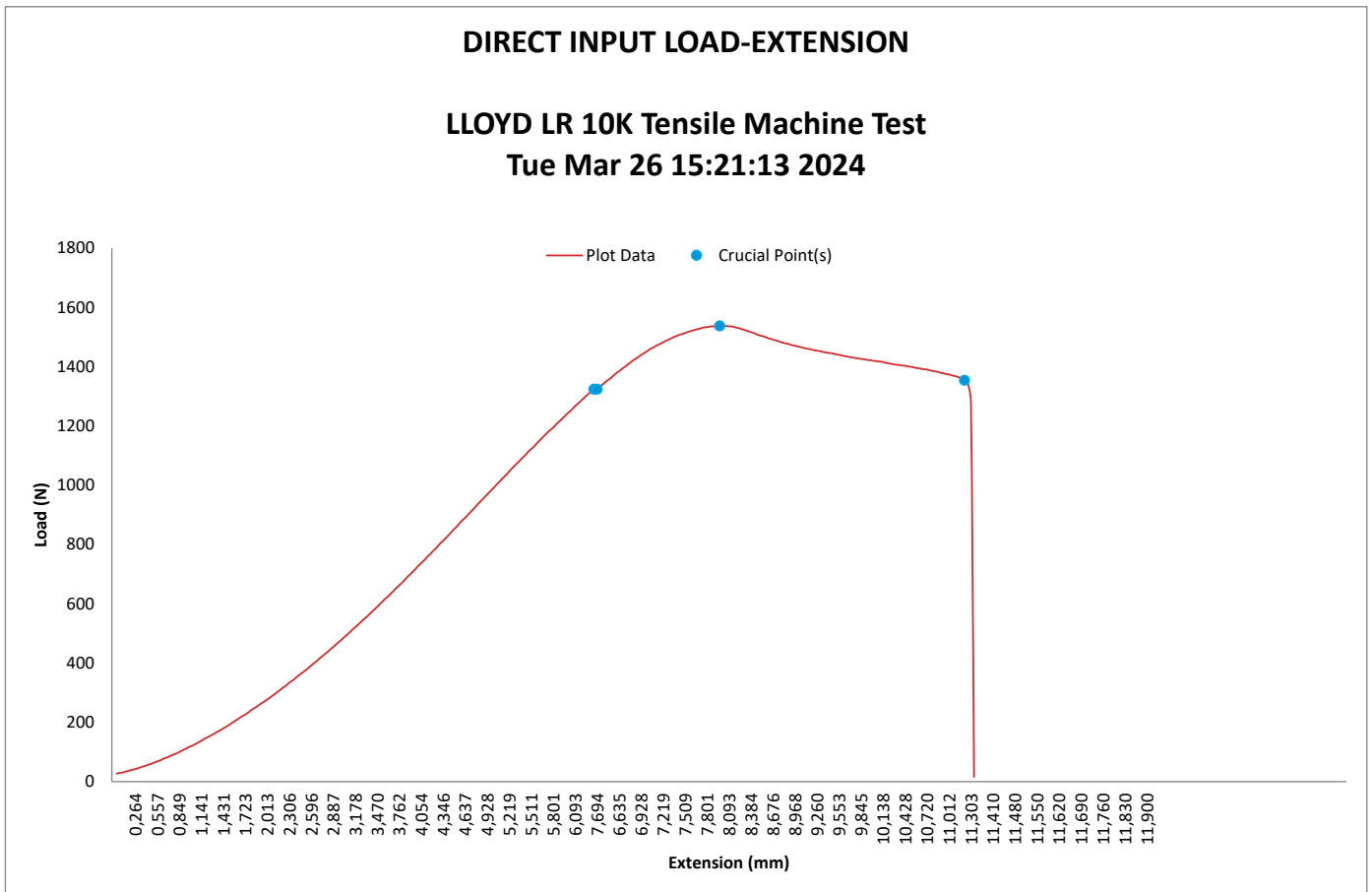


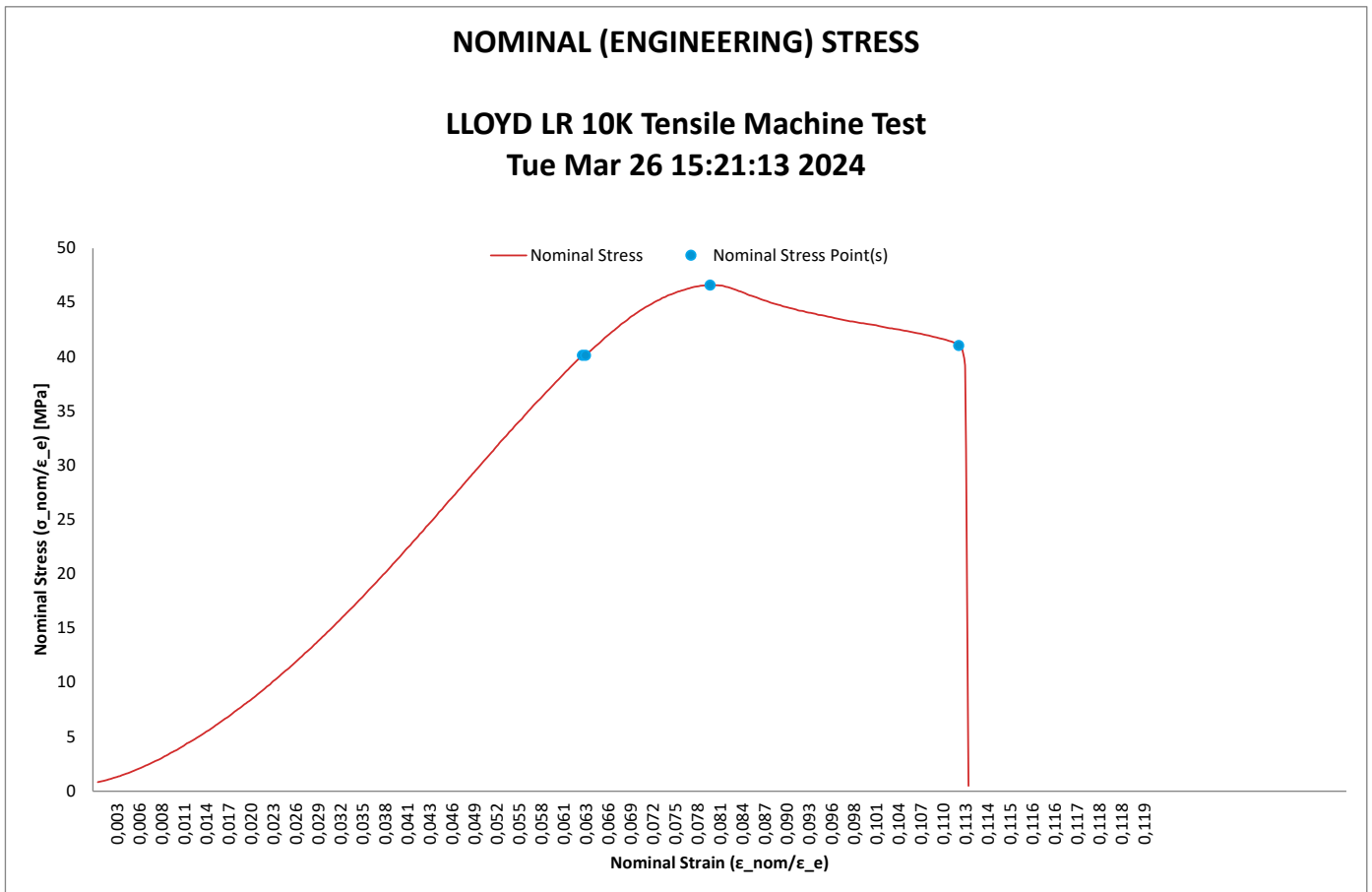
C.8 Results T2-8

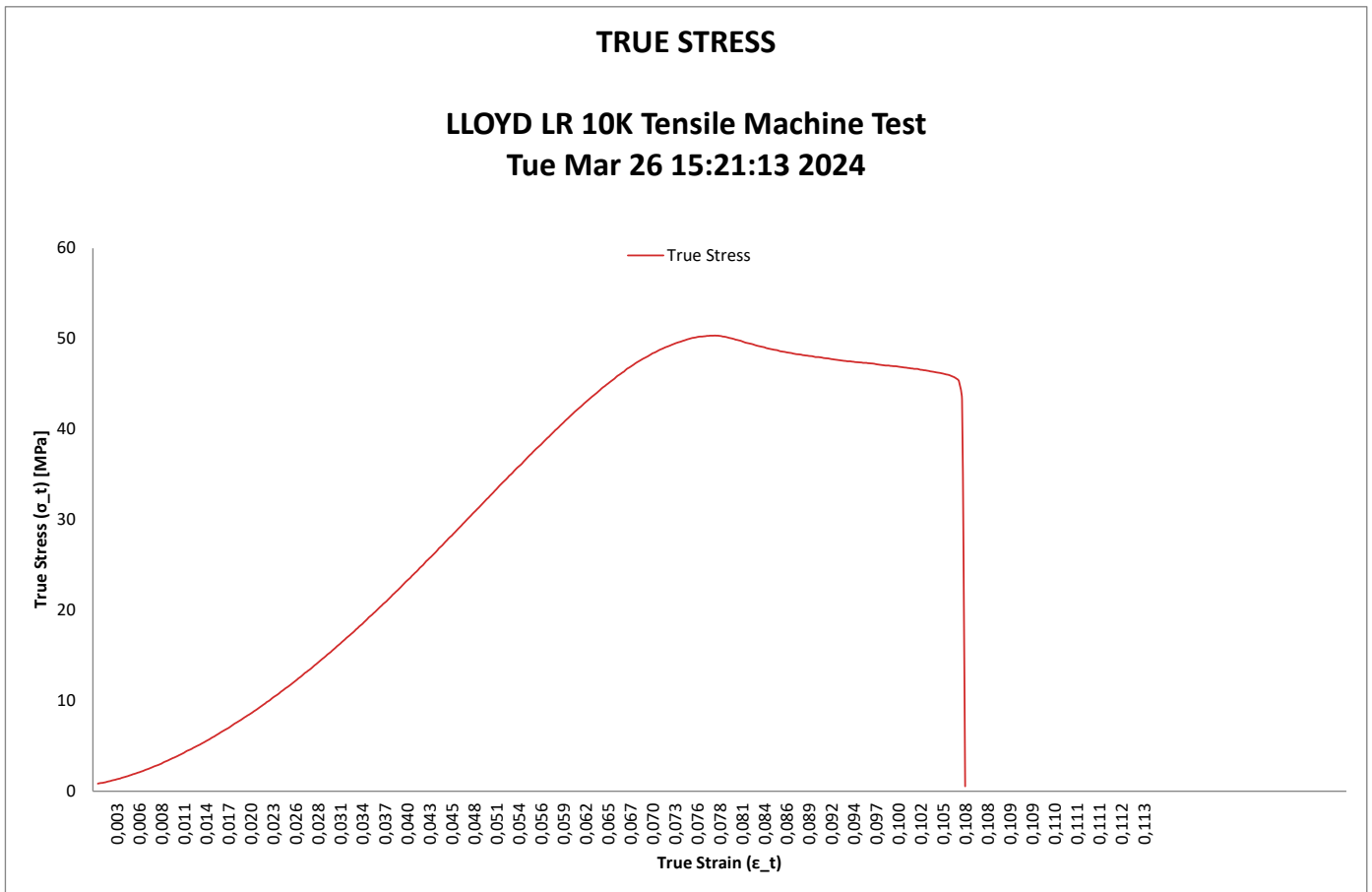
MATERIAL TESTING REPORT		
Tue Mar 26 15:21:13 2024		
Details:	Test T2-8	
Material Properties:		
Start Thickness	3,000	mm
Start Width	11,000	mm
Start Length	100,000	mm
Proportional limit:		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,121	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,34%	
<u>Young's modulus, E</u>	632,418	MPa [N/mm ²]
True Stress, σ_t	42,666	MPa [N/mm ²]
True Strain, ϵ_t	6,15%	
Yielding (lower):		
Nominal Stress, σ_{nom}/σ_e (S _y)	40,121	MPa [N/mm ²]
Nominal Strain, ϵ_n	6,34%	
True Stress, σ_t	42,666	MPa [N/mm ²]
True Strain, ϵ_t	6,15%	
Ultimate Tensile Strength:		
Nominal Stress, σ_{nom}/σ_e (S _{ut})	46,595	MPa [N/mm ²]
Nominal Strain, ϵ_n	7,97%	
True Stress, σ_t	50,307	MPa [N/mm ²]
True Strain, ϵ_t	7,67%	
Fracture:		
Nominal Stress, σ_{nom}/σ_e (S _y)	41,041	MPa [N/mm ²]
Nominal Strain, ϵ_n	11,22%	
True Stress, σ_t	45,646	MPa [N/mm ²]
True Strain, ϵ_t	10,63%	

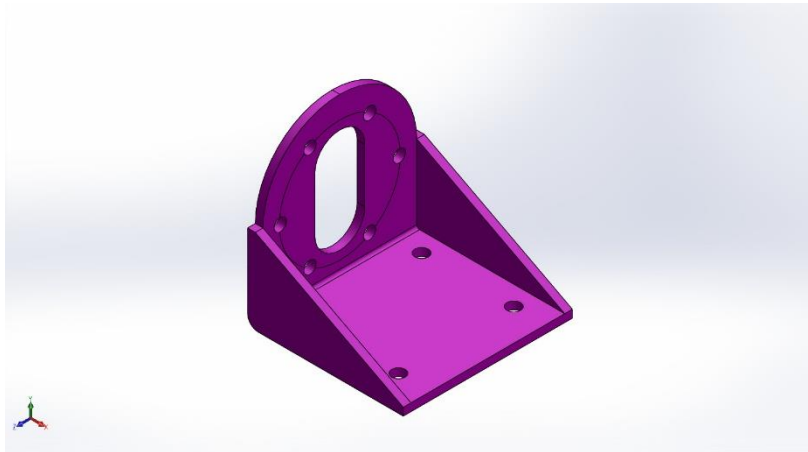
LR 10K Tensile Machine Settings**Tue Mar 26 15:21:13 2024***Details:* ...

Crosshead Speed	50,000	mm/min
(Jog speed)	100,000	mm/min
Load error (graph)	-2,000	%









Simulation of 520 DC motor fastener

Date: fredag 17. mai 2024
Designer: Adrian Elias Haugjord
Study name: Weight
Analysis type: Static

Table of Contents

Description	1
Assumptions.....	2
Model Information	2
Study Properties.....	3
Units	3
Material Properties	4
Loads and Fixtures	5
Connector Definitions	6
Interaction Information	6
Mesh information.....	7
Study Results	8
Conclusion.....	11

Description

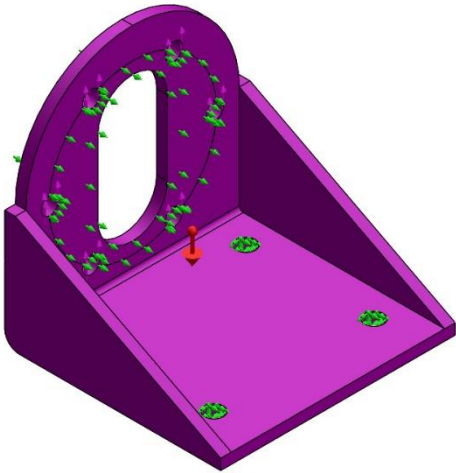
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Assumptions

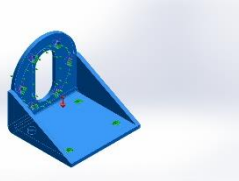
When mounted on the car, the force from the weight of the car will work parallel to the ZY-plane in the positive Y direction.

The weight of the car is 5,4 kilograms distributed over the four brackets. 1,35 kilograms of load will be applied on the bracket, for the simulation we rounded this up to 15 Newtons.

Model Information



Model name: 520 DC motor fastener
Current Configuration: 3d printable

Solid Bodies			
Document Name and Reference	Treated As	Volumetric Properties	Document Path/Date Modified
Split Line1 	Solid Body	Mass:0,00920958 kg Volume:7,25164e-06 m ³ Density:1 270 kg/m ³ Weight:0,0902539 N	C:\Users\haugj\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Final model\Robot car iteration 3-2024_04_29_1428\Brackets & fasteners\520 DC motor fastener.SLDPRT May 17 13:26:00 2024

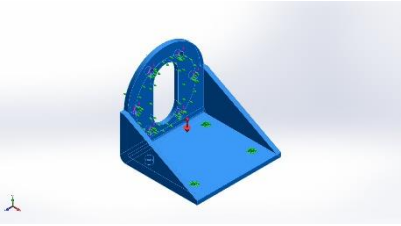
Study Properties

Study name	Weight
Analysis type	Static
Mesh type	Solid Mesh
Thermal Effect:	On
Thermal option	Include temperature loads
Zero strain temperature	298 Kelvin
Include fluid pressure effects from SOLIDWORKS Flow Simulation	Off
Solver type	Automatic
Inplane Effect:	Off
Soft Spring:	Off
Inertial Relief:	Off
Incompatible bonding options	Automatic
Large displacement	Off
Compute free body forces	On
Friction	Off
Use Adaptive Method:	Off
Result folder	SOLIDWORKS document (C:\Users\haugj\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Final model\Robot car iteration 3-2024_04_29_1428\Brackets & fasteners)


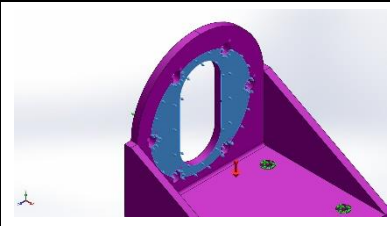
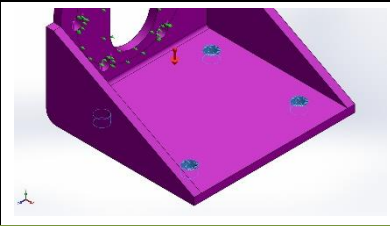
Units

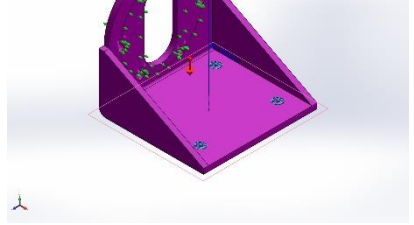
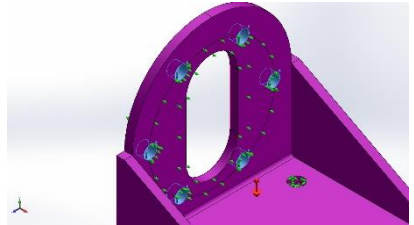
Unit system:	SI (MKS)
Length/Displacement	mm
Temperature	Kelvin
Angular velocity	Rad/sec
Pressure/Stress	N/mm ² (MPa)

Material Properties

Model Reference	Properties	Components
	<p> Name: Clas Ohlson PLA Model type: Linear Elastic Isotropic Default failure criterion: Unknown Yield strength: 35,607 N/mm² Tensile strength: 48,859 N/mm² Elastic modulus: 651,75 N/mm² Poisson's ratio: 0,36 Mass density: 1,27 g/cm³ </p>	<p>SolidBody 1(Split Line1)(520 DC motor fastener)</p>
<p>Curve Data:N/A</p>		

Loads and Fixtures

Fixture name	Fixture Image	Fixture Details		
Roller/Slider-1		Entities: 2 face(s) Type: Roller/Slider		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	-27,4228	-15,2269	0,00790618	31,3667
Reaction Moment(N.m)	0	0	0	0
Roller/Slider-2		Entities: 1 face(s) Type: Roller/Slider		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	23,1315	8,02534e-14	-4,01267e-15	23,1315
Reaction Moment(N.m)	0	0	0	0
Fixed-1		Entities: 4 face(s) Type: Fixed Geometry		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	4,69682	0,246572	6,11799e-08	4,70329
Reaction Moment(N.m)	0	0	0	0

Load name	Load Image	Load Details
Gravity-1		<p>Reference: Top Plane Values: 0 0 -9,81 Units: m/s²</p>
Force-1		<p>Entities: 6 face(s) Reference: Edge< 1 > Type: Apply force Values: ---; ---; 15 N</p>

Connector Definitions

No Data

Interaction Information

No Data

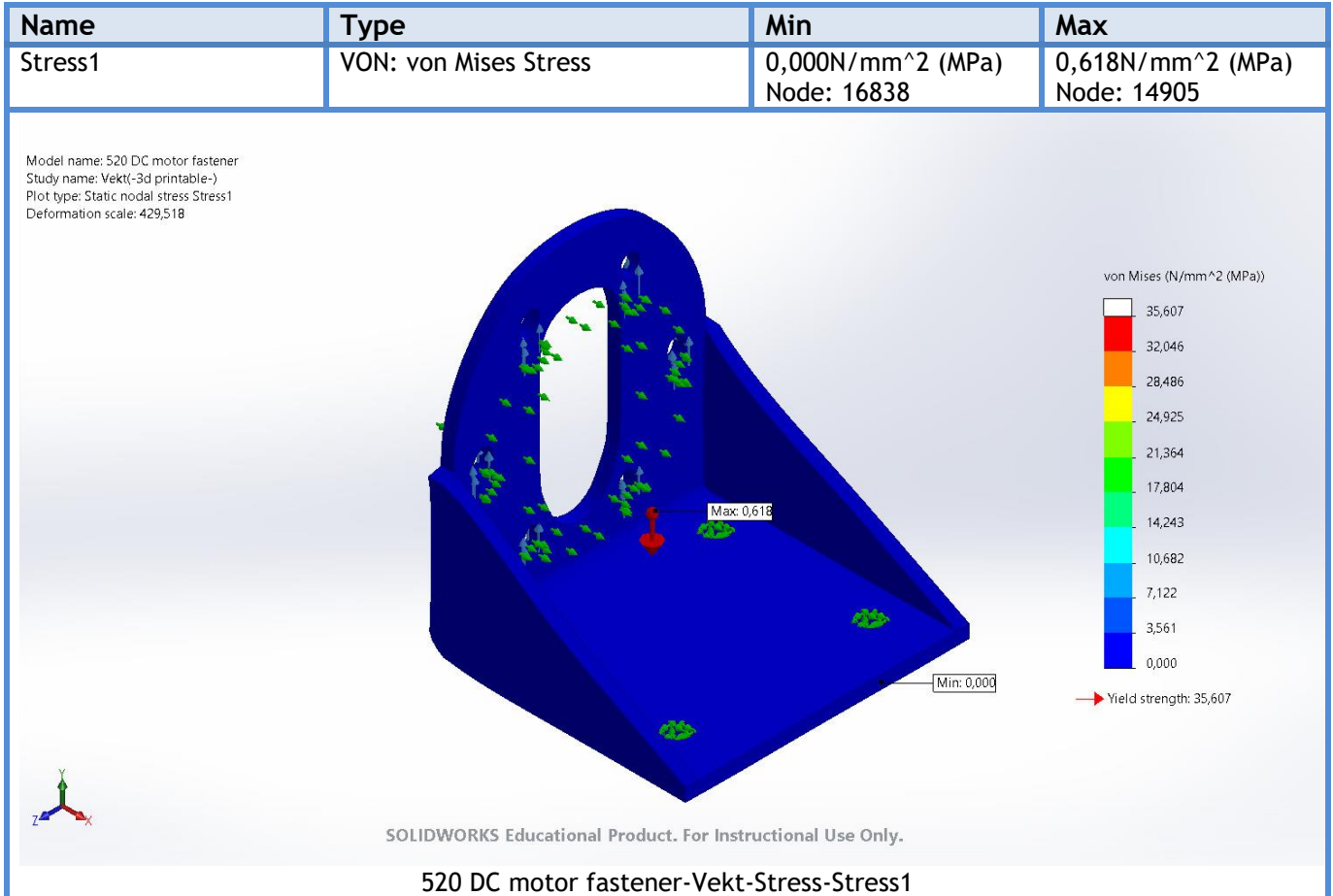
Mesh information

Mesh type	Solid Mesh
Mesher Used:	Blended curvature-based mesh
Jacobian points for High quality mesh	16 Points
Maximum element size	2,52973 mm
Minimum element size	0,563769 mm
Mesh Quality	High

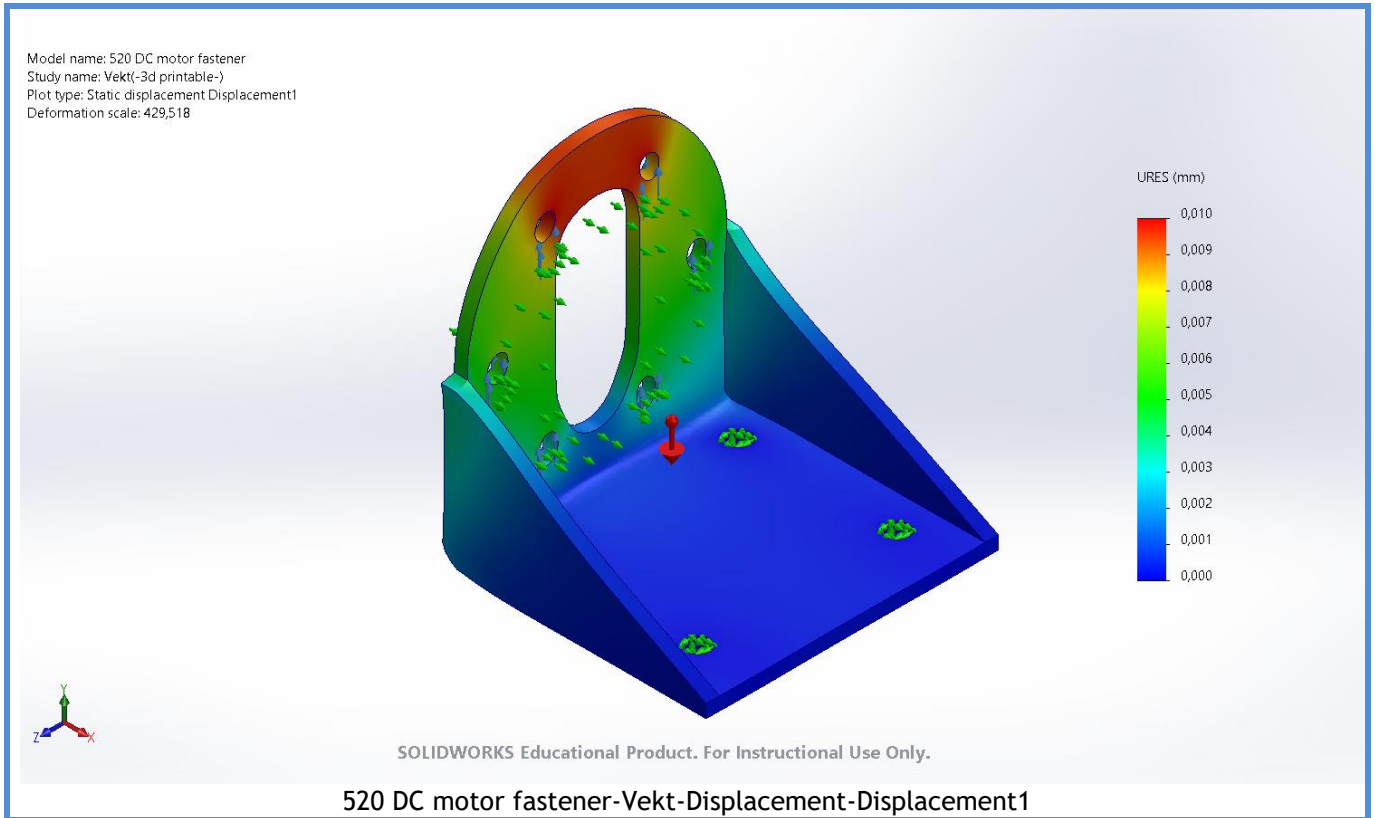
Mesh information - Details

Total Nodes	17449
Total Elements	9128
Maximum Aspect Ratio	5,871
% of elements with Aspect Ratio < 3	96,6
Percentage of elements with Aspect Ratio > 10	0
Percentage of distorted elements	0
Time to complete mesh(hh:mm:ss):	00:00:07
Computer name:	

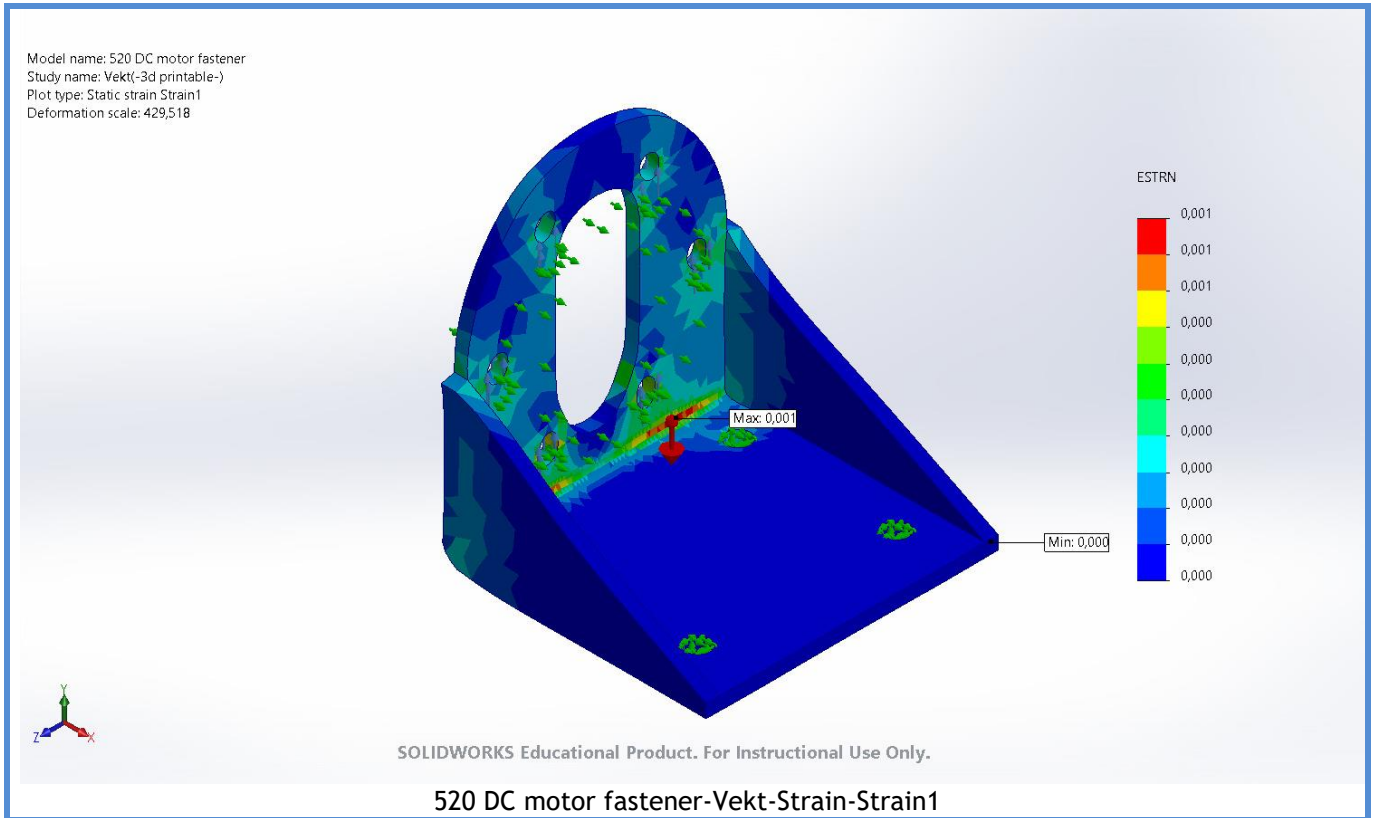
Study Results



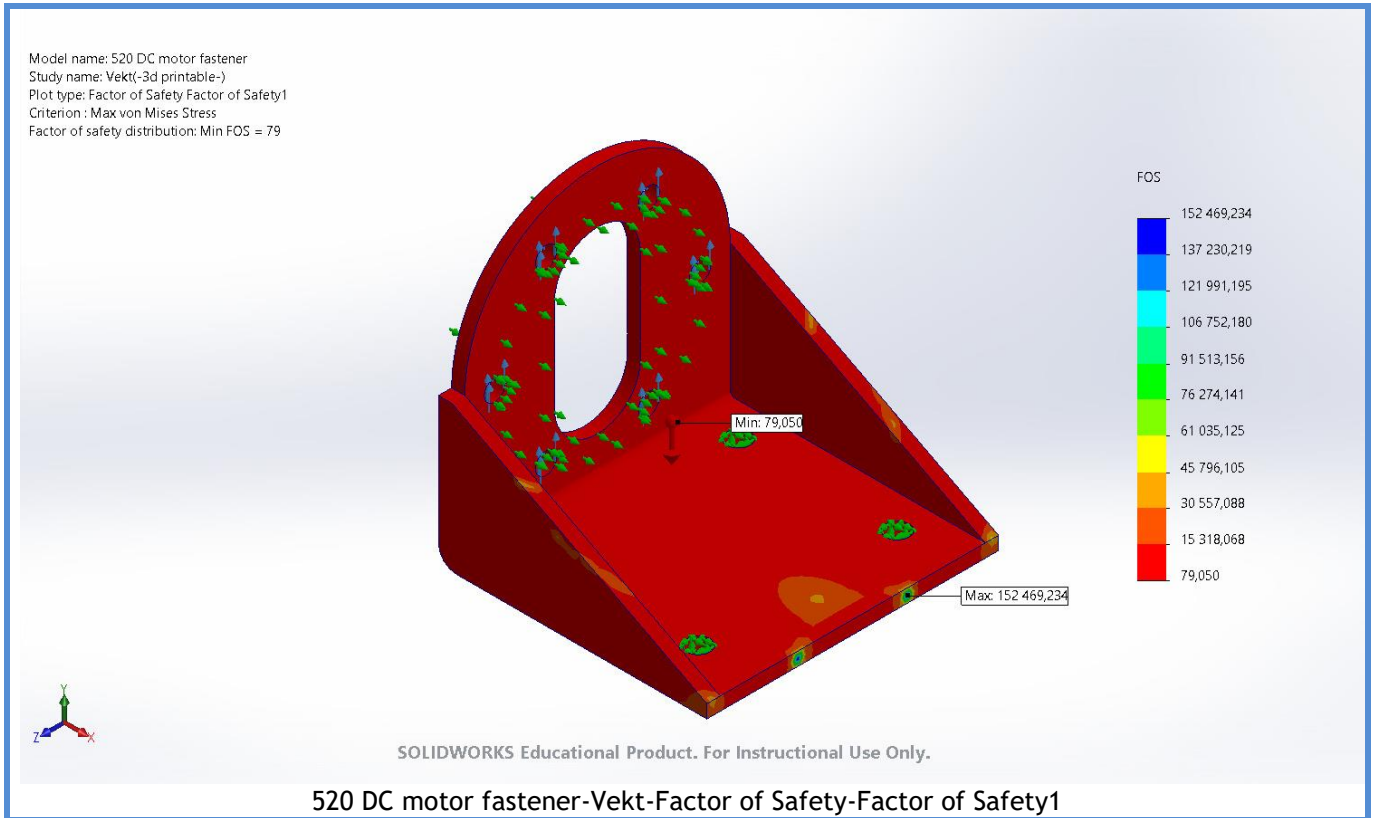
Name	Type	Min	Max
Displacement1	URES: Resultant Displacement	0,000mm Node: 18	0,010mm Node: 13826



Name	Type	Min	Max
Strain1	ESTRN: Equivalent Strain	0,000 Element: 8118	0,001 Element: 7524

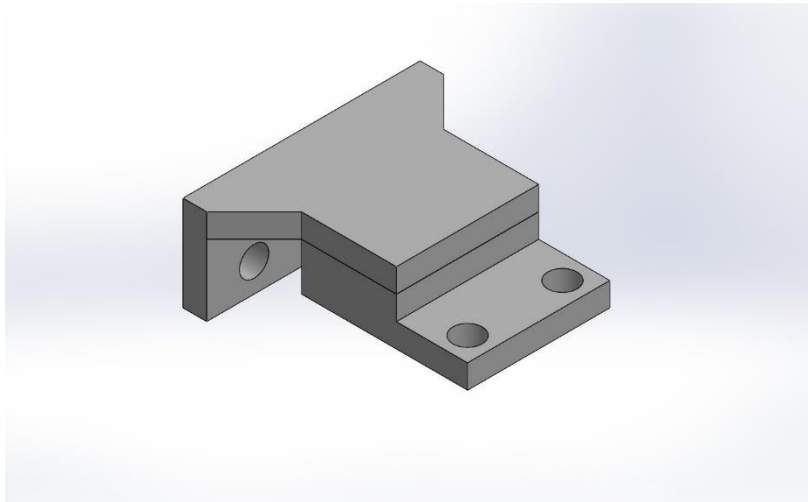


Name	Type	Min	Max
Factor of Safety1	Max von Mises Stress	79,050 Node: 14905	152 469,234 Node: 16838



Conclusion

When only taking the weight of the car into consideration the brackets will work without a problem.



Simulation of Wall attachment bracket assy

Date: fredag 17. mai 2024
Designer: Henrik Bertelsen
Study name: Static 1
Analysis type: Static

Table of Contents

Description	1
Assumptions.....	2
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Material Properties	5
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Interaction Information	7
Mesh information.....	8
Sensor Details.....	8
Resultant Forces	9
Beams.....	9
Study Results	10
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Description

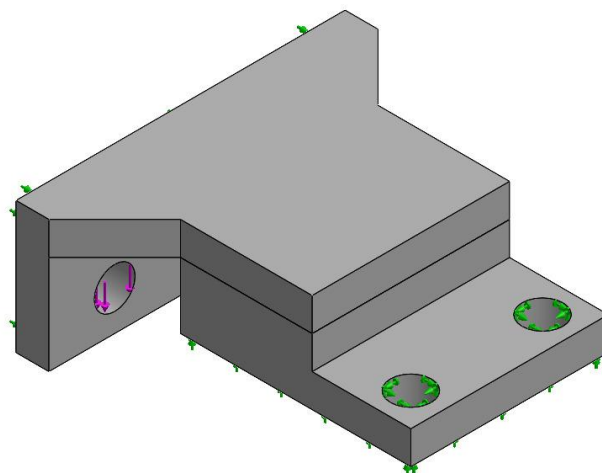
Simulation of wall bracket assembly

Assumptions

Comments:

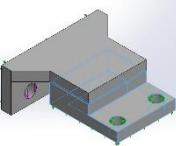
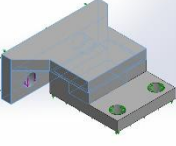
Assuming the lower part are attached to the robot car and the upper part are attached to the wall. The forces represent the wall being put in to its attachment (lower bracket), the force is sat to 2.5 Newtons.

Model Information



Model name: Wall attachment bracket assy
Current Configuration: Default

Solid Bodies

Document Name and Reference	Treated As	Volumetric Properties	Document Path/Date Modified
Female bracket hole camfer 	Solid Body	Mass:0,000810232 kg Volume:6,37978e-07 m ³ Density:1 270 kg/m ³ Weight:0,00794028 N	C:\Users\Henrik\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Individuelle deler\Walls\Wall attachment bracket female.SLDPRT May 10 17:11:56 2024
Support camfer 	Solid Body	Mass:0,00196926 kg Volume:1,55059e-06 m ³ Density:1 270 kg/m ³ Weight:0,0192987 N	C:\Users\Henrik\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Individuelle deler\Walls\Wall attachment bracket male.SLDPRT May 10 17:11:21 2024

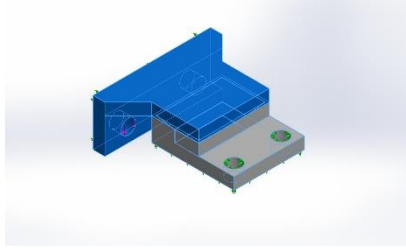
Study Properties

Study name	Static 1
Analysis type	Static
Mesh type	Solid Mesh
Thermal Effect:	On
Thermal option	Include temperature loads
Zero strain temperature	298 Kelvin
Include fluid pressure effects from SOLIDWORKS Flow Simulation	Off
Solver type	Automatic
Inplane Effect:	Off
Soft Spring:	Off
Inertial Relief:	Off
Incompatible bonding options	Automatic
Large displacement	On
Compute free body forces	On
Friction	Off
Use Adaptive Method:	Off
Result folder	SOLIDWORKS document (C:\Users\Henrik\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Individuelle deler\Walls)

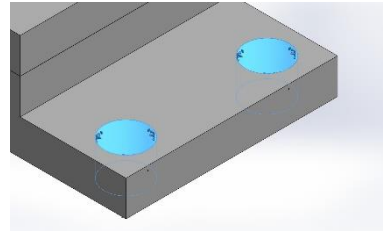
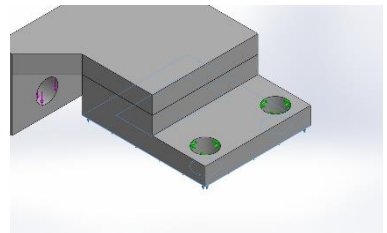
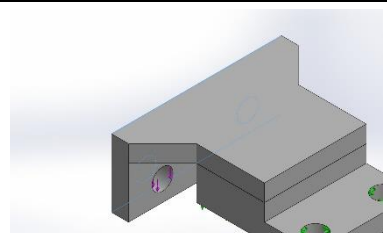
Units

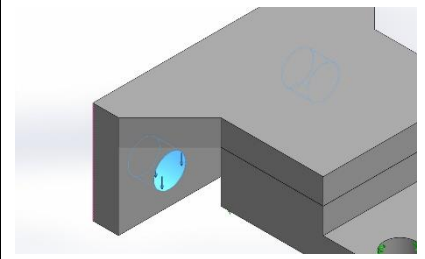
Unit system:	SI (MKS)
Length/Displacement	mm
Temperature	Kelvin
Angular velocity	Rad/sec
Pressure/Stress	N/m ²

Material Properties

Model Reference	Properties	Components
	<p>Name: Clas Ohlson PLA Model type: Linear Elastic Isotropic Default failure criterion: Unknown Yield strength: 3,5607e+07 N/m² Tensile strength: 4,8859e+07 N/m² Elastic modulus: 6,5175e+08 N/m² Poisson's ratio: 0,331 Mass density: 1 270 kg/m³</p>	<p>SolidBody 1(Female bracket hole camfer)(Wall attachment bracket female-1), SolidBody 1(Support camfer)(Wall attachment bracket male-1)</p>
<p>Curve Data:N/A</p>		

Loads and Fixtures

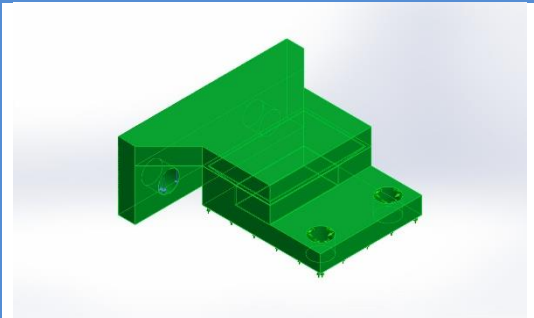
Fixture name	Fixture Image	Fixture Details		
Fixed-1		Entities: 2 face(s) Type: Fixed Geometry		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	0,219818	0,0310233	-3,31784e-08	0,221996
Reaction Moment(N.m)	0	0	0	0
Roller/Slider-1		Entities: 1 face(s) Type: Roller/Slider		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	0,0115366	9,96546	0,000215633	9,96547
Reaction Moment(N.m)	0	0	0	0
Roller/Slider-2		Entities: 1 face(s) Type: Roller/Slider		
Resultant Forces				
Components	X	Y	Z	Resultant
Reaction force(N)	-0,219787	0	0	0,219787
Reaction Moment(N.m)	0	0	0	0

Load name	Load Image	Load Details
Force-1		Entities: 2 face(s) Reference: Edge< 1 > Type: Apply force Values: ---; ---; -5 N

Connector Definitions

No Data

Interaction Information

Interaction	Interaction Image	Interaction Properties
Component Interaction-1		Type: Contact (Surface to surface) Components: 2 Solid Body (s)

Mesh information

Mesh type	Solid Mesh
Mesher Used:	Blended curvature-based mesh
Jacobian points for High quality mesh	16 Points
Maximum element size	0,90923 mm
Minimum element size	0,303074 mm
Mesh Quality	High
Remesh failed parts independently	Off

Mesh information - Details

Total Nodes	38081
Total Elements	23822
Maximum Aspect Ratio	3,8851
% of elements with Aspect Ratio < 3	99,9
Percentage of elements with Aspect Ratio > 10	0
Percentage of distorted elements	0
Time to complete mesh(hh:mm:ss):	00:00:10
Computer name:	

Sensor Details

No Data

Resultant Forces

Reaction forces

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N	3,02866e-05	10,0001	-3,31784e-08	10,0001

Reaction Moments

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N.m	0	0	0	0

Free body forces

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N	0	0	0	0

Free body moments

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N.m	0	0	0	0

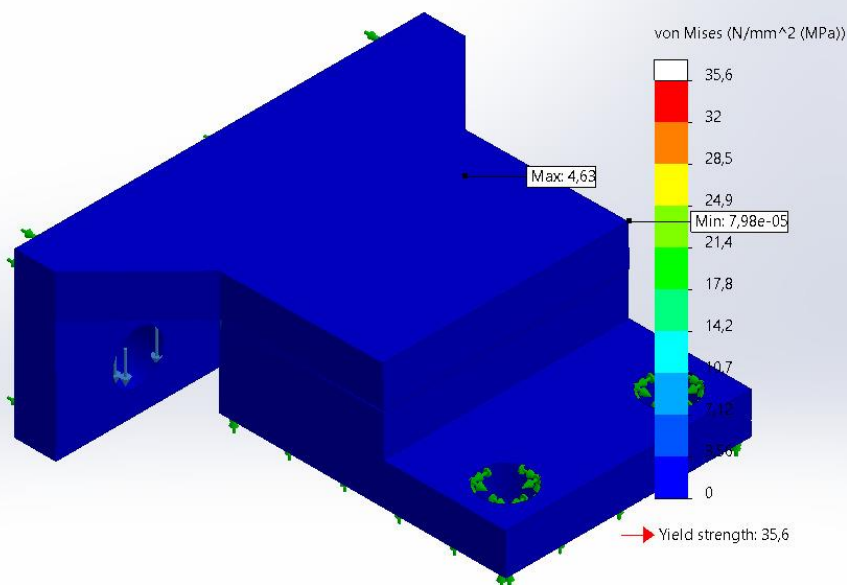
Beams

No Data

Study Results

Name	Type	Min	Max
Stress1	VON: von Mises Stress	7,98e-05N/mm ² (MPa) Node: 11894	4,63N/mm ² (MPa) Node: 1

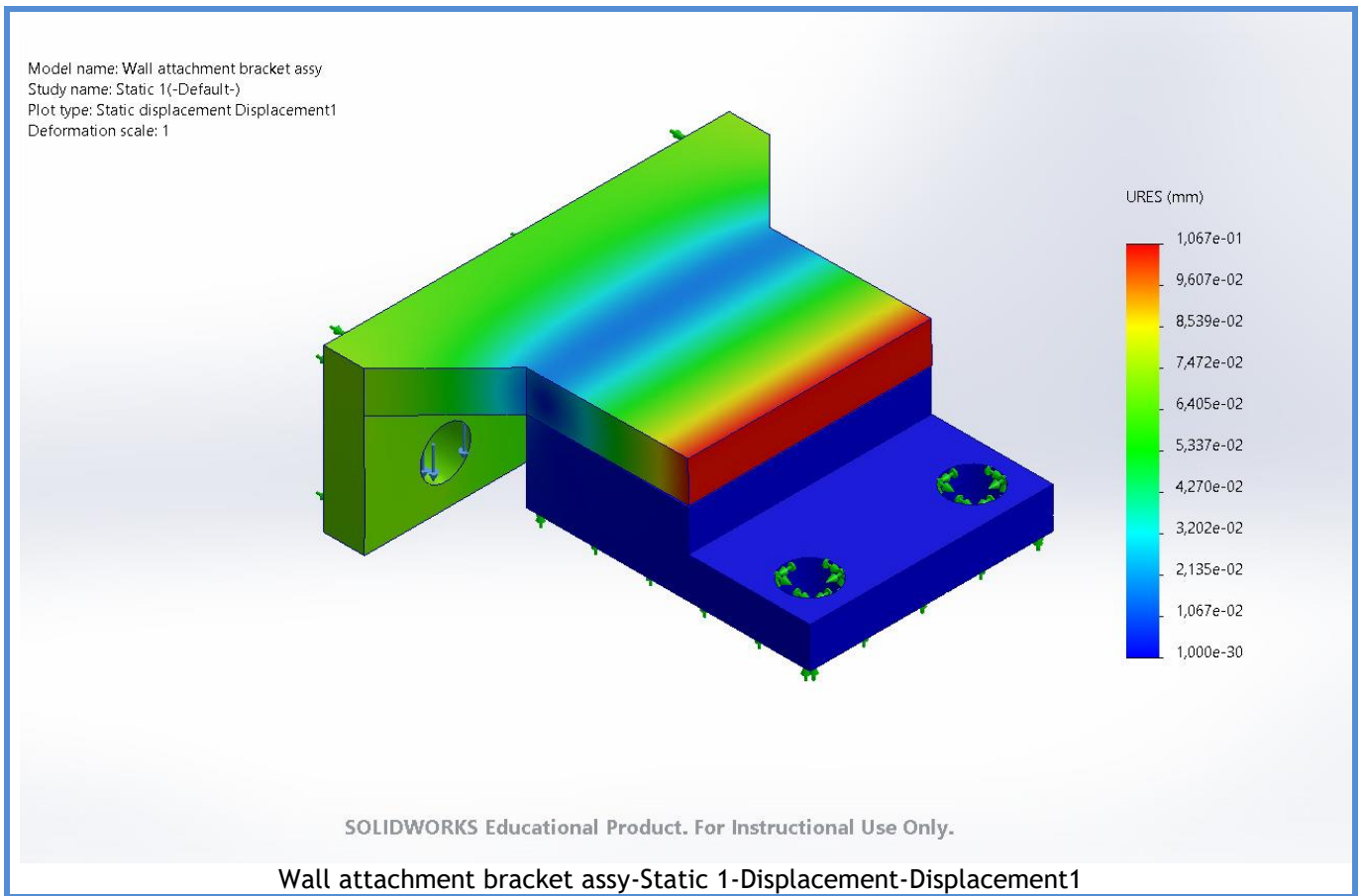
Model name: Wall attachment bracket assy
Study name: Static 1(-Default-)
Plot type: Static nodal stress Stress1
Deformation scale: 1



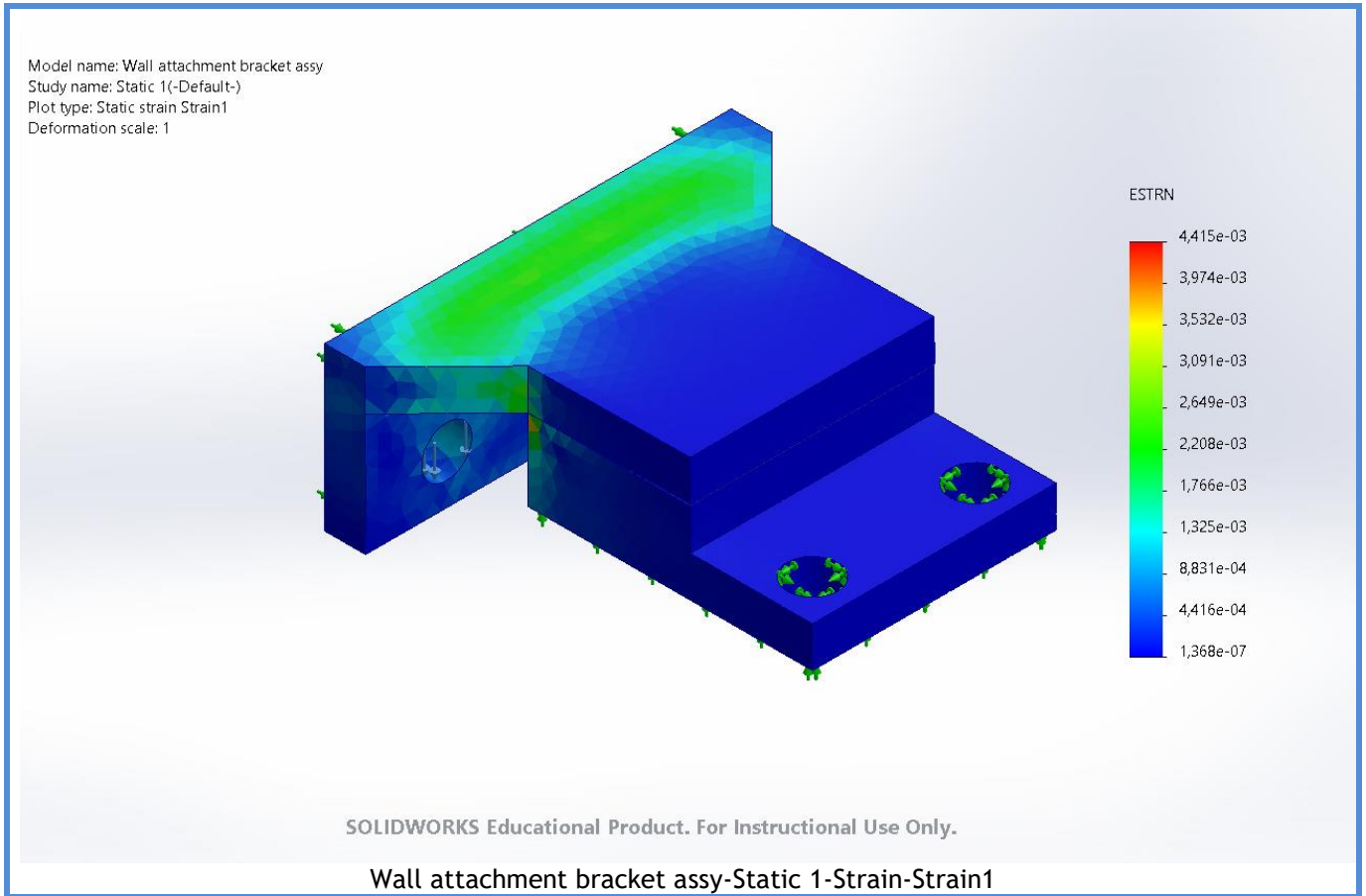
SOLIDWORKS Educational Product. For Instructional Use Only.

Wall attachment bracket assy-Static 1-Stress-Stress1

Name	Type	Min	Max
Displacement1	URES: Resultant Displacement	0,000e+00mm Node: 5	1,067e-01mm Node: 12166

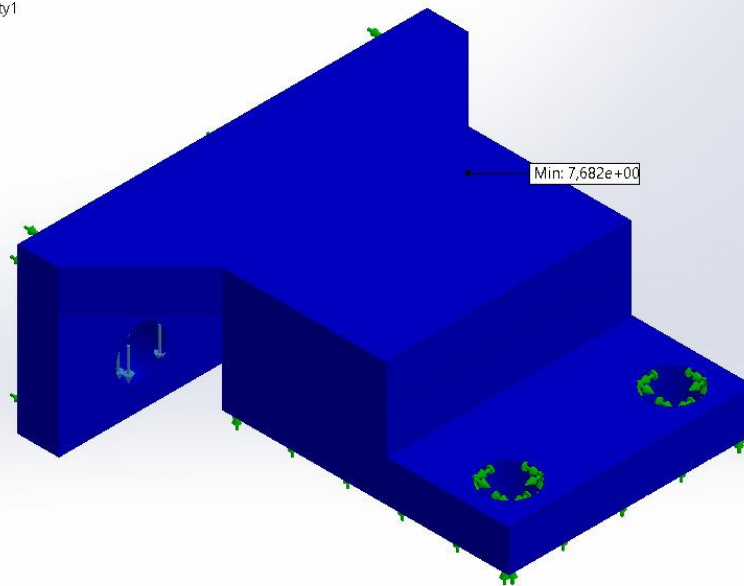


Name	Type	Min	Max
Strain1	ESTRN: Equivalent Strain	1,368e-07 Element: 23035	4,415e-03 Element: 1807



Name	Type	Min	Max
Factor of Safety1	Automatic	7,682e+00 Node: 1	4,464e+05 Node: 11894

Model name: Wall attachment bracket assy
Study name: Static 1(-Default-)
Plot type: Factor of Safety Factor of Safety1
Criterion : Automatic
Red < FOS = 2 < Blue



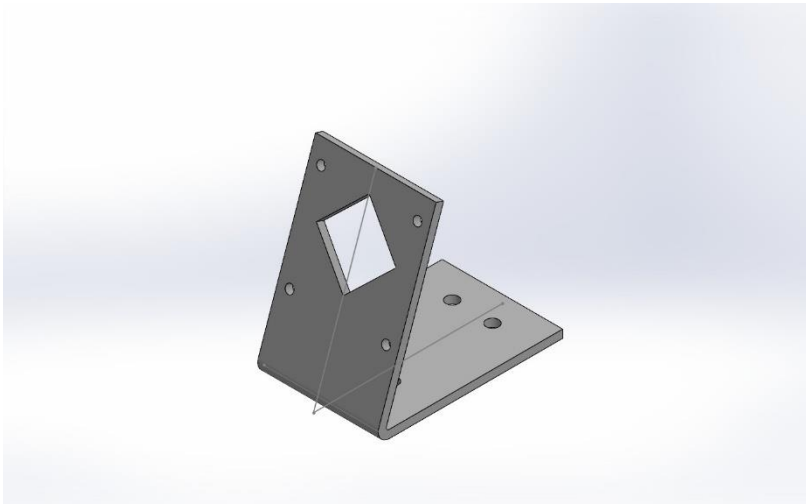
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Wall attachment bracket assy-Static 1-Factor of Safety-Factor of Safety1

Conclusion

Comments:

The brackets will hold the weight of the walls and the clips attaching and detaching.



Simulation of Arm cam holder fixed angle

Date: fredag 17. mai 2024
Designer: Henrik Bertelsen
Study name: Static 1
Analysis type: Static

Table of Contents

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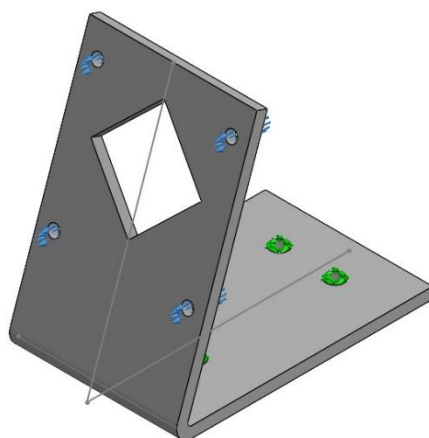
Description
Simulation of a part

Assumptions

Comments:

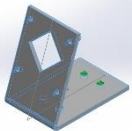
If the arm camera cable is too loose it can attach itself to the car. The forces added to the simulation is meant to represent the forces of the cable dragging in the arm camera bracket.

Model Information



Model name: Arm cam holder fixed angle
Current Configuration: Default

Solid Bodies

Document Name and Reference	Treated As	Volumetric Properties	Document Path/Date Modified
Arm screw holes diam.3.1mm 	Solid Body	Mass:0,00626532 kg Volume:4,93332e-06 m ³ Density:1 270 kg/m ³ Weight:0,0614001 N	C:\Users\Henrik\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Individuelle deler\Arm camera\Arm cam holder fixed angle.SLDPRT May 17 19:53:44 2024

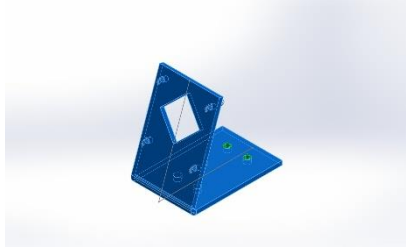
Study Properties

Study name	Static 1
Analysis type	Static
Mesh type	Solid Mesh
Thermal Effect:	On
Thermal option	Include temperature loads
Zero strain temperature	298 Kelvin
Include fluid pressure effects from SOLIDWORKS Flow Simulation	Off
Solver type	Automatic
Inplane Effect:	Off
Soft Spring:	Off
Inertial Relief:	Off
Incompatible bonding options	Automatic
Large displacement	On
Compute free body forces	On
Friction	Off
Use Adaptive Method:	Off
Result folder	SOLIDWORKS document (C:\Users\Henrik\OneDrive - USN\Dokumenter - Bachelor gruppe\04 Mekanikk\Deler\3D-models\Individuelle deler\Arm camera)

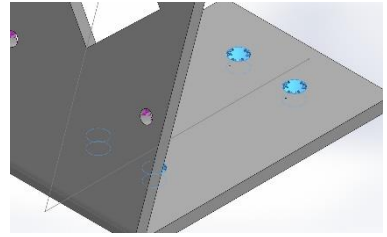
Units

Unit system:	SI (MKS)
Length/Displacement	mm
Temperature	Kelvin
Angular velocity	Rad/sec
Pressure/Stress	N/m ²

Material Properties

Model Reference	Properties	Components
	<p> Name: Clas Ohlson PLA Model type: Linear Elastic Isotropic Default failure criterion: Unknown Yield strength: 3,5607e+07 N/m² Tensile strength: 4,8859e+07 N/m² Elastic modulus: 6,5175e+08 N/m² Poisson's ratio: 0,331 Mass density: 1 270 kg/m³ </p>	<p>SolidBody 1(Arm screw holes diam.3.1mm)(Arm cam holder fixed angle)</p>
<p>Curve Data:N/A</p>		

Loads and Fixtures

Fixture name	Fixture Image	Fixture Details
Fixed-1		Entities: 4 face(s) Type: Fixed Geometry

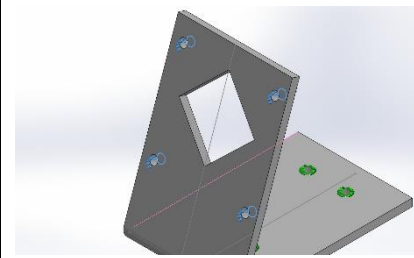
Resultant Forces

Components	X	Y	Z	Resultant
Reaction force(N)	-4,18797e-05	0,662924	-10	10,022
Reaction Moment(N.m)	0	0	0	0

Reference Geometry-1		Entities: 1 face(s) Reference: Edge< 1 > Type: Use reference geometry Translation: ---; ---; 0 Units: mm
----------------------	--	---

Resultant Forces

Components	X	Y	Z	Resultant
Reaction force(N)	0	-0,81006	0	0,81006
Reaction Moment(N.m)	0	0	0	0

Load name	Load Image	Load Details
Force-1		Entities: 4 edge(s) Reference: Edge< 1 > Type: Apply force Values: ---; ---; -2,5 N

Mesh information

Mesh type	Solid Mesh
Mesher Used:	Blended curvature-based mesh
Jacobian points for High quality mesh	16 Points
Maximum element size	1,43595 mm
Minimum element size	0,380544 mm
Mesh Quality	High

Mesh information - Details

Total Nodes	34131
Total Elements	19869
Maximum Aspect Ratio	4,6722
% of elements with Aspect Ratio < 3	99,5
Percentage of elements with Aspect Ratio > 10	0
Percentage of distorted elements	0
Time to complete mesh(hh:mm:ss):	00:00:06
Computer name:	

Sensor Details

No Data

Resultant Forces

Reaction forces

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N	-4,18797e-05	-0,000537096	-10	10

Reaction Moments

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N.m	0	0	0	0

Free body forces

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N	0	0	0	0

Free body moments

Selection set	Units	Sum X	Sum Y	Sum Z	Resultant
Entire Model	N.m	0	0	0	0

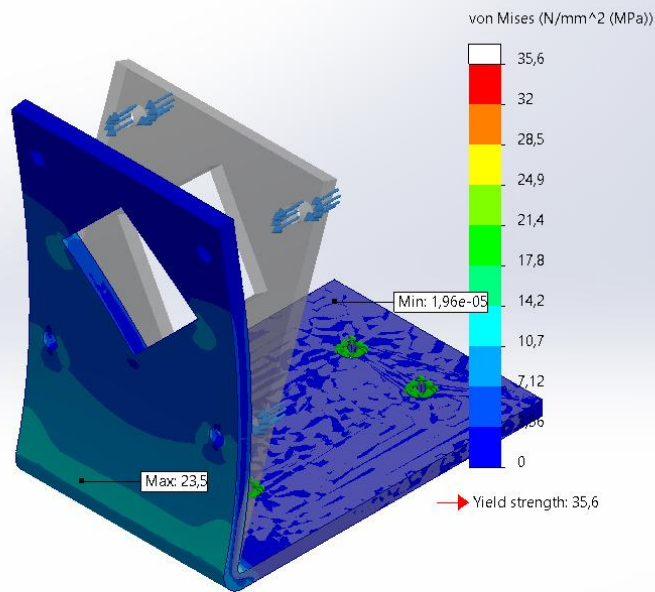
Beams

No Data

Study Results

Name	Type	Min	Max
Stress1	VON: von Mises Stress	1,96e-05N/mm ² (MPa) Node: 20109	23,5N/mm ² (MPa) Node: 1991

Model name: Arm cam holder fixed angle
Study name: Static 1(-Default-)
Plot type: Static nodal stress Stress1
Deformation scale: 1

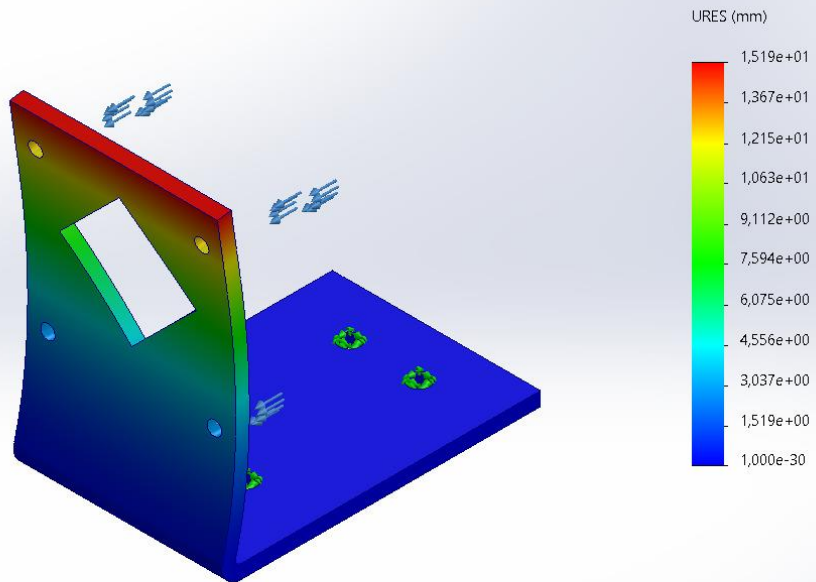


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Arm cam holder fixed angle-Static 1-Stress-Stress1

Name	Type	Min	Max
Displacement1	URES: Resultant Displacement	0,000e+00mm Node: 13	1,519e+01mm Node: 6

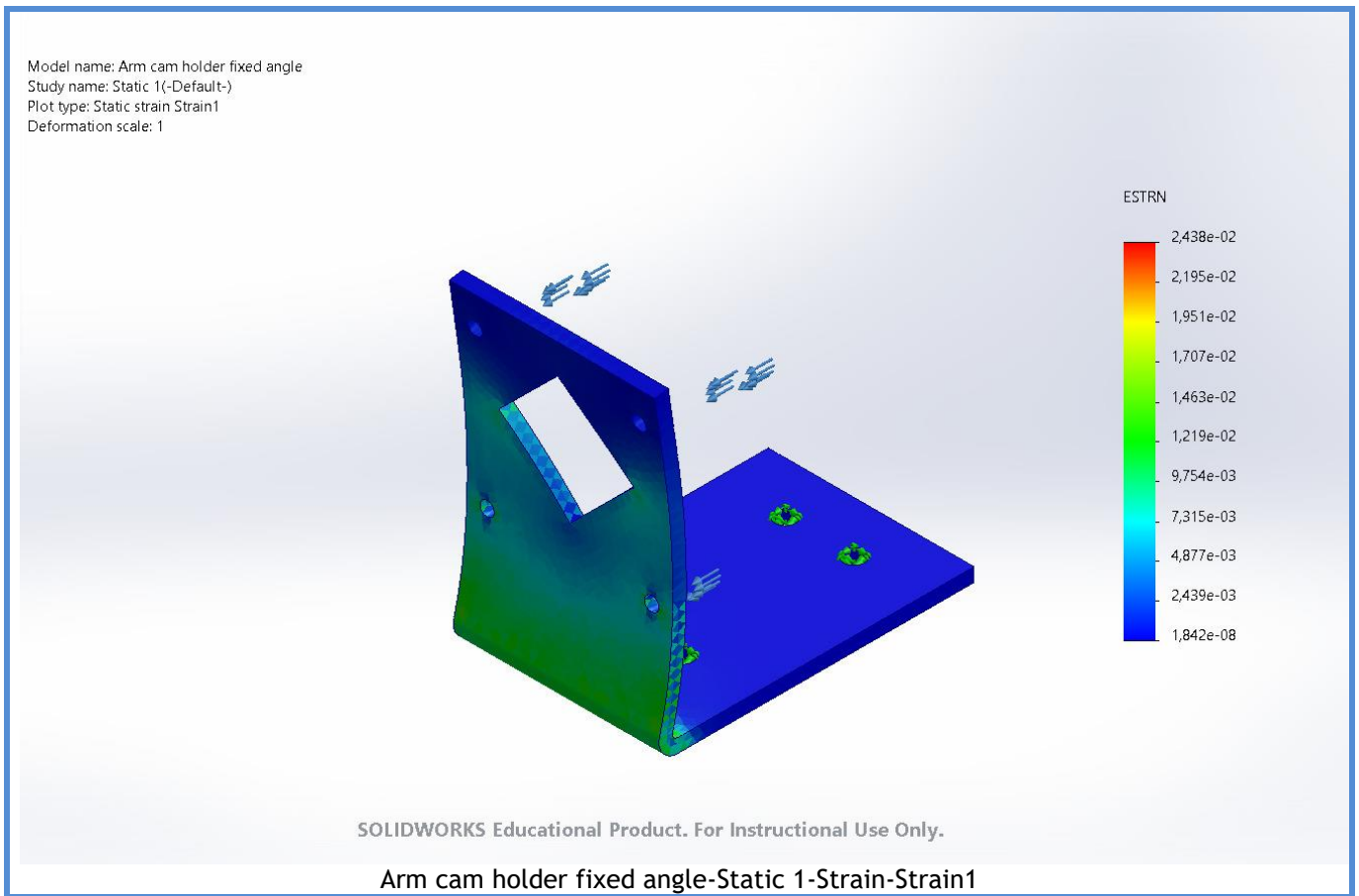
Model name: Arm cam holder fixed angle
Study name: Static 1(-Default-)
Plot type: Static displacement Displacement1
Deformation scale: 1



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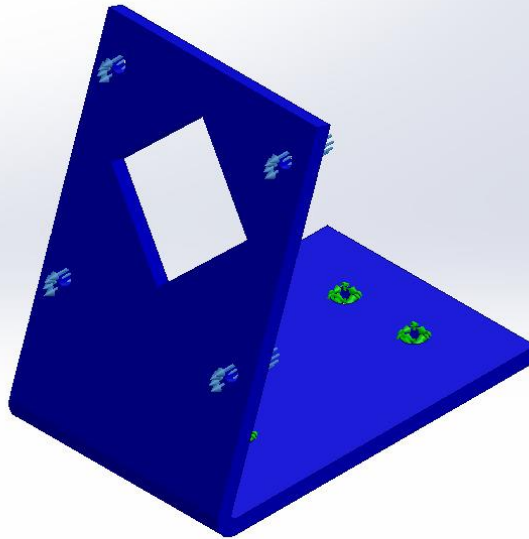
Arm cam holder fixed angle-Static 1-Displacement-Displacement1

Name	Type	Min	Max
Strain1	ESTRN: Equivalent Strain	1,842e-08 Element: 12522	2,438e-02 Element: 7657



Name	Type	Min	Max
Factor of Safety1	Max von Mises Stress	1,513e+00 Node: 1991	1,818e+06 Node: 20109

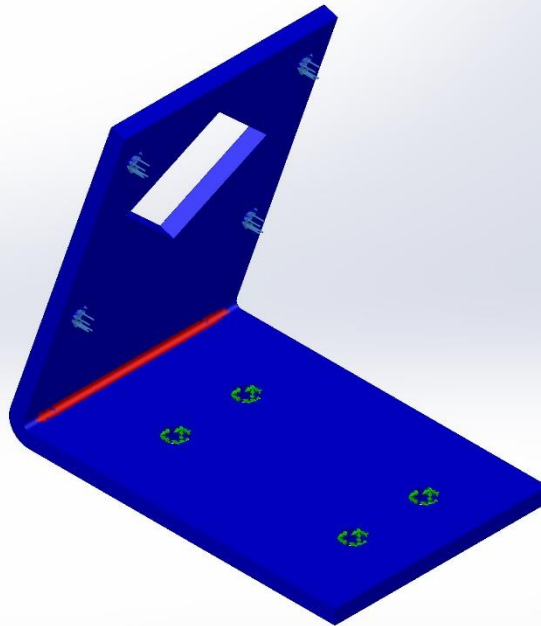
Model name: Arm cam holder fixed angle
Study name: Static 1(-Default-)
Plot type: Factor of Safety Factor of Safety1
Criterion : Max von Mises Stress
Red < FOS = 2 < Blue



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Arm cam holder fixed angle-Static 1-Factor of Safety-Factor of Safety1

Model name: Arm Cam Holder Fixed Angle
Task name: Static (Contact)
Material: Material (Spring Steel or Alloy)
Color: Material (Steel)
Red = FOC = 2 = Blue



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Image-1

Conclusion

Comments:

The bracket will most likely bend and take some of the load off the cable. If the load is greater than assumed, the bracket will most likely break.